

Lecture No. 17

Standard Galerkin Solutions to the Convection-Diffusion Equation

$$\frac{\partial u}{\partial t} + V \frac{\partial u}{\partial x} = \frac{\partial}{\partial x} \left(D \frac{\partial u}{\partial x} \right)$$

- For the standard Galerkin method, the weighting function equals the interpolating functions.
- The standard Galerkin technique is also referred to as the Bubnov-Galerkin method.
- Typically, we use the FEM only to resolve the differential spatial dependence.

This leads to:

$$\underline{M}u_t + (\underline{A} + \underline{B})u = \underline{P}$$

- Now, we apply the FD method to resolve the remaining differential time dependence. Using a weighted implicit/explicit scheme we have:

$$\{\underline{M} + \Delta t\theta(\underline{A}_{j+1} + \underline{B}_{j+1})\}\underline{u}_{j+1} = \underline{M}u_j + (\theta - 1)\Delta t(\underline{A}_j + \underline{B}_j)\underline{u}_j + \Delta t\theta\underline{P}_{j+1} + \Delta t(1 - \theta)\underline{P}_j$$

Linear Finite Elements

Applying linear elements we will obtain the following general nodal equation:

$$\begin{aligned} & \frac{1}{6} \left\{ \frac{u_{i+1,j+1} - u_{i+1,j}}{\Delta t} + 4 \frac{u_{i,j+1} - u_{i,j}}{\Delta t} + \frac{u_{i-1,j+1} - u_{i-1,j}}{\Delta t} \right\} \\ & \frac{V}{2\Delta x} \left\{ \theta [u_{i+1,j+1} - u_{i-1,j+1}] + (1 - \theta) [u_{i+1,j} - u_{i-1,j}] \right\} \\ & - \frac{D}{\Delta x^2} \left\{ \theta [u_{i+1,j+1} - 2u_{i,j+1} + u_{i-1,j+1}] + (1 - \theta) [u_{i+1,j} - 2u_{i,j} + u_{i-1,j}] \right\} = 0 \end{aligned}$$

Lumping, we have the following nodal equation:

$$\begin{aligned} & \frac{u_{i,j+1} - u_{i,j}}{\Delta t} + \frac{V}{2\Delta x} \left\{ \theta [u_{i+1,j+1} - u_{i-1,j+1}] + (1 - \theta) [u_{i+1,j} - u_{i-1,j}] \right\} \\ & - \frac{D}{\Delta x^2} \left\{ \theta [u_{i+1,j+1} - 2u_{i,j+1} + u_{i-1,j+1}] + (1 - \theta) [u_{i+1,j} - 2u_{i,j} + u_{i-1,j}] \right\} = 0 \end{aligned}$$

This equation for the lumped linear FE formulation is identical to that obtained for the central difference FD formulation.

Comparisons between consistent finite element (FE) and lumped finite elements FE-L/FD solutions

A. Fully explicit $\theta = 0.0$

1. Pure Convection $\mathbf{P}_e = \infty$

i. Fig. L29.1a 1/2: Amplitude Portraits

- Shows that there is no damping for either FE-L/FD or FE solutions for any $\mathbf{C}_\#$ values.
- Thus both methods are unconditionally unstable at $\mathbf{P}_e = \infty$.

Recall that stable solutions require $|\xi'_n| \leq 1$.

- FE seems to be more severely unstable (i.e. it will grow unstable more quickly).

ii. Fig. L29.1b 1/2: Ratio Portraits

- Shows that $|\xi'_n|/|\xi_n| > 1 \quad \forall \mathbf{C}_\#$ values. Thus the numerical solution is under-damped as compared to the analytical solution.
- Recall that a perfect solution has a ratio = 1 and an overdamped solution has ratio $\ll 1$.

iii. Fig. L29.1b 1/2: Phase Portraits

- Shows that FE has superior phase propagation characteristics as compared to FE-L/FD. Therefore the numerical dispersion properties are better and the wiggles are less severe.
- Recall perfect solution has phase lag = 0.

iv. Fig. L29.2a 1/2: Computed Solution at $\mathbf{C}_{\#} = 0.1, \mathbf{P}_e \infty$

- Both solutions show statically unstable behavior.
- Instability grows more quickly in the FE solution than FE-L/FD solution
- These features are predicted by the amplitude portrait.

v. Fig. L29.2b 1/2: Computed Solution at $\mathbf{C}_{\#} = 1.1, \mathbf{P}_e \infty$

- Dynamically unstable behavior for both solutions
- Predicted by amplitude portrait

Note: Table L29.1: defines error measures given on all computed results

2. Convection/Diffusion: $P_e = 2$

i. Fig. L29.3a 1/2: Amplitude Portrait

- Indicates that FE-L/FD is stable for $C_{\#} \leq 1.0$ at $P_e = 2$
- FE solution seems to be unstable for $C_{\#} \geq 0.5$ (or lower ?) at $P_e = 2$. It is stable for $C_{\#} = 0.1$
- For stable regions the solutions will be damped

ii. Fig. L29.3b 1/2: Ratio Portrait

- Solutions are for the most part underdamped
- FE-L/FD and FE are very similar except at $C_{\#} = 0.1$, the FE-L/FD underdamps whereas FE overdamps but only at high wave numbers.

iii. Fig. L29.3c 1/2: Phase Portrait

- Indicates that FE-L/FD has excellent phase propagation characteristics except at $C_{\#} = 0.1$.

- FE also has good phase propagation characteristics except at $C_{\#} = 0.5$ where there is a phase lead.
- iv. Fig. L29.4a 1/2: Computed Solution at $P_e = 2$ and $C_{\#} = 0.1$
- Both solution are good, FE is slightly superior over FE-L/FD.
 - Slight overall phase lag for FE-L/FD and slight phase lead for FE as indicated by the phase portraits.
 - Underdamping is not a problem as is indicated by Ratio portraits
 - Any wiggles introduced will be damped out due to amplitude damping as indicated by amplitude portrait (physical damping present).
- v. Fig. L29.4b 1/2: Computed Solution at $P_e = 2$ and $C_{\#} = 1.1$
- Dynamically unstable
 - FE much more so than FE-L/FD. This was indicated by amplitude portraits.
3. General Conclusions FE-L/FD and FE explicit

- Stability is a severe problem for both FE-L/FD and FE schemes when an explicit time discretization is used. Always unstable for convection dominant cases.
- FE has more restrictive ability characteristics than FE-L/FD. Also unstable behavior for FE is more severe (but this really doesn't matter).
- Diffusion dominant low $C_{\#}$ problems handled reasonably well (less restrictive $C_{\#}$ conditions for FE-L/FD than FE).

B. Crank-Nicolson $\theta = 0.5$

1. Pure Convection: $P_e = \infty$

i. Fig. L29.5a 1/2 and L29.5b 1/2: Amplitude Ratio Portraits

- Shows no damping and perfect analytical damping for all $C_{\#}$ values for both FE-L/FD and FE
- Thus no stability constraints and no under or over damping.
- Thus amplitude and ratio values are exact or perfect.

ii. Fig. L29.5c 1/2: Phase Portraits

- Shows that FE has much better phase propagation characteristics than FE-L/FD. This is especially true of lower $C_{\#}$ values ($C_{\#} = 0.1, 0.5$) although it is in general the case.
- iii. Fig. L29.6a 1/2: Computed Solution at $P_e = \infty$ and $C_{\#} = 0.1$
- FE solution has fewer and much smaller wiggles
 - FE solution has a peak which is well propagated while FE-L/FD solution has severe peak phase lag.
 - These differences result from the better phase behavior of the FE scheme.
- iv. Fig. L29.6b 1/2: Computed Solution at $P_e = \infty$ and $C_{\#} = 1.1$
- FE solution is again better.
- However difference is not as large as at $C_{\#} = 0.1$ since although phase behavior of FE is better it is not drastically better than FE-L/FD at high $C_{\#}$ values.

2. Convection/Diffusion: $P_e = 2$

- i. Fig. L29.7a 1/2: Amplitude Portrait

- All cases damp for all wavelengths
- Thus stable and will help damp wiggles

ii. Fig. L29.7b 1/2: Ratio Portrait

- For FE-L/FD, all cases damp less than analytical solution. Thus there is no over-damping whatsoever.
- For FE low $C_{\#}$ values have some overdamping but only at low wavelengths. High $C_{\#}$ values behave like the FE-L/FD solution.

iii. Fig. L29.7c 1/2:

- FE has much superior phase properties compared to FE-L/FD.
- FE has slight phase lead behavior at higher $C_{\#}$ values but only at very low wavelength ratios.

iv. Fig. L29.8

- FE is superior to FE-L/FD.
- FE-L/FD exhibits slight overall phase lag due to poorer phase behavior.

- Because of longer wavelengths involved in this problem, no substantial phase lag problems.
 - Also any wiggles initially generated due to phase errors will be damped out due to physical damping present.
- v. Fig. L29.8b 1/2: Computed Solutions at $\mathbf{P}_e = 2$ and $\mathbf{C}_\# = 1.1$
- FE again superior to FE-L/FD.
 - Both solutions have deteriorated as compared to low $\mathbf{C}_\#$ values, but only slightly so.

3. General Conclusions FE-L/FD and Fe with Crank-Nicolson

- Unconditionally stable for both
- Accuracy is much superior than FE than for FE-L/FD mainly due to much better phase behavior.

In highly physically damped cases this leads to better overall phase of the distribution (with no wiggles present in either solution).

C. Fully Implicit $\theta = 1.0$

1. Pure Convection: $\mathbf{P}_e = \infty$

i. Fig. L29.9a 1/2:

- All wavelengths are damped for $\mathbf{P}_e = \infty$ and $\mathbf{C}_\# = 0.1 \rightarrow 1.1$.
- Thus we can expect stable solutions which are damped for both FE-L/FD and FE.

ii. Fig. L29.9b 1/2: Ratio Portrait

- Numerical solutions are overdamped as compared to analytical solutions.
- For $\mathbf{C}_\# = 0.1$ this overdamping is not severe and is slightly greater at small wavelengths in FE than for FE-L/FD.
- For higher $\mathbf{C}_\#$ values ($0.5 \rightarrow 1.1$), overdamping is severe for the entire wavelength range and is similar for both FE-L/FD and FE.

iii. Fig. L29.9c 1/2: Phase Portrait

- Shows that FE phase behavior is much better than FE-L/FD. This is especially true for lower $\mathbf{C}_\#$ values ($0.1 \rightarrow 0.5$).

iv. Fig. L29.10a 1/2: Computed Solution at $\mathbf{P}_e = \infty$ and $\mathbf{C}_\# = 0.1$

- FE-L/FD solution is very poor exhibiting wiggles which are not sufficiently damped.
- Overall phase is also very poor.
- These wiggles are due to FE-L/FD having poor phase behavior in important wavelength range while there is not sufficient damping to get rid of them.
- FE solution has no wiggles and overall phase is excellent.
- However the solution is overdamped. This is due to the fact that the phase propagation characteristics are very good and we have sufficient damping for the very short wavelengths. However we've lost parts of our solution through this combined dispersion and damping process and this accounts for the overdamped solution.

v. Fig. L29.10b 1/2: Computed Solution at $\mathbf{P}_e = \infty$ and $\mathbf{C}_\# = 1.1$

- Both FE-L/FD and Fe solutions are severely overdamped. This feature is predicted by ratio portraits.

2. Convection/Diffusion $\mathbf{P}_e = 2$

i. Fig. L29.11a 1/2: Amplitude Portrait

- Shows that both FE-L/FD and Fe always damp and thus have stable behavior for all $\mathbf{C}_\#$ values considered.

ii. Fig. L29.11b 1/2: Ratio Portraits

- Shows that for all higher $\mathbf{C}_\#$ values (0.5 – 1.1) there is severe overdamping in the high wavelength range.
- At $\mathbf{C}_\# = 0.1$, behavior is much better and only the FE shows damping at very low wavelength values.

iii. Fig. L29.11c 1/2: Phase Portrait

- FE has superior phase behavior, especially at lower $C_{\#}$ values.

iv. Fig. L29.12a 1/2: Computed Solution at $P_e = \infty$ and $C_{\#} = 0.1$

- Both FE-L/FD and FE solutions are reasonable
- FE solution has slightly better overall phase.

v. Fig. L29.12b 1/2: Computed Solution at $P_e = \infty$ and $C_{\#} = 1.1$

- Both FE-L/FD and Fe are severely overdamped. This is predicted by ratio portrait.

3. General Conclusions FE-L/FD and FE fully implicit

- FE is somewhat superior in that it is slightly more accurate and eliminates wiggles better.
- In general, implicit methods for both FE-L/FD and FE often lead to severely overdamped results.

D. General Conclusions – sections A, B, C

- Best time integration scheme is Crank-Nicolson. Explicit methods were generally unstable for convection dominated flows while implicit methods have severe overdamping problems. Recall that C-N is $0(\Delta t)^2$ accurate whereas explicit and implicit methods were only $0(\Delta t)$ accurate.
- FE is much better than FE-L/FD for C-N in terms of overall quality of results. Quality is excellent for low P_e and good for high P_e (some wiggles remained).
- If you're applying consistent FE is it always computationally worthwhile to use C-N since you're backsubstituting into matrices anyway (although matrices may have to be reset).
- Always exercise extreme caution with any numerical solution.

Table L29.1

Error E1: Integral measure of the overall error of the numerical solution.

$$E1 = \frac{1}{m(t)} \left[\int_0^L (\phi^{\text{num}}(x, t) - \phi^{\text{ex}}(x, t))^2 dx \right]^{1/2}$$

Value for Exact Solution = 0.0

Error E2: Discrete measure of the overall error of the numerical solution.

$$E2 = \frac{1}{m(t)} \left[\sum_i (\phi_i^{\text{num}}(t) - \phi^{\text{ex}}(x_i, t))^2 \right]^{1/2}$$

Value for Exact Solution = 0.0

Error E3: Point measure of the artificial damping of the numerical solution (peak depression).

$$E3 = \left| \frac{\phi_{\max}^{\text{ex}}(t) - \phi_{\max}^{\text{num}}(t)}{\phi_{\max}^{\text{ex}}(t)} \right|$$

Value for Exact Solution = 0.0

Error E4: Point measure of the maximum spurious oscillation in the numerical solution.

$$E4 = \left| \frac{\phi_{\max, \text{neg}}^{\text{num}}(t)}{\phi_{\max}^{\text{ex}}(t)} \right|$$

Value for Exact Solution = 0.0

Error E5: Point measure of the phase shift introduced in the numerical solution.

$$E5 = \frac{x_{\max}^{\text{ex}}(t) - x_{\max}^{\text{num}}(t)}{x_{\max}^{\text{ex}}(t)}$$

Value for Exact Solution = 0.0

Error E6: Integral measure of mass preservation.

$$E6 = \frac{1}{m(t)} \int_0^L \phi^{\text{num}}(x, t) dx$$

Value for Exact Solution = 1.0

Fig. L29.1a1

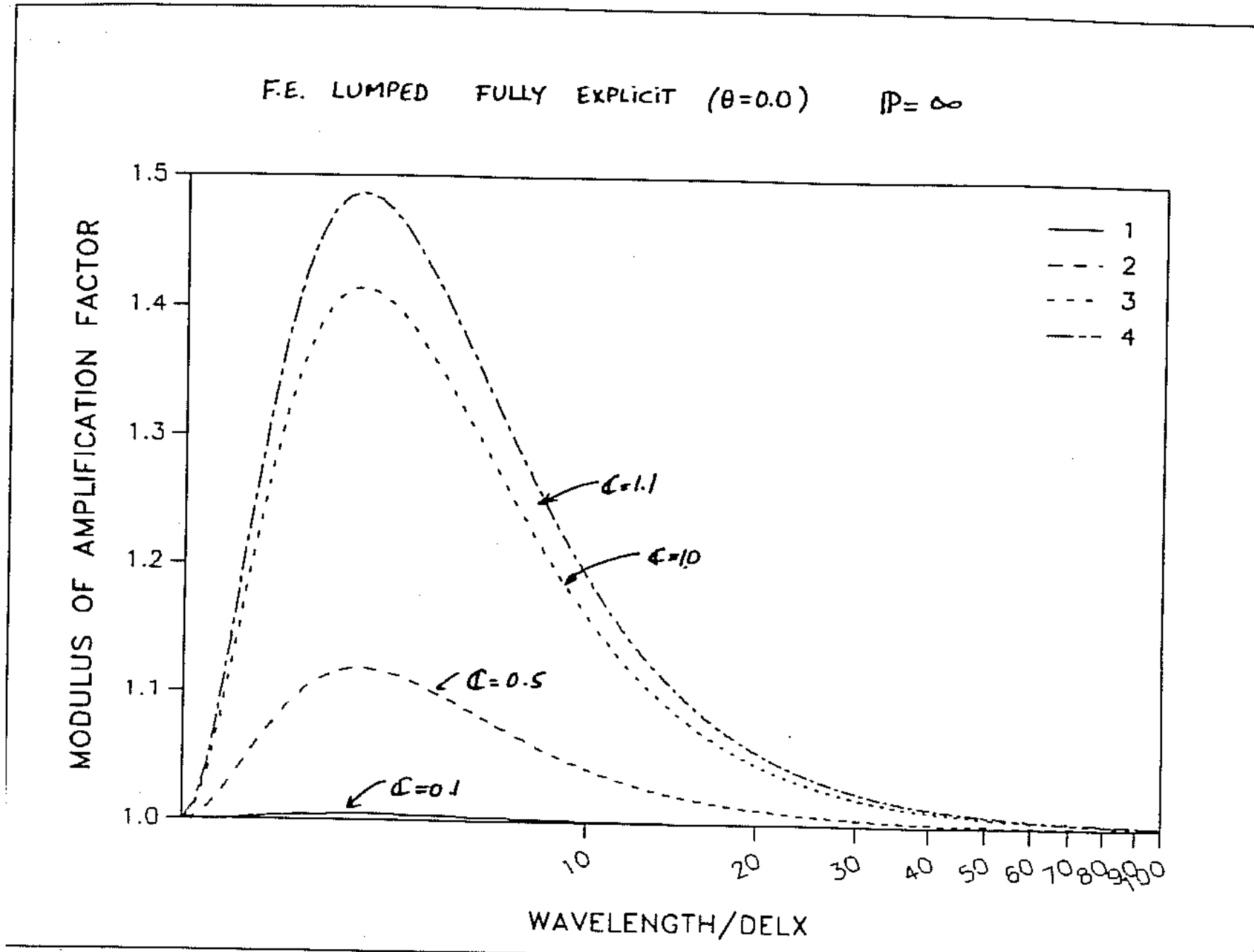


Fig. L29.1a2

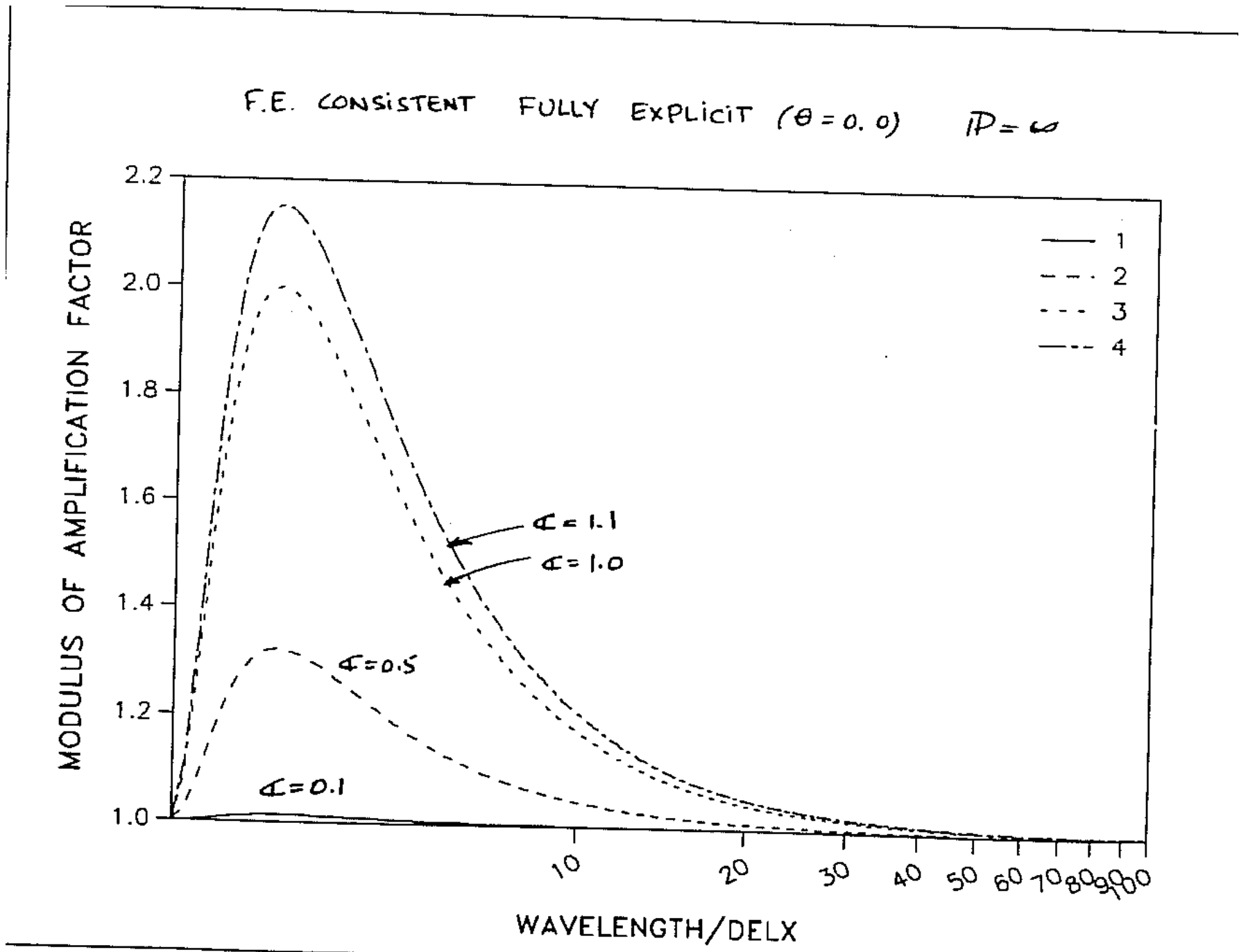


Fig. L29.1b1

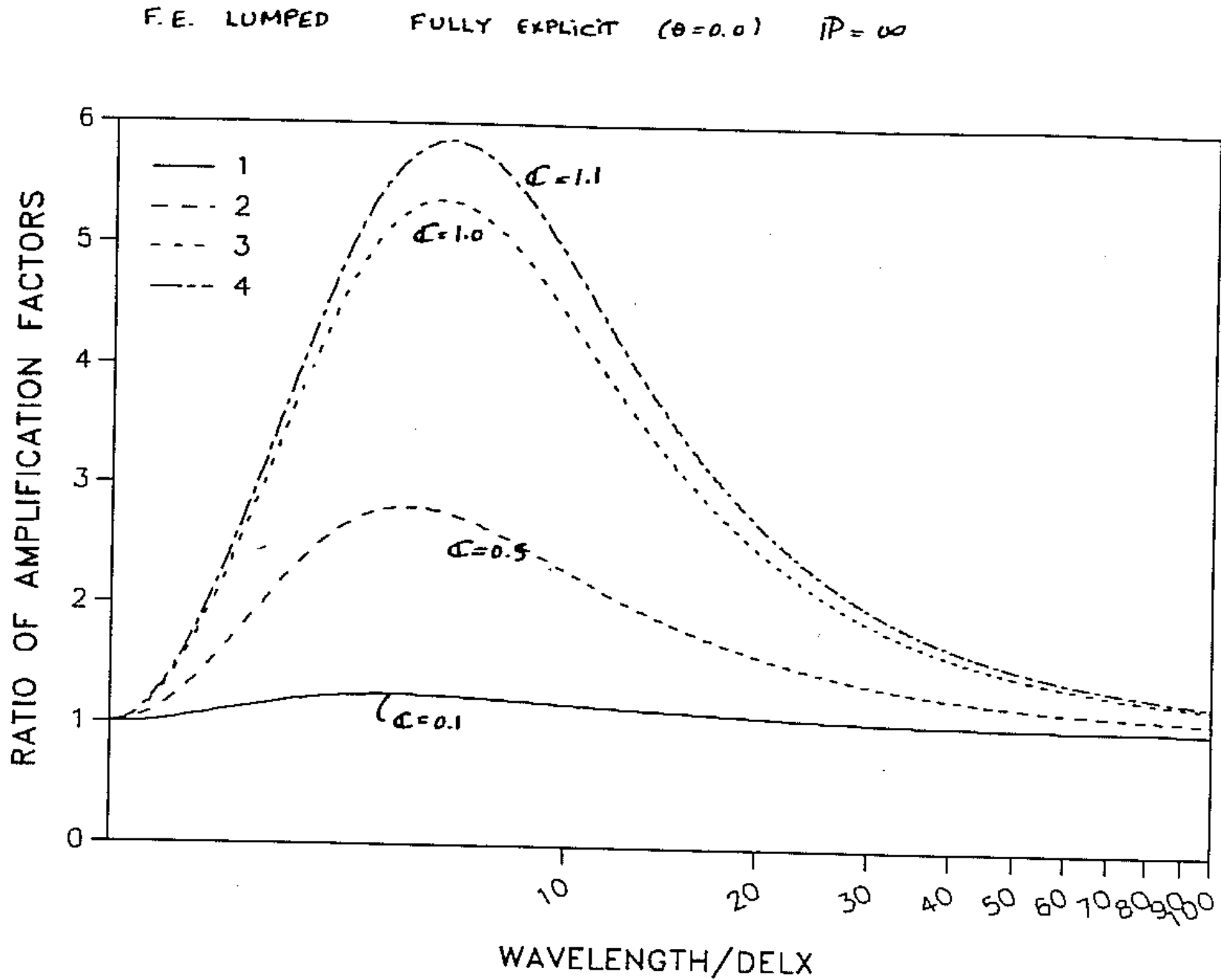


Fig. L29.1b2

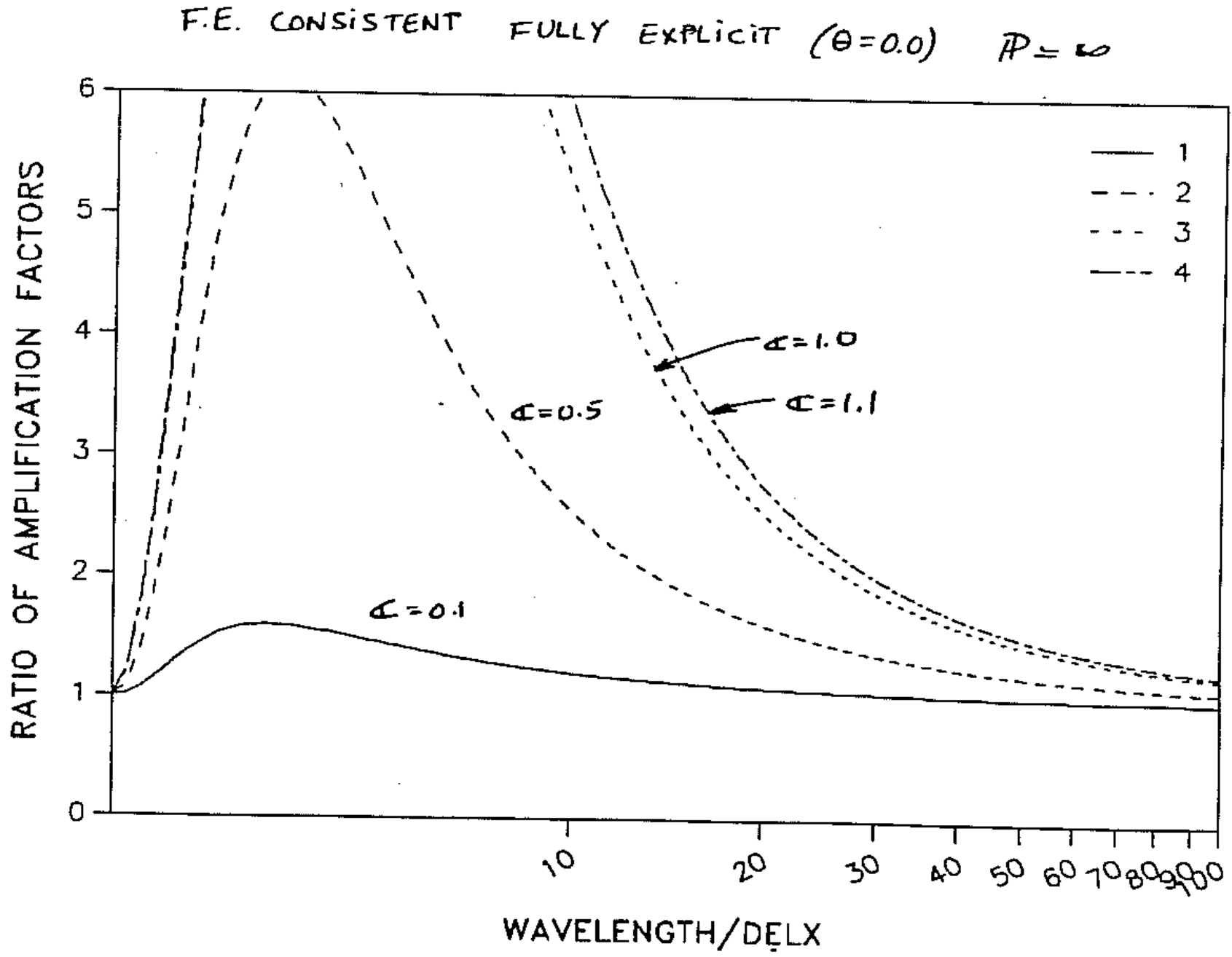


Fig. L29.1c1

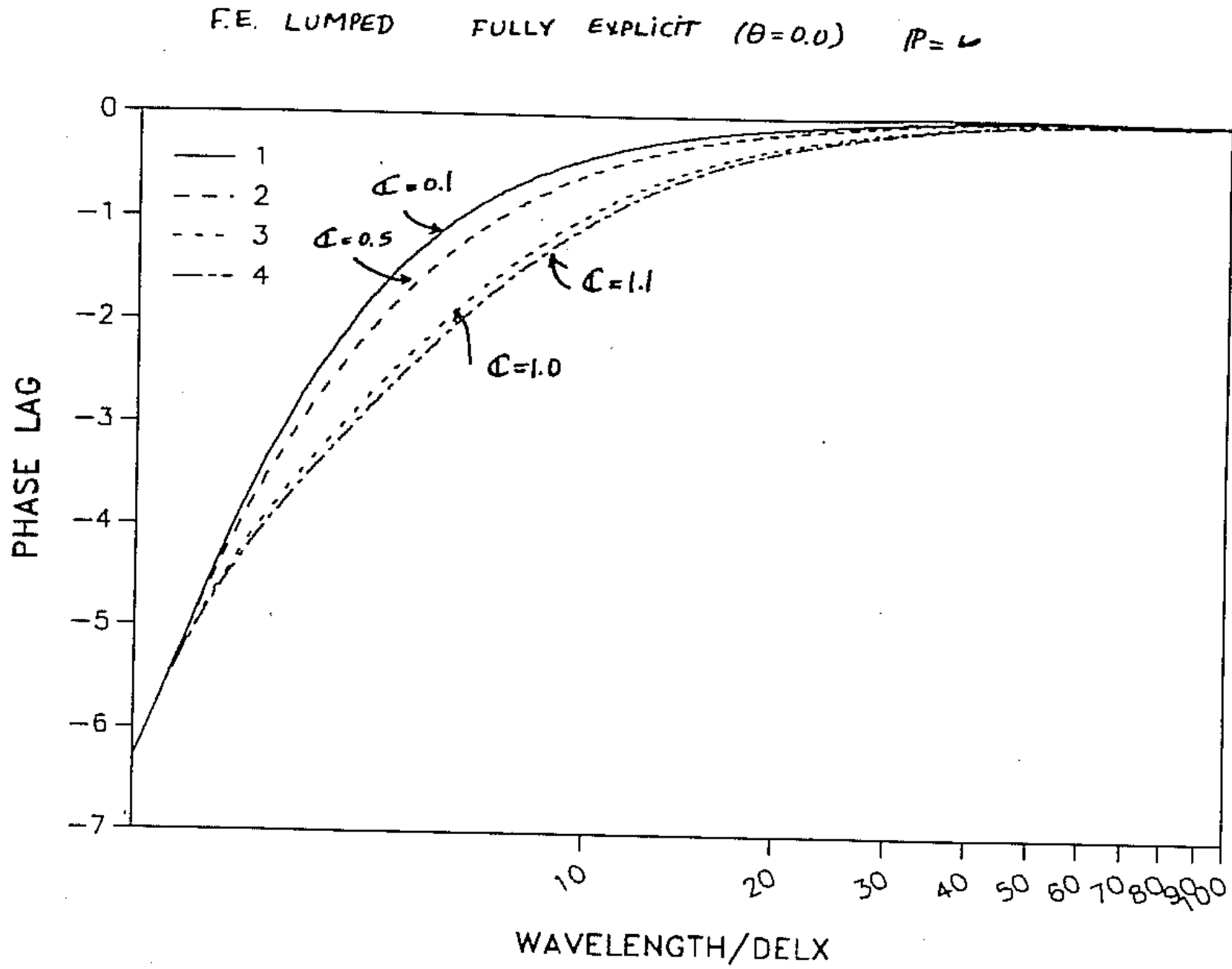


Fig. L29.1c2

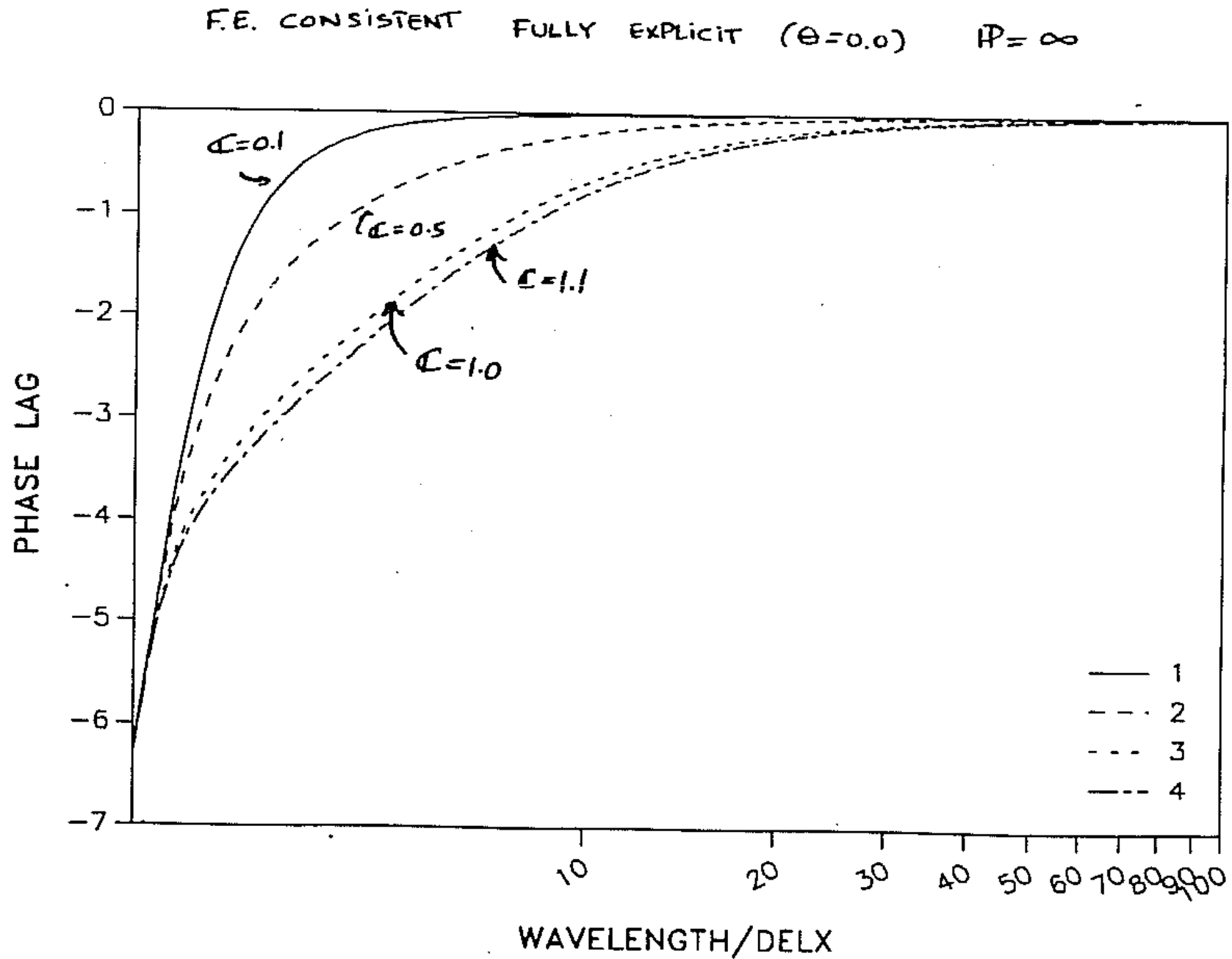


Fig. L29.2a1

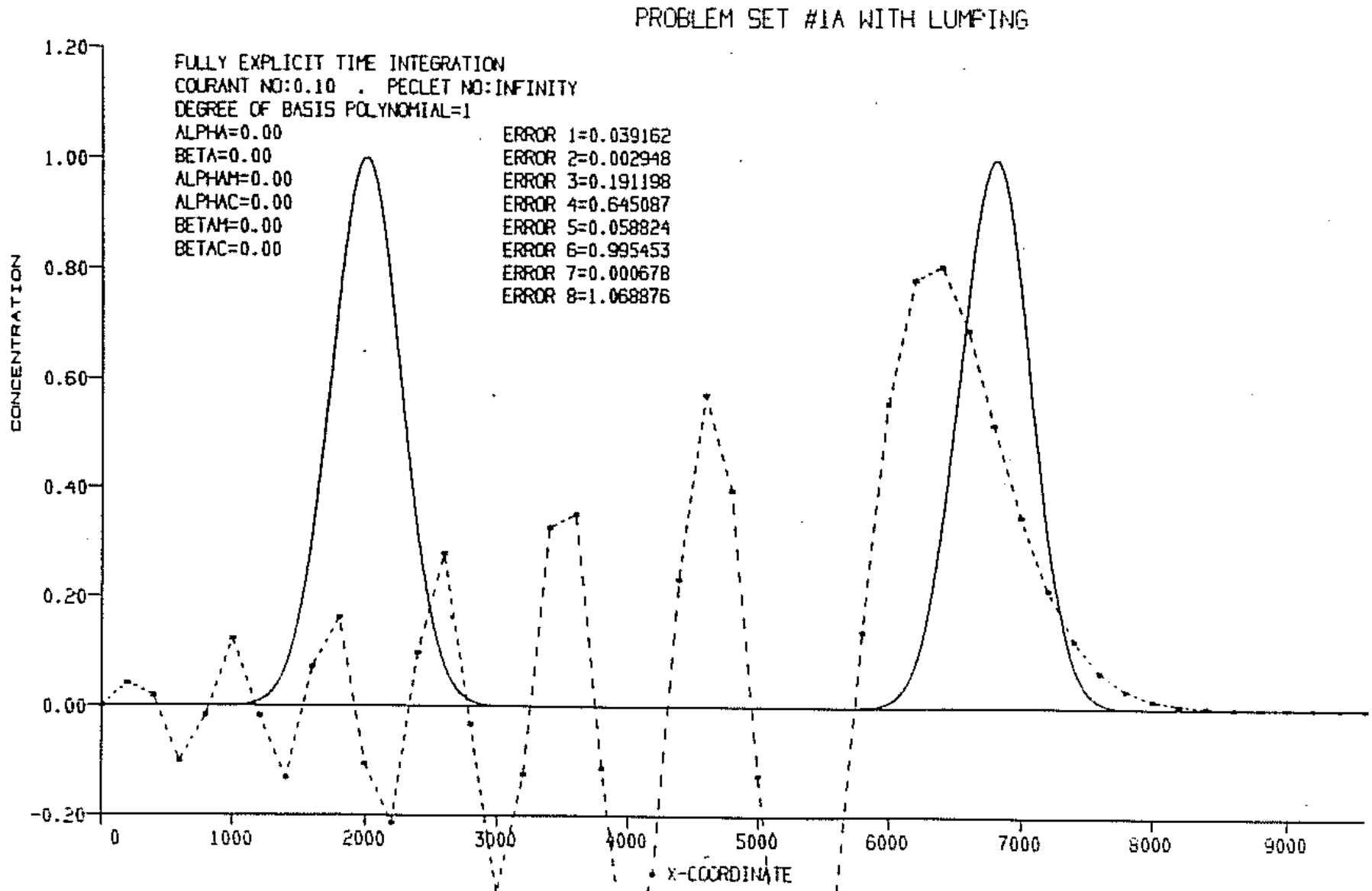


Fig. L29.2a2

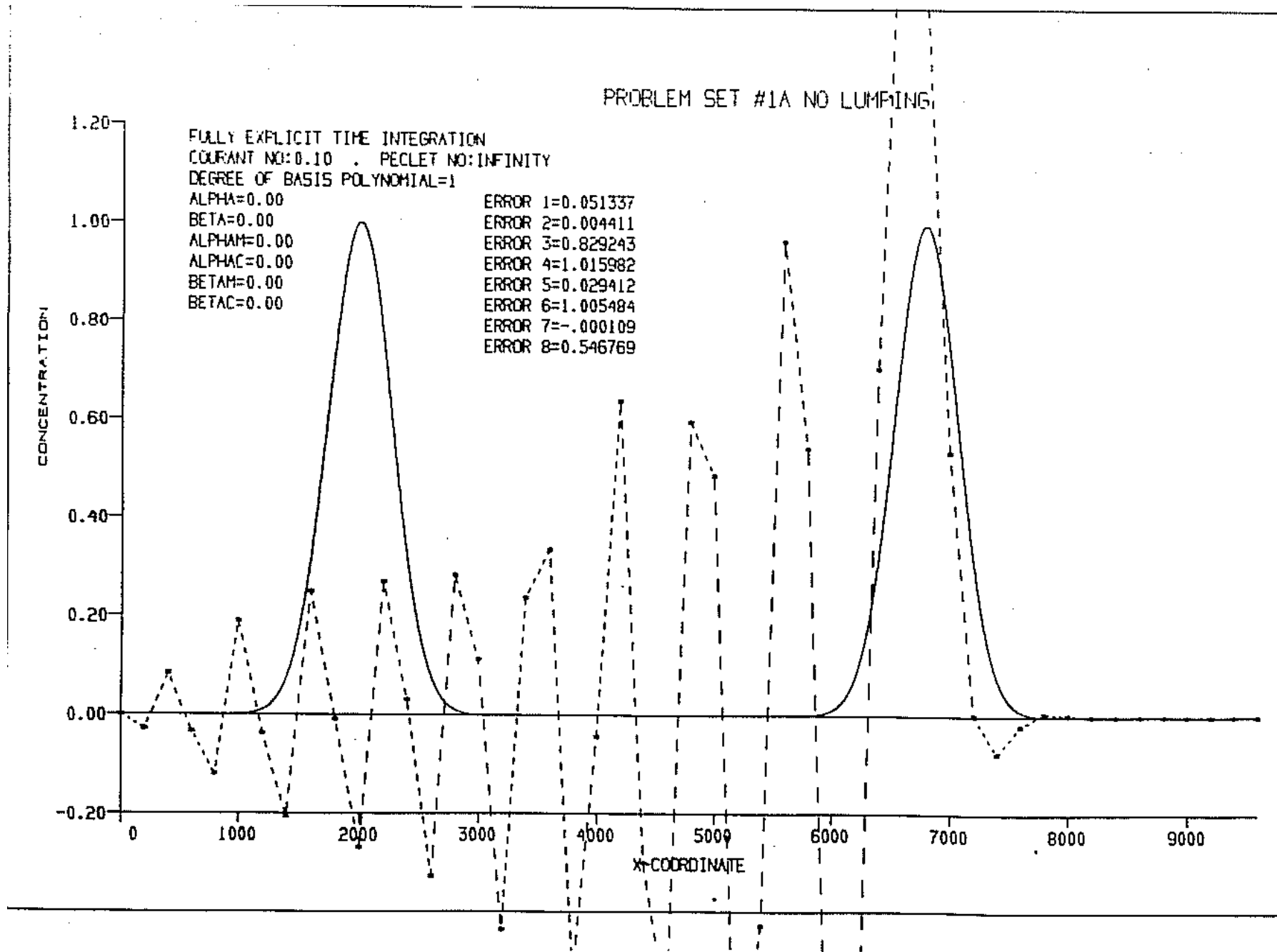


Fig. L29.2b1

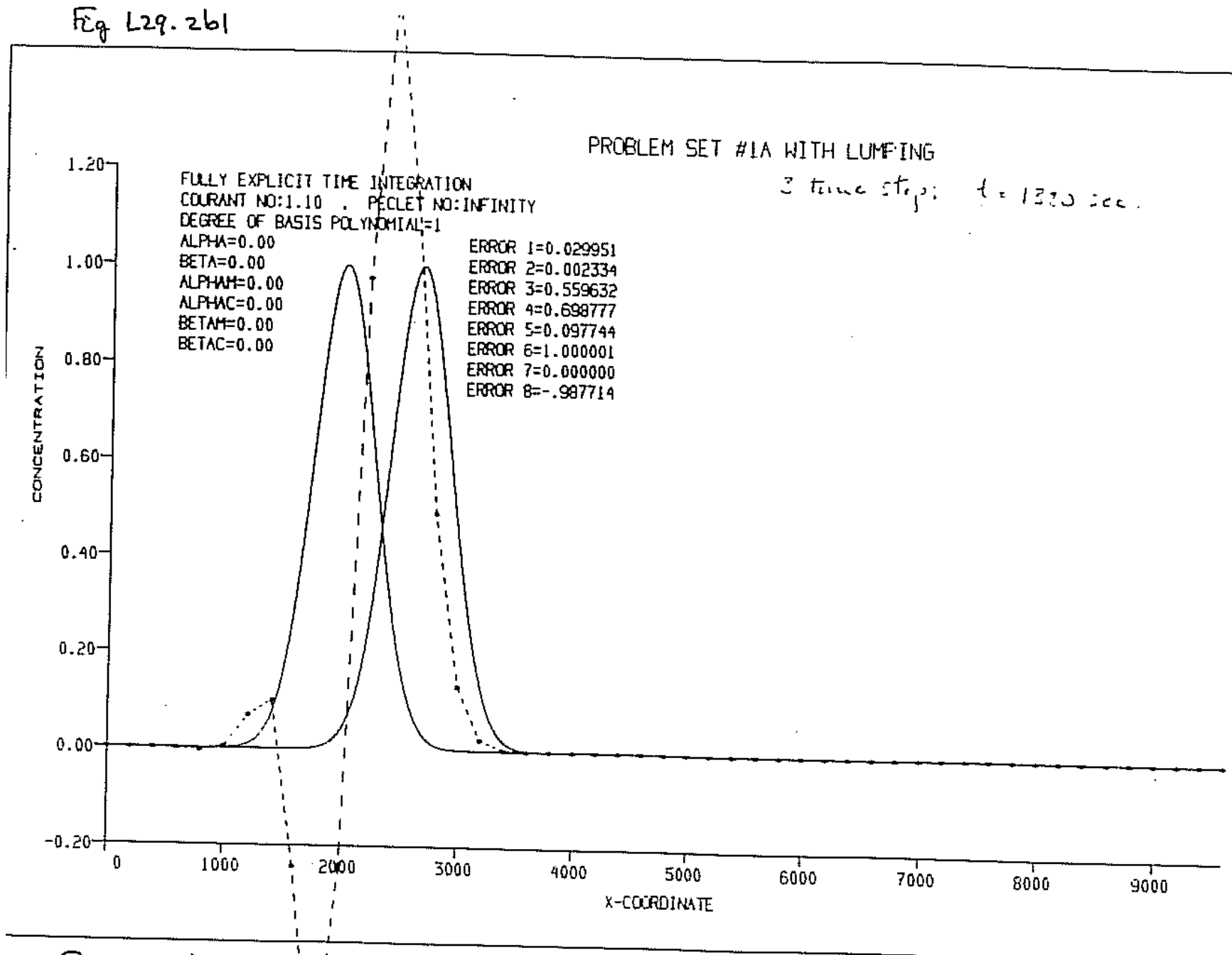


Fig. L29.2b2

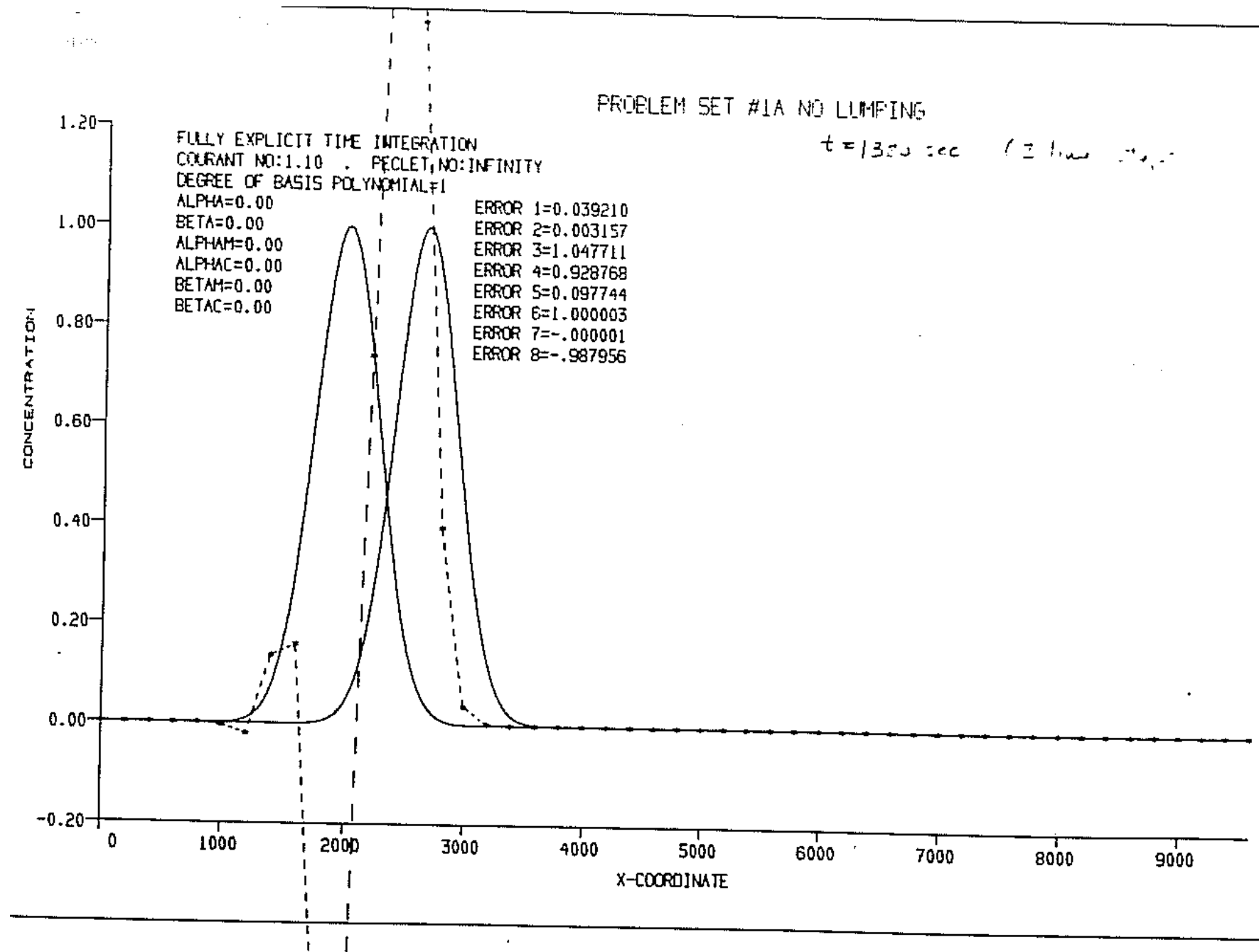


Fig. L29.3a1

F.E. LUMPED FULLY EXPLICIT ($\theta=0.0$) $IP=2$

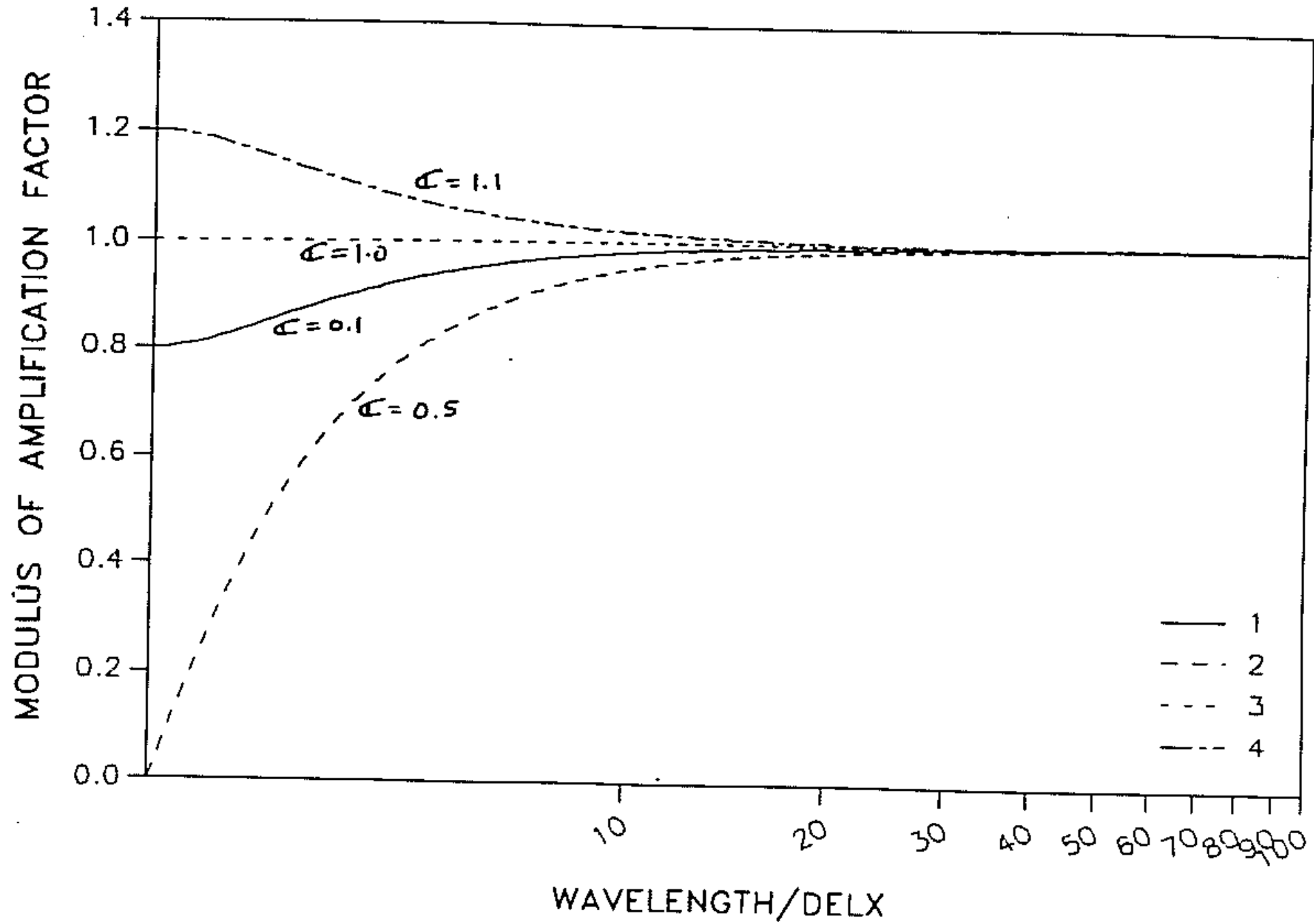


Fig. L29.3a2

F.E. CONSISTENT FULLY EXPLICIT ($\theta=0.0$) $\mu=2$

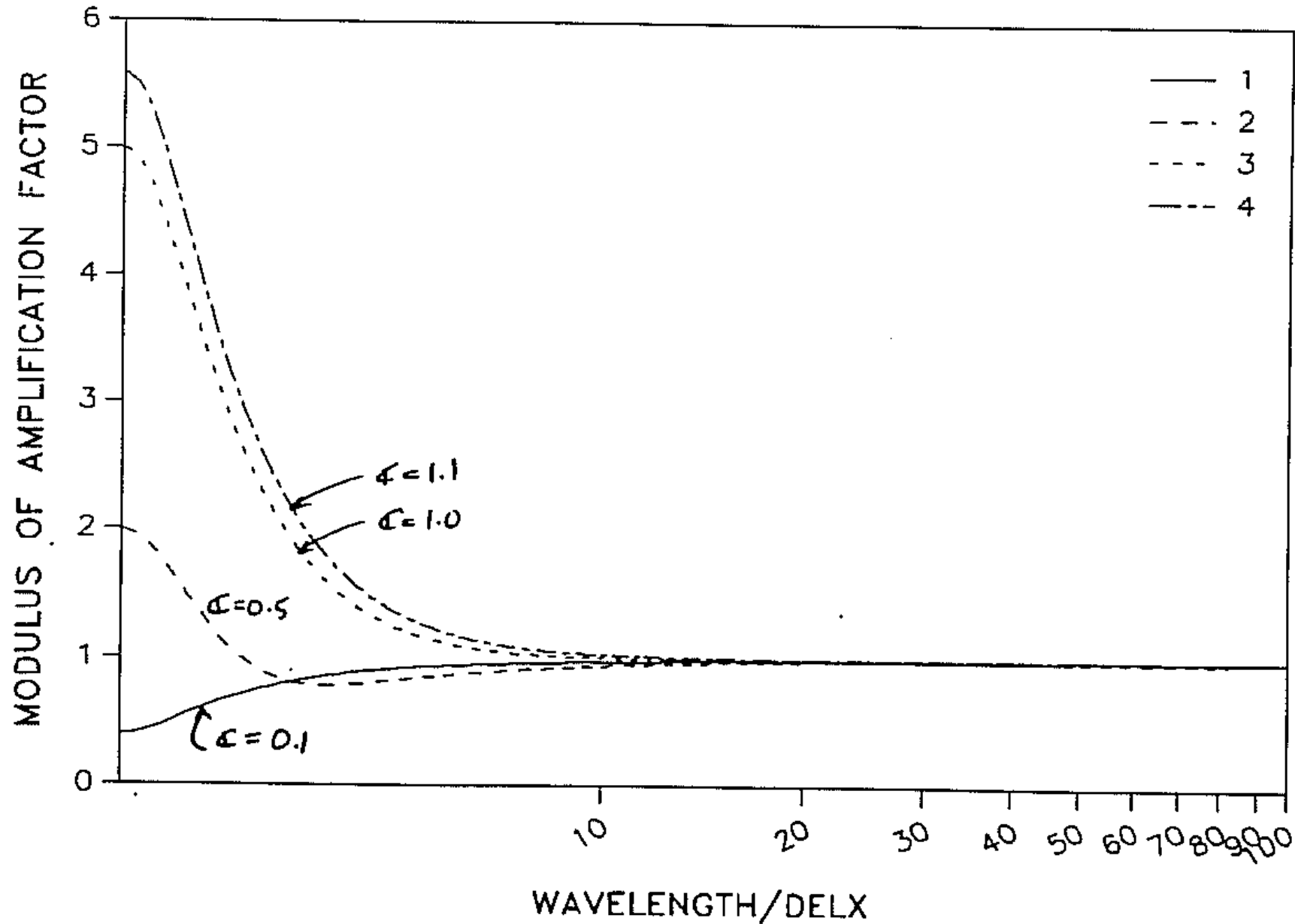


Fig. L29.3b1

F.E. LUMPED FULLY EXPLICIT ($\theta=0.0$) $P=2$

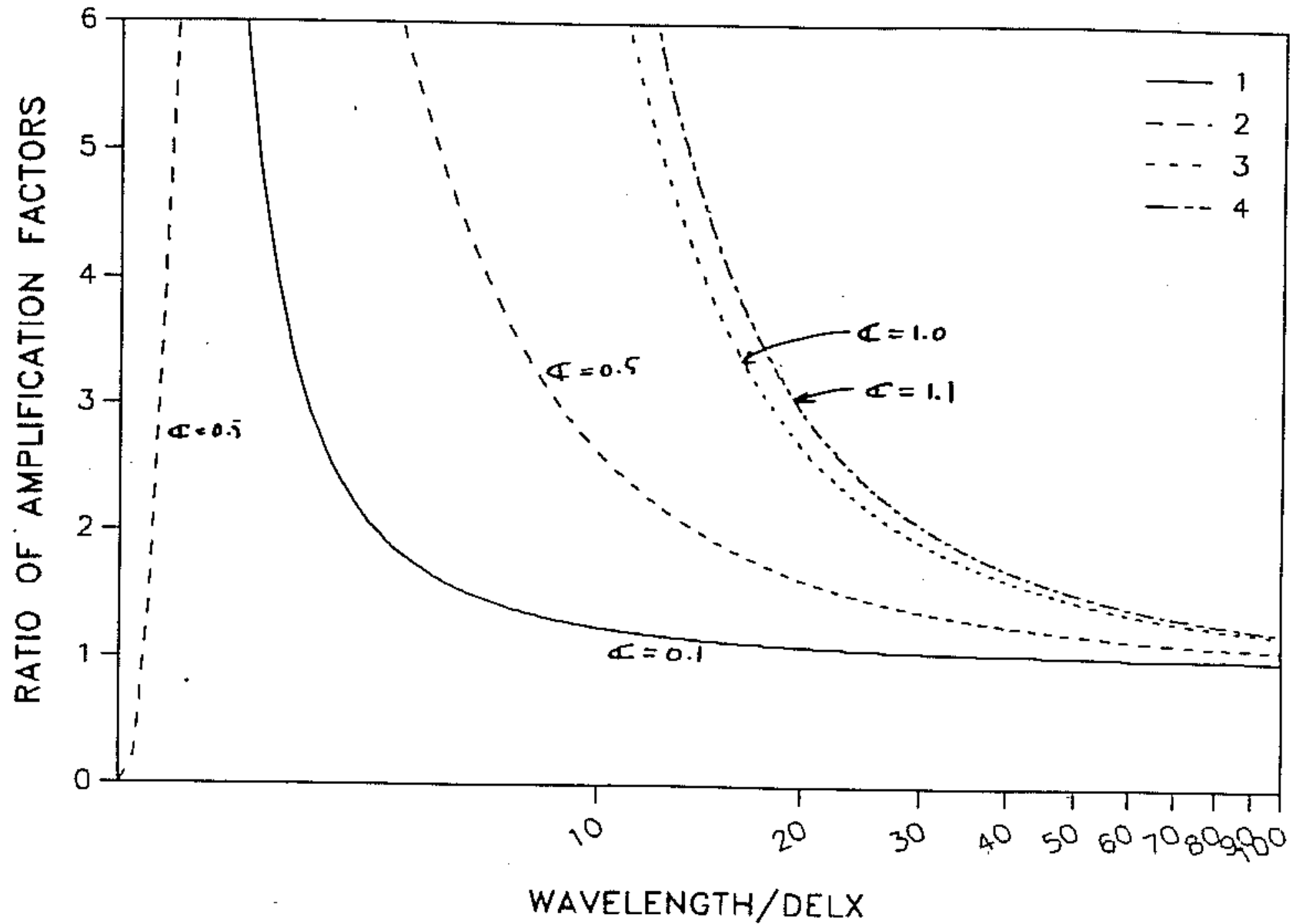


Fig. L29.3b2

F.E. CONSISTENT FULLY EXPLICIT ($\theta=0.0$) $\beta=2$

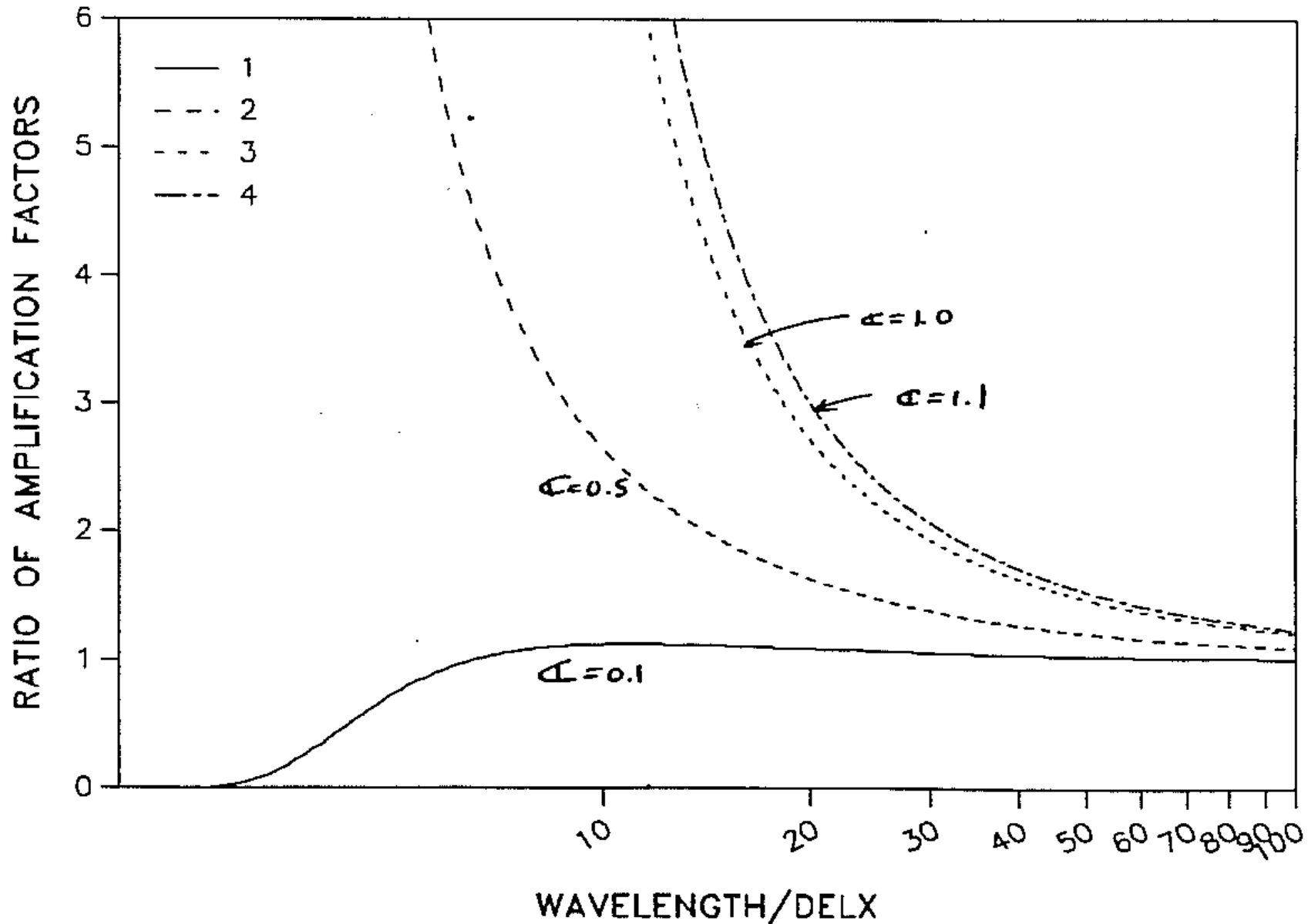


Fig. L29.3c1

F.E. LUMPED FULLY EXPLICIT ($\theta = 0.0$) $P=2$

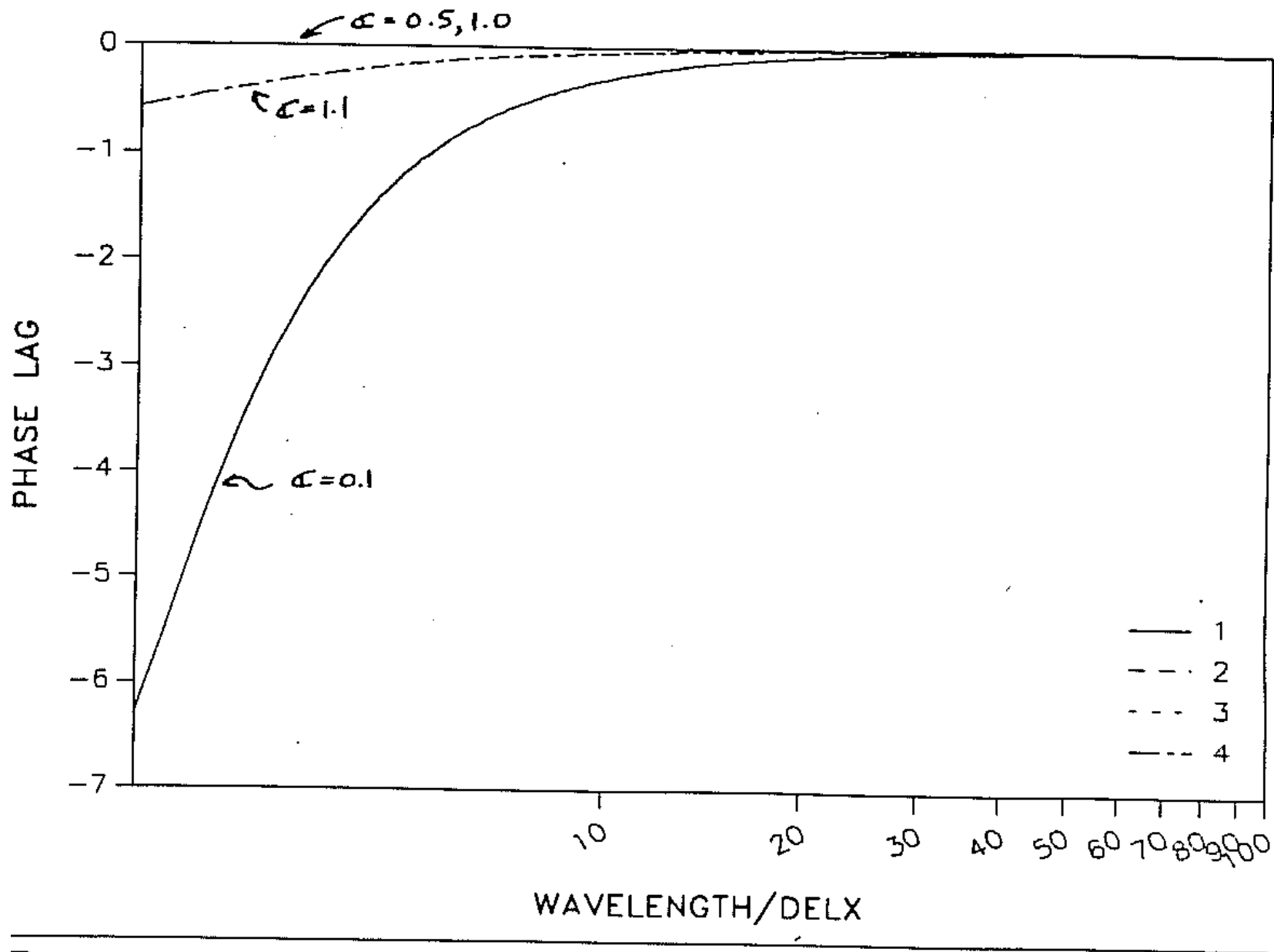


Fig. L29.3c2

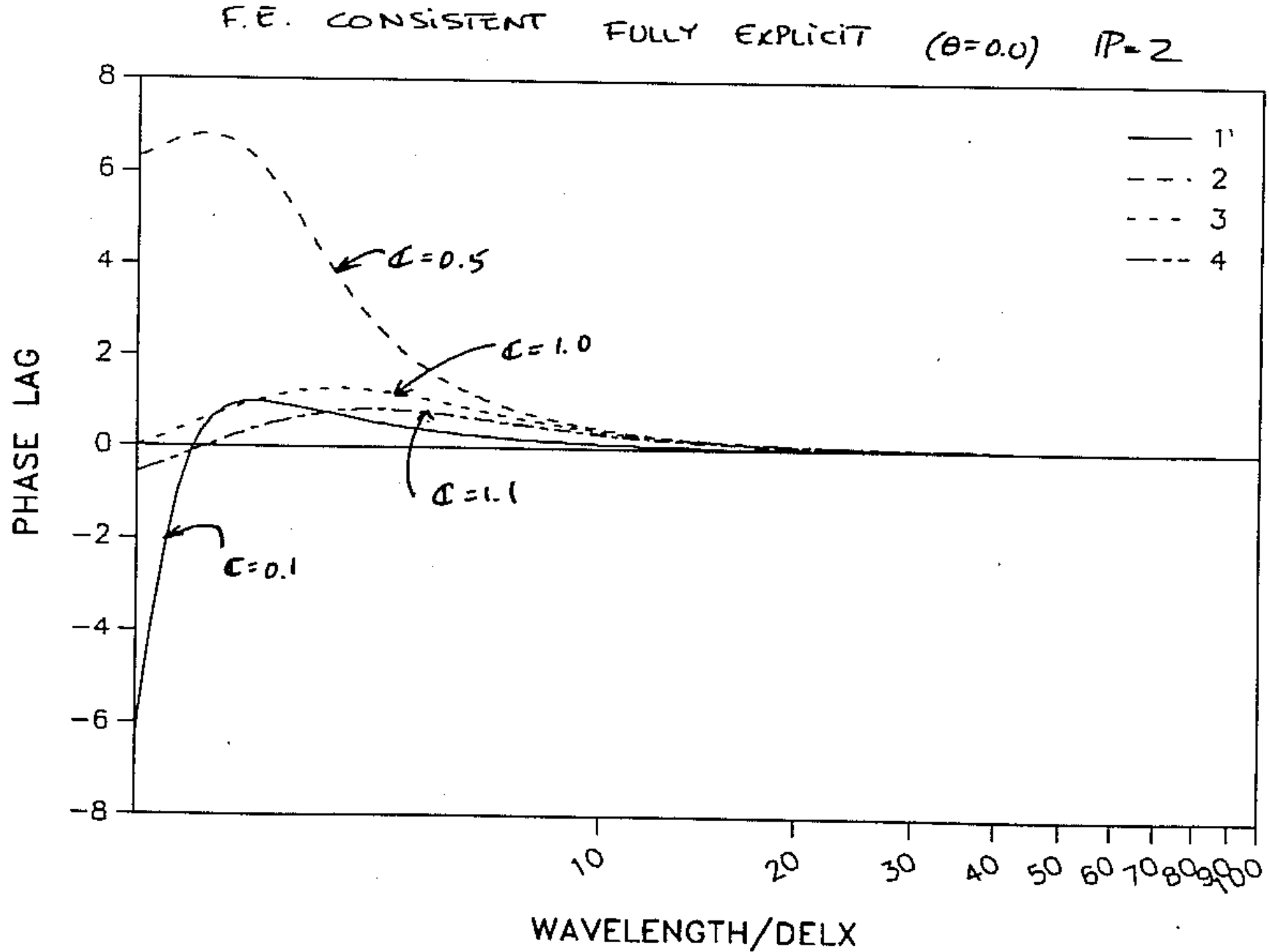


Fig. L29.4a1

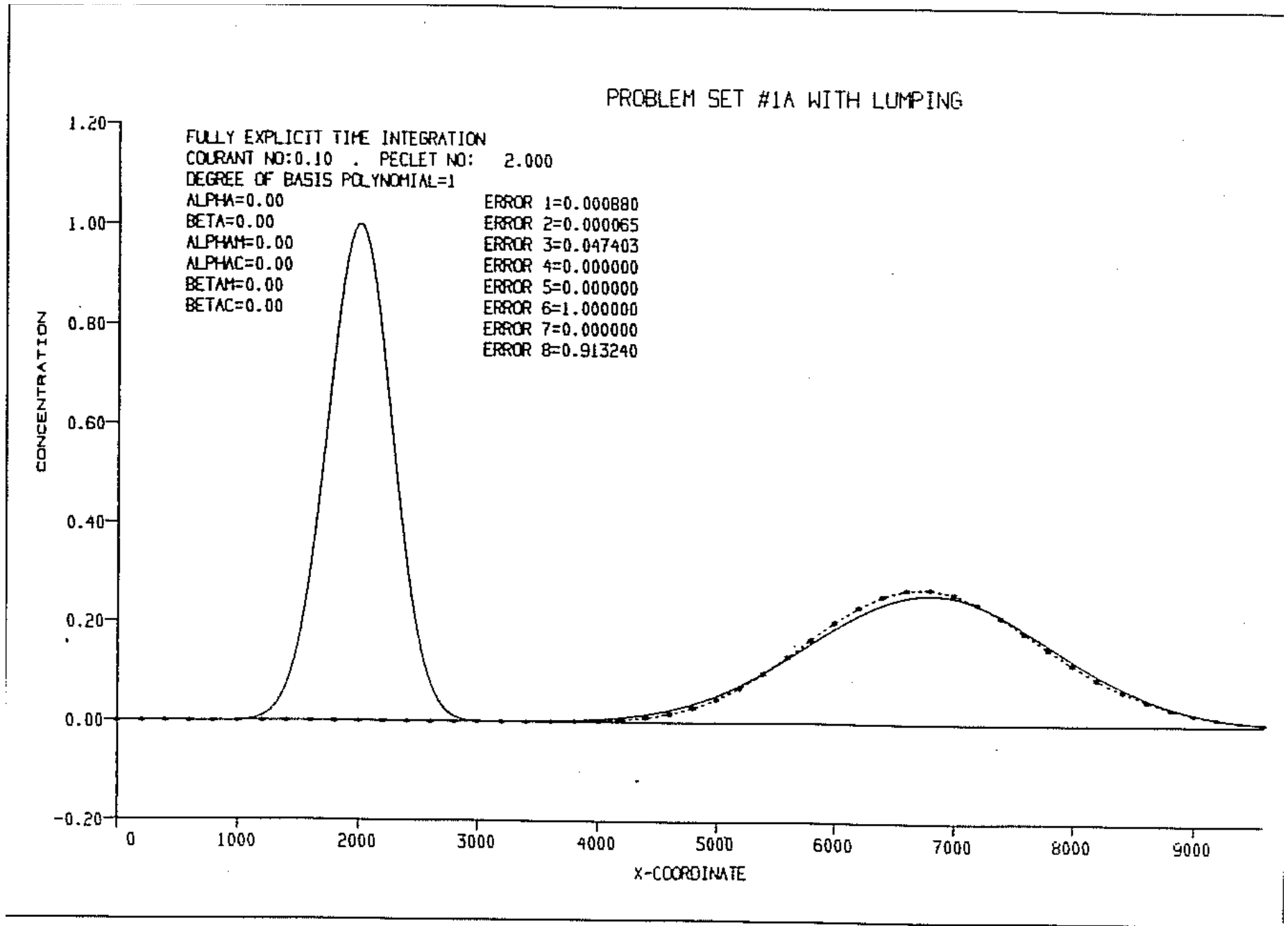


Fig. L29.4a2

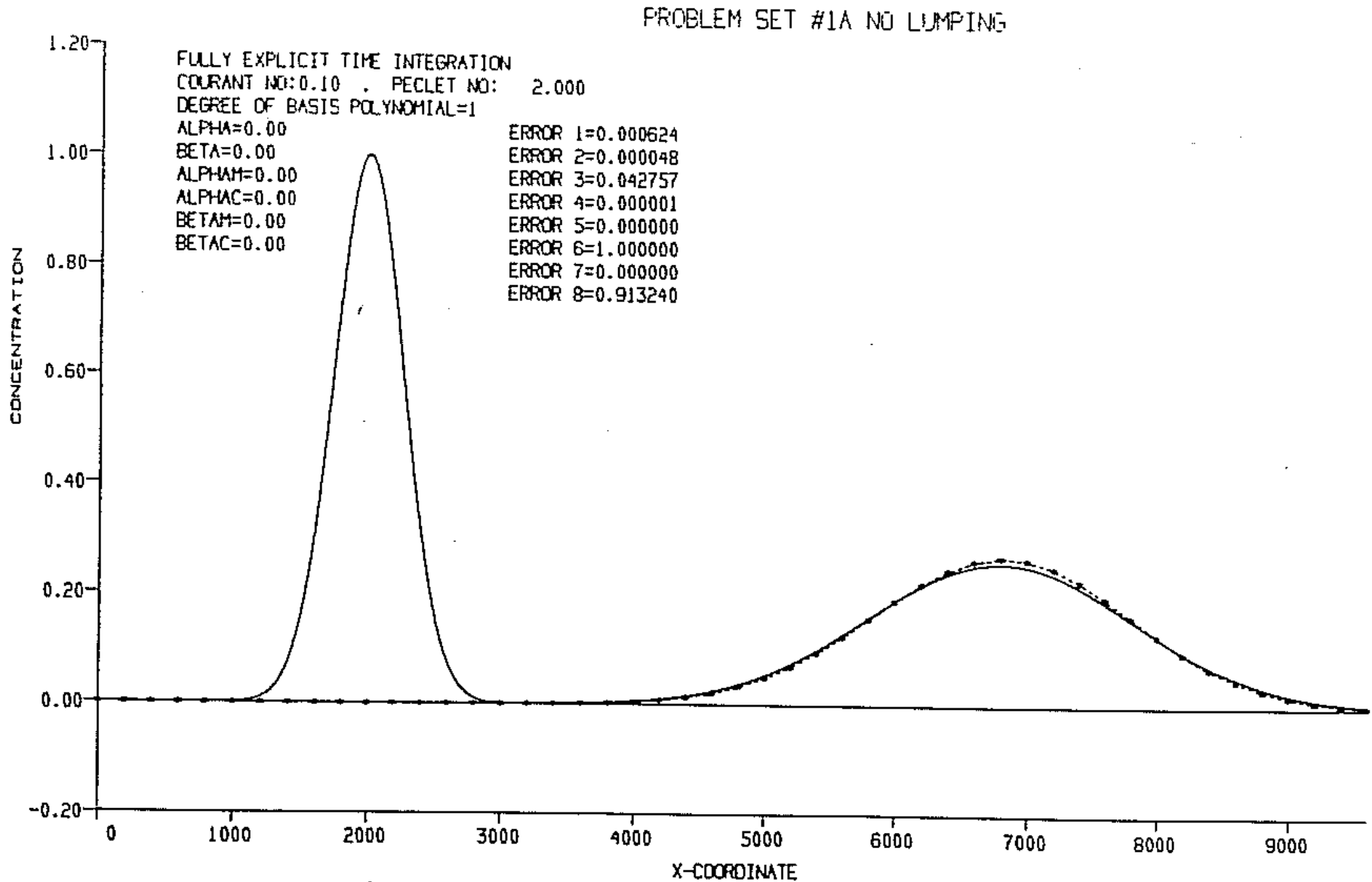


Fig. L29.4b1

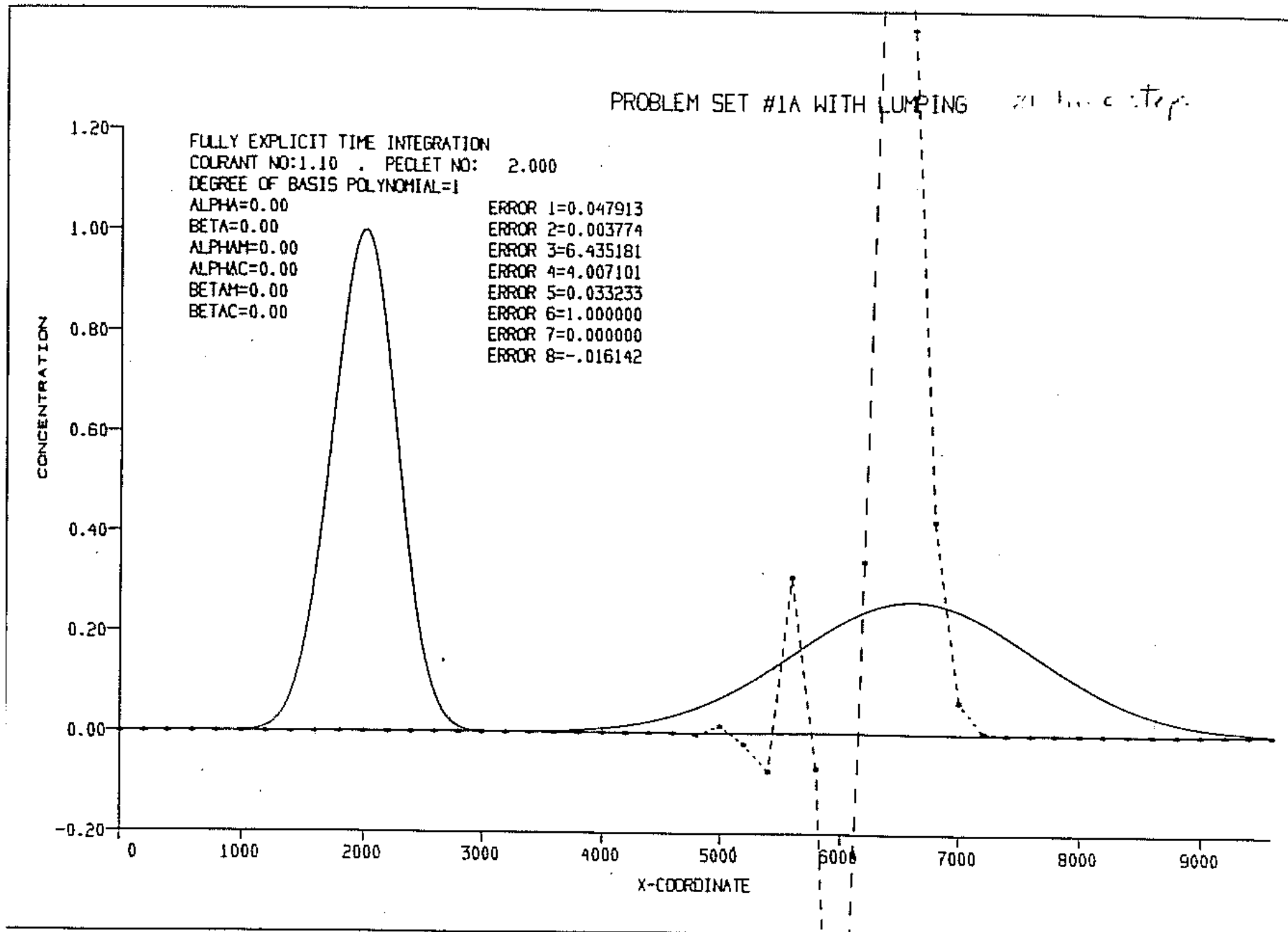


Fig. L29.4b2

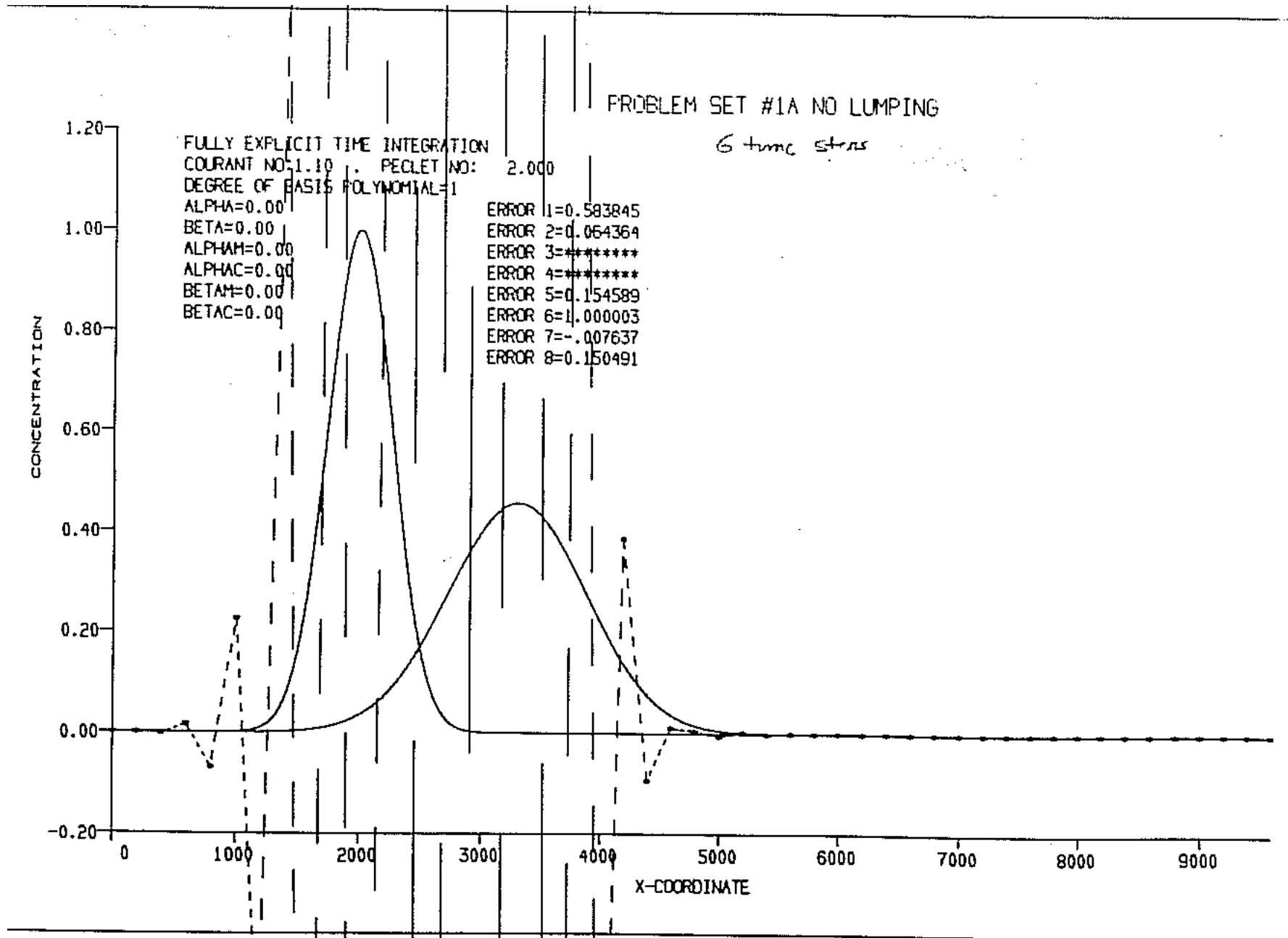


Fig. L29.5a1

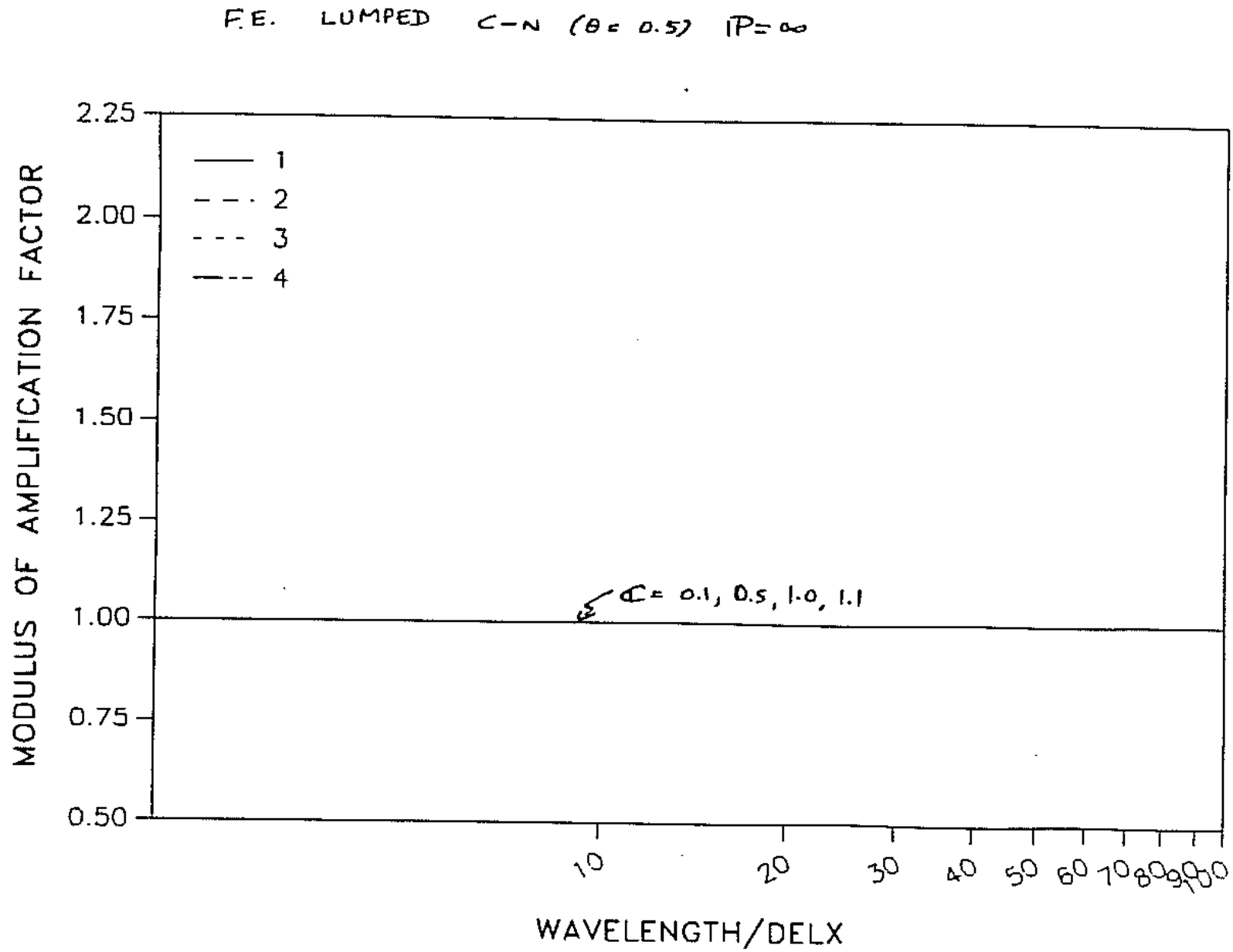


Fig. L29.5a2

F.E. CONSISTENT, C-N ($\theta=0.5$) $P=\infty$

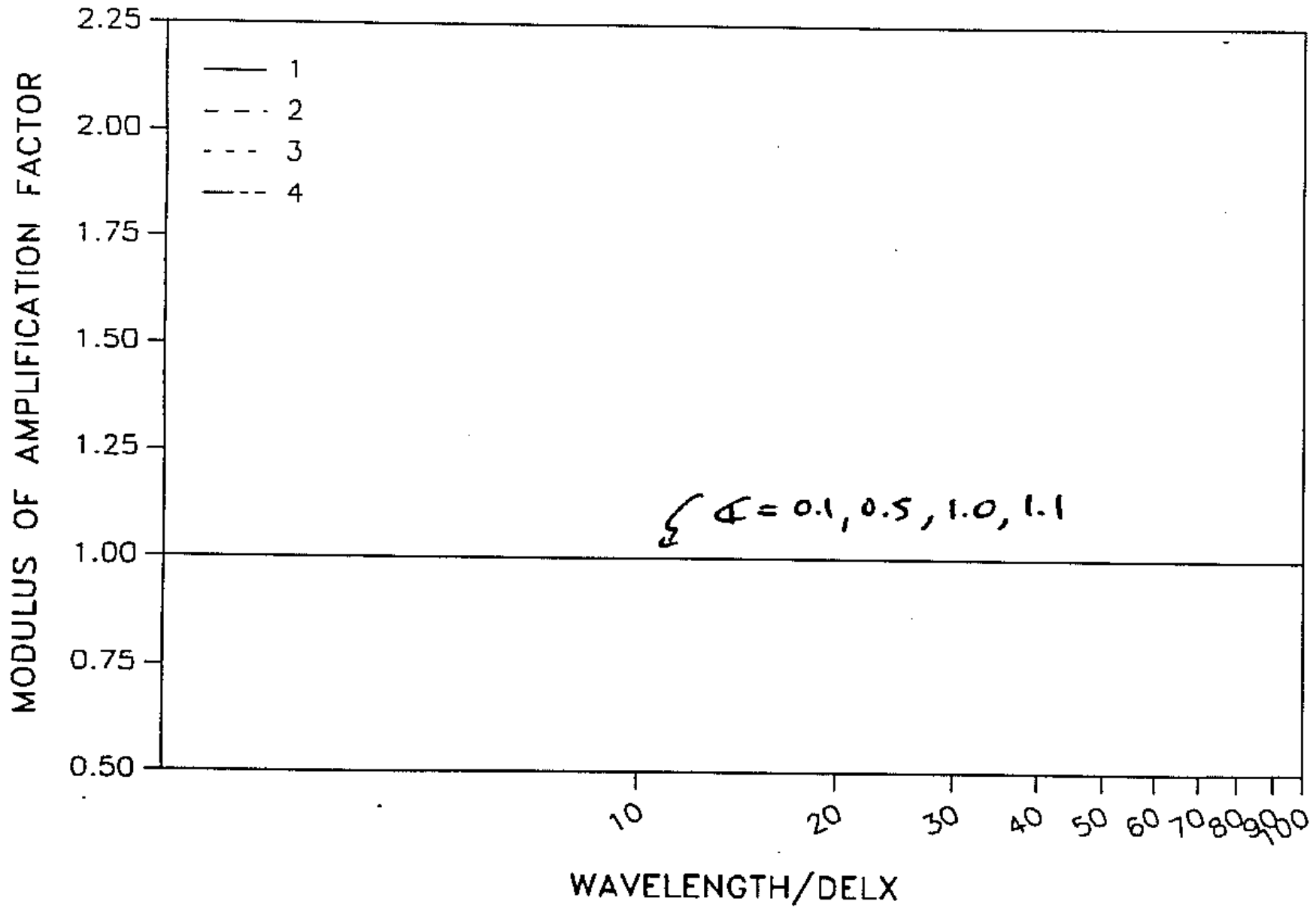


Fig. L29.5b1

F.E. LUMPED C-N ($\theta = 0.5$) $P = \infty$

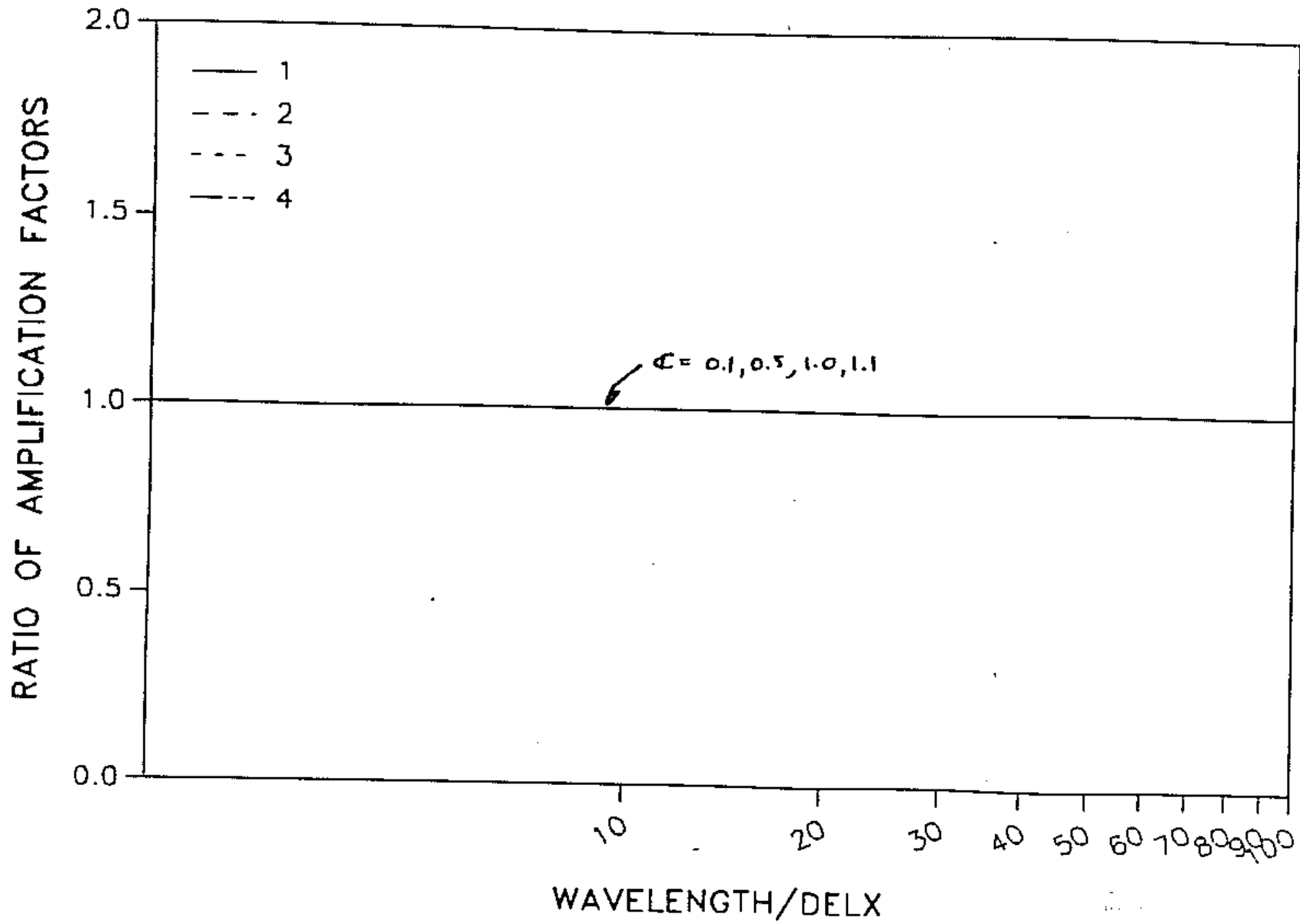


Fig. L29.5b2

F.E. CONSISTENT, C-N ($\theta = 0.5$) $IP = \infty$

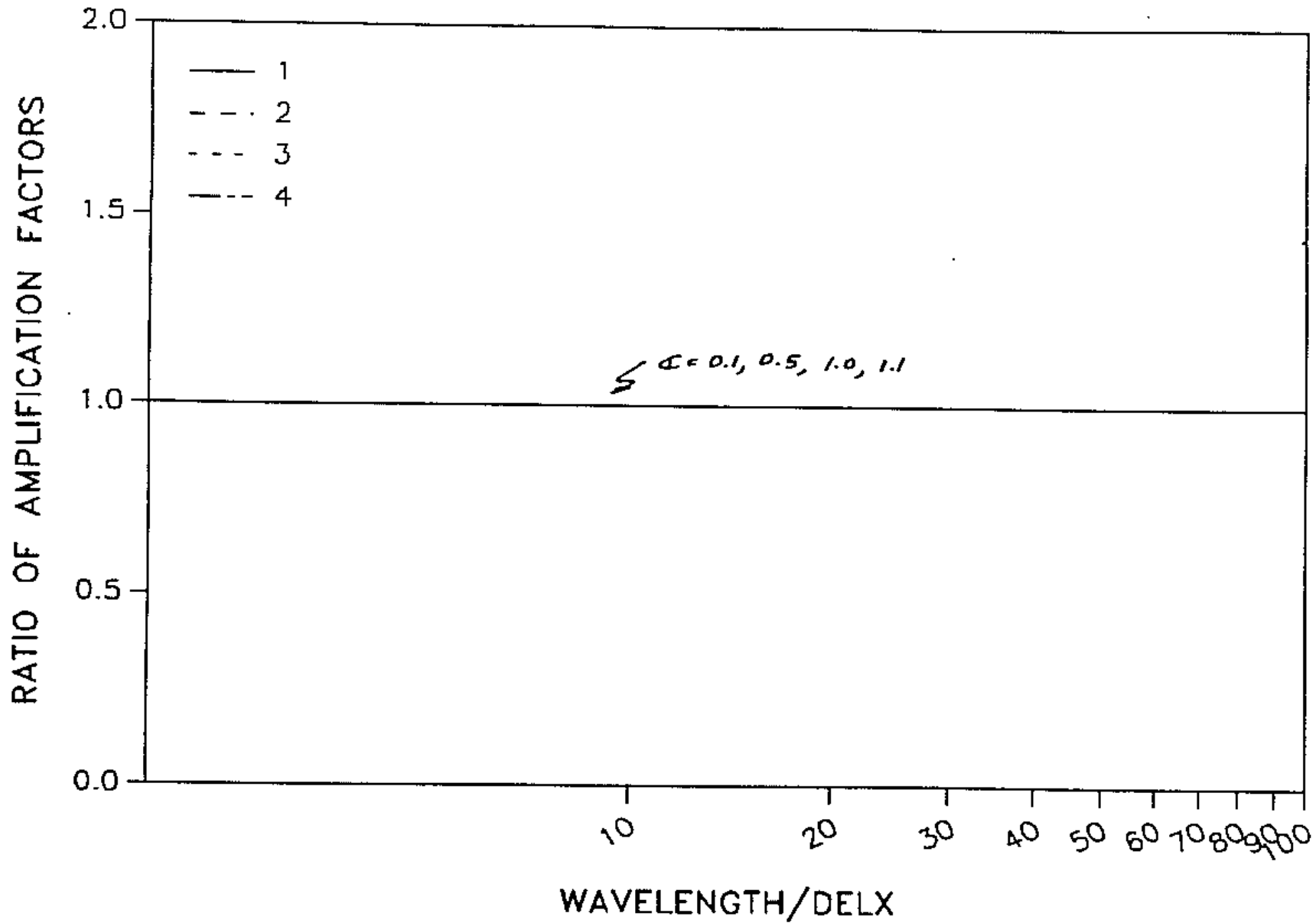


Fig. L29.5c1

F.E. LUMPED C-N ($\theta = 0.5$) $IP = \infty$

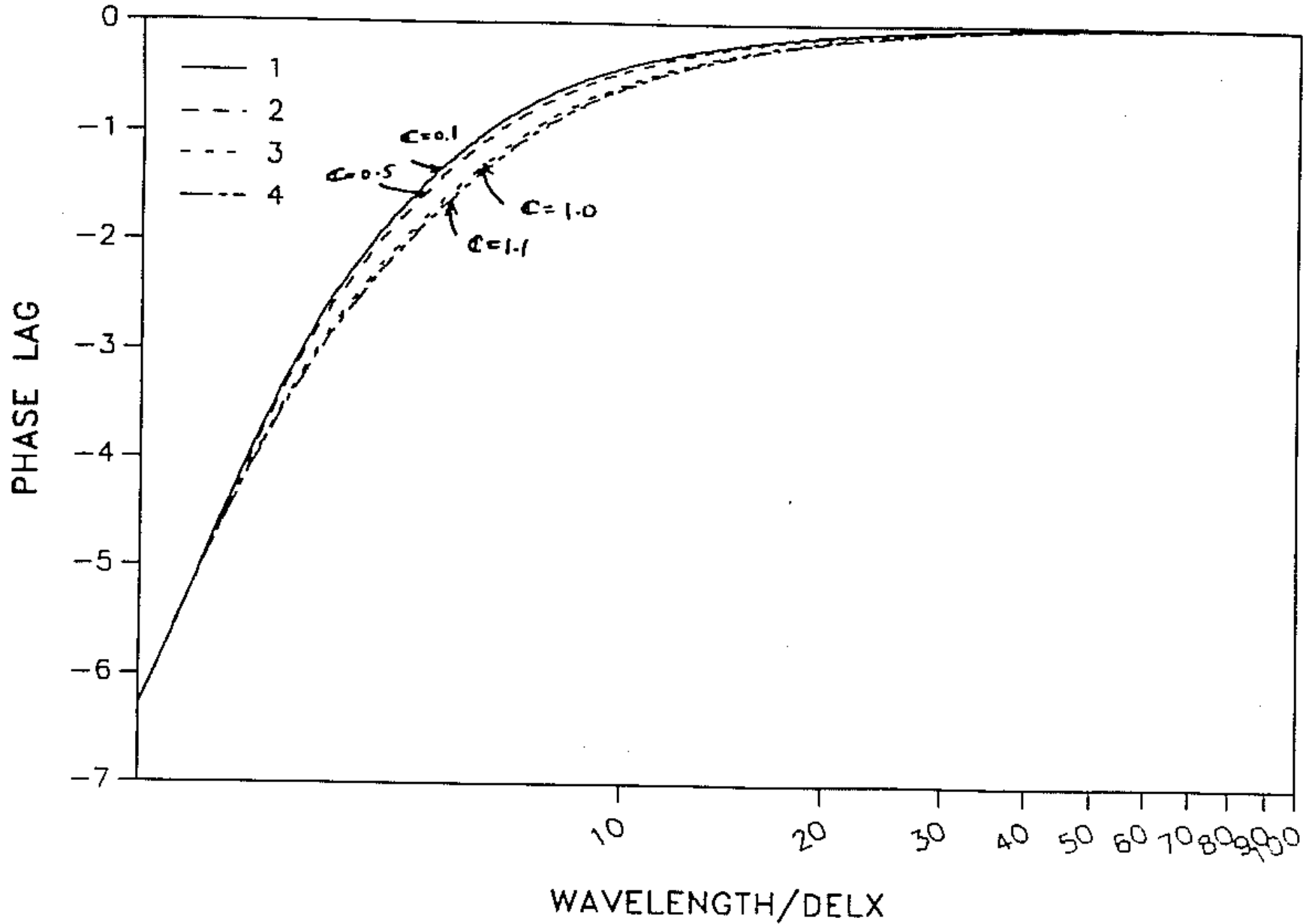


Fig. L29.5c2

F.E. CONSISTENT, C-N ($\theta=0.5$) $iP = \infty$

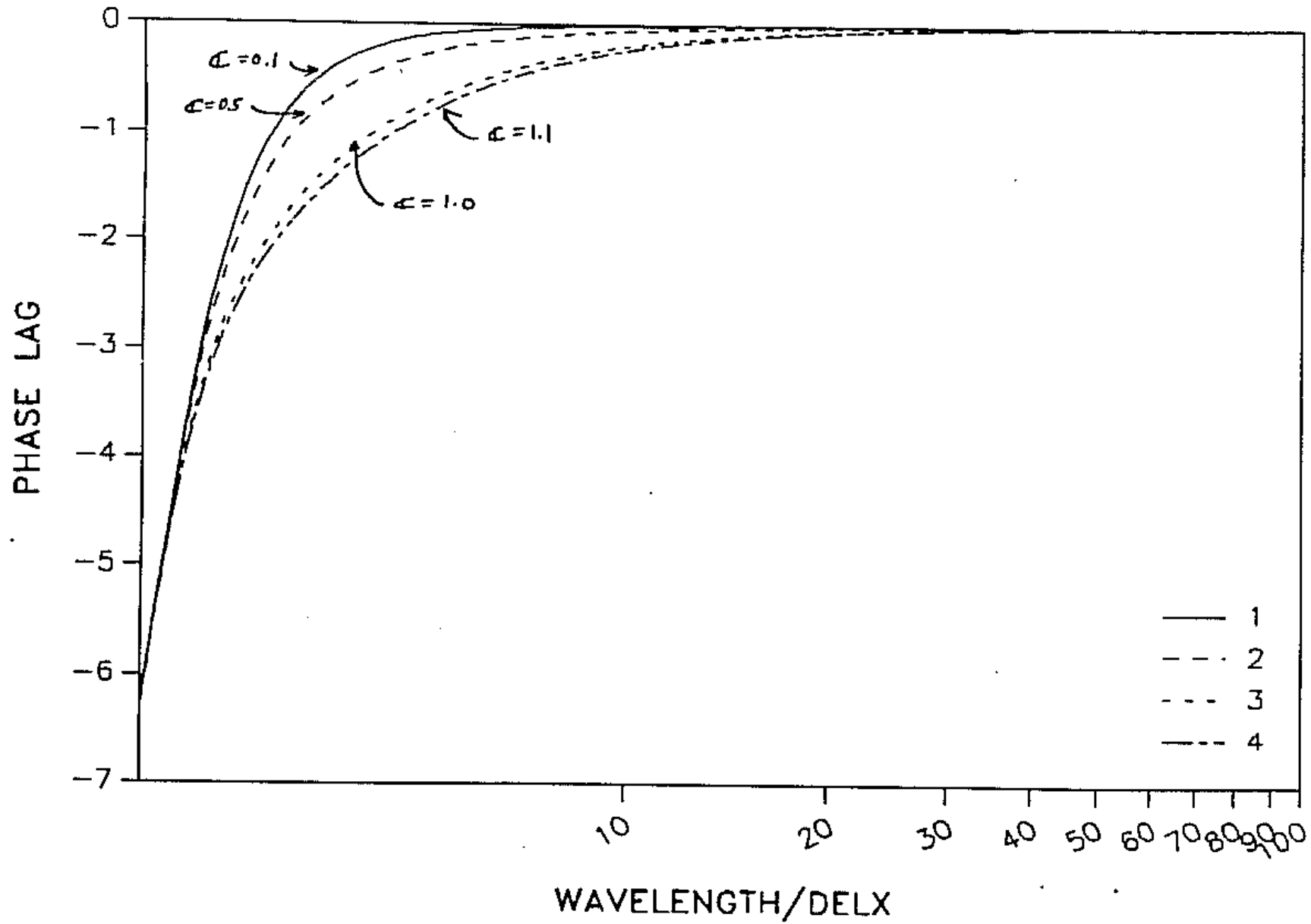


Fig. L29.6a1

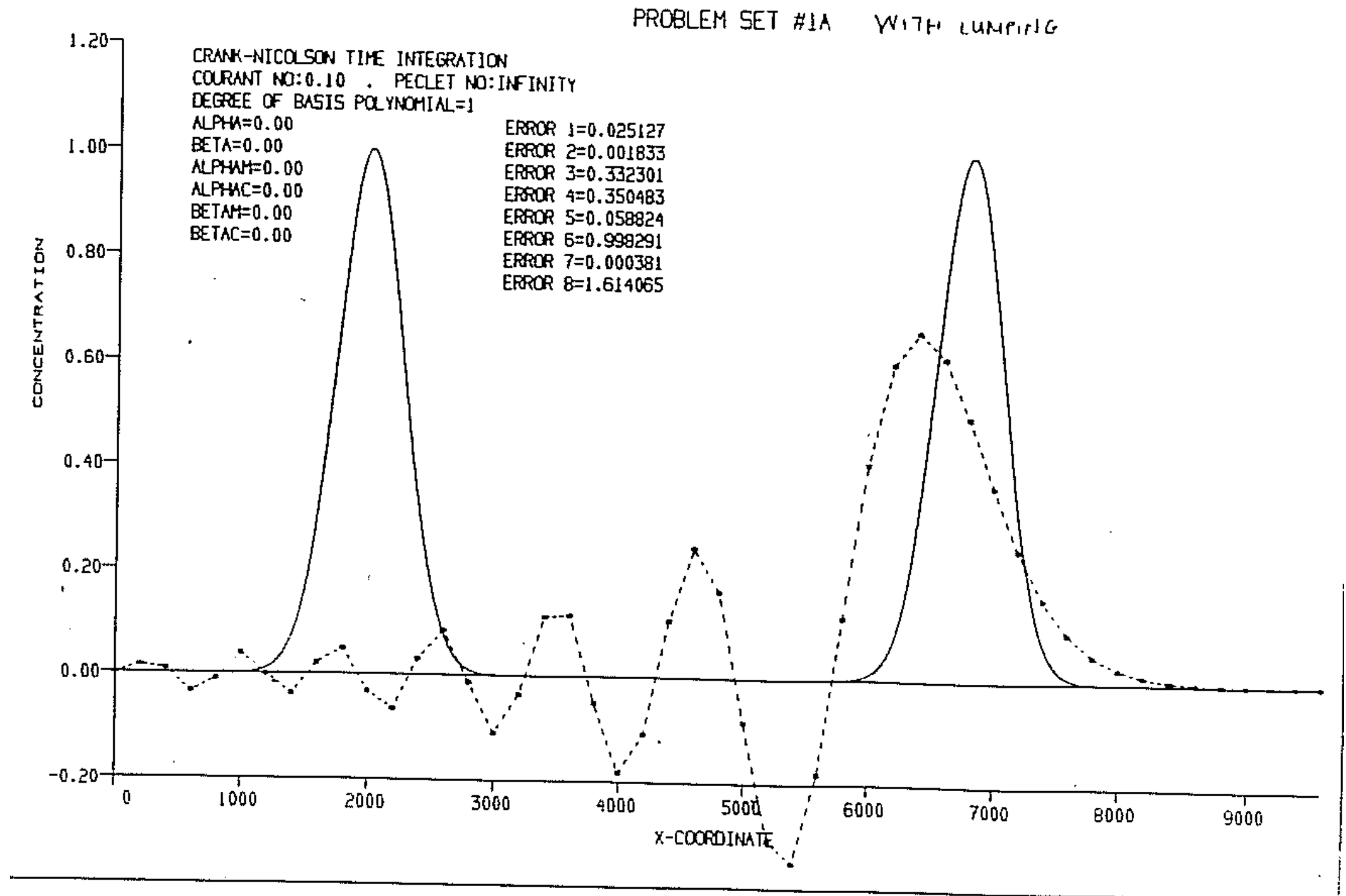


Fig. L29.6a2

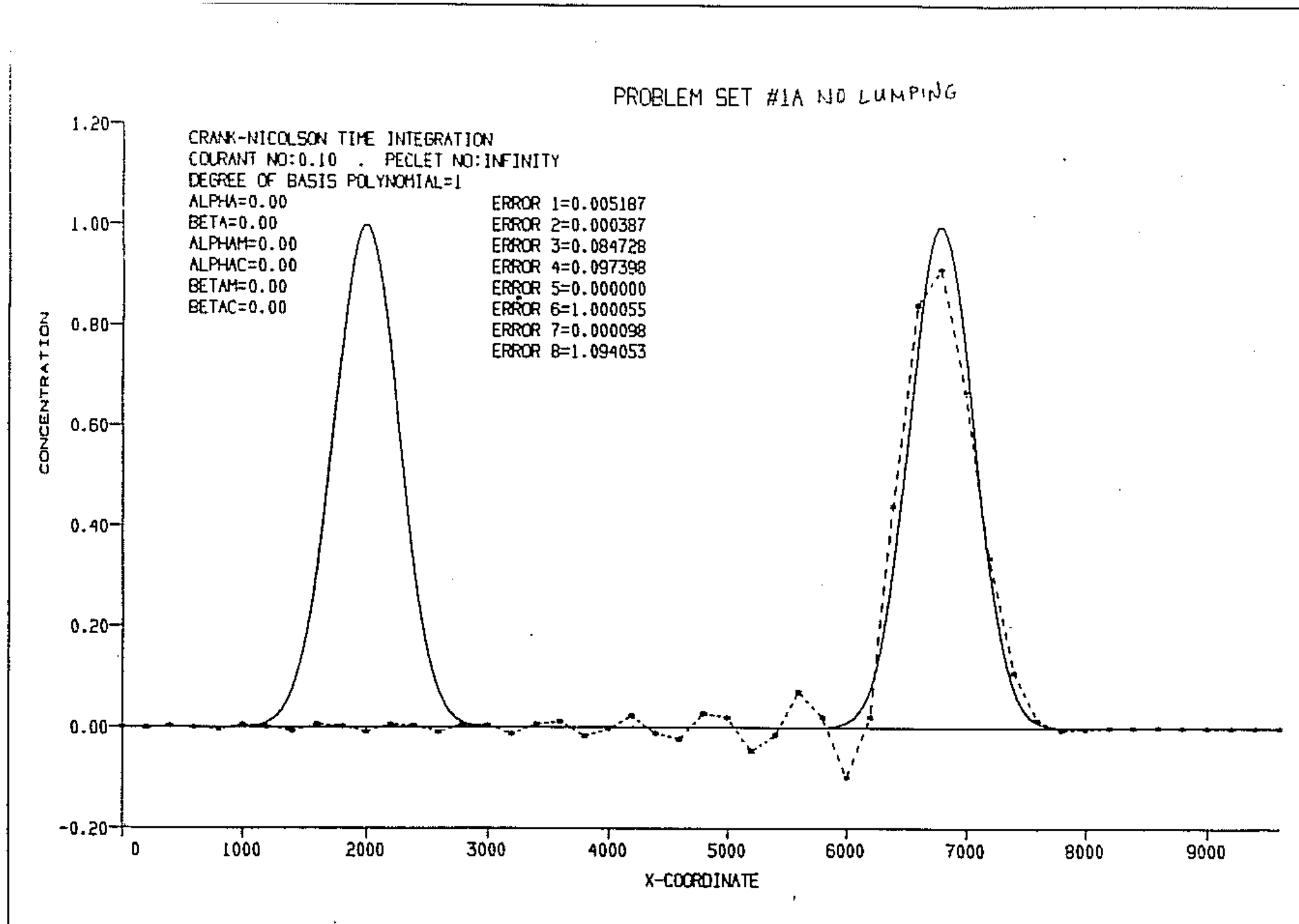


Fig. L29.6b1

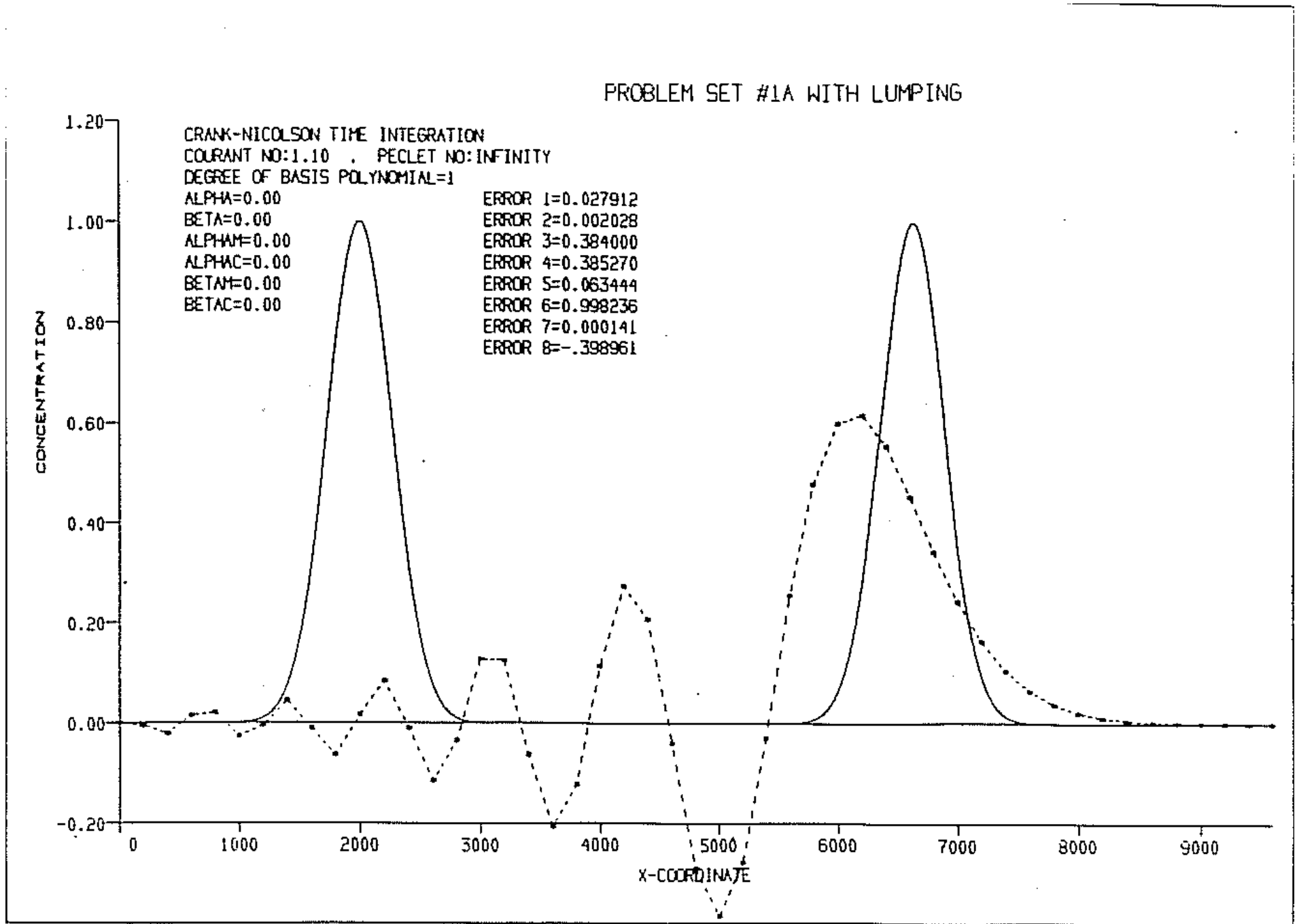


Fig. L29.6b2

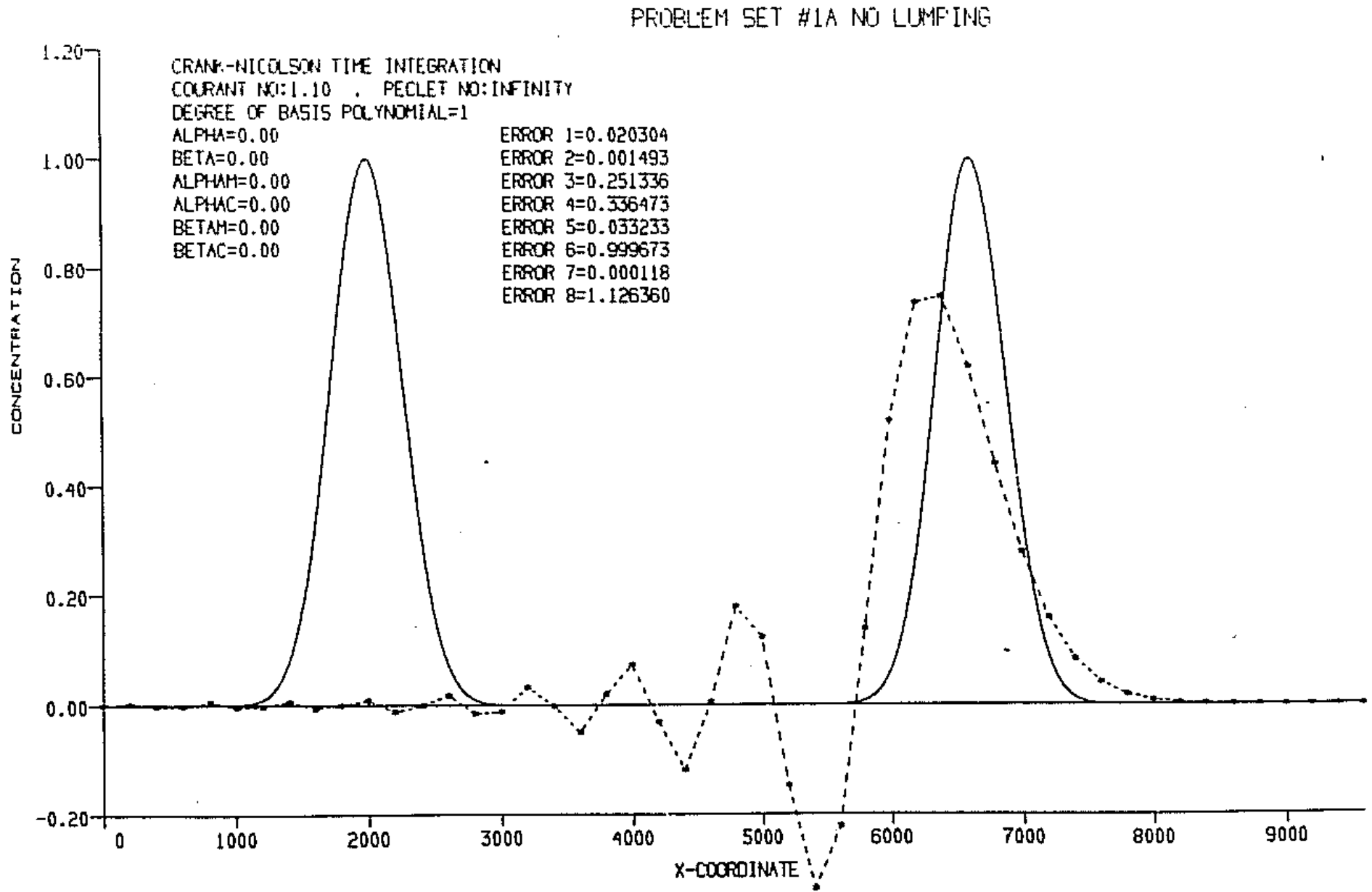


Fig. L29.7a1

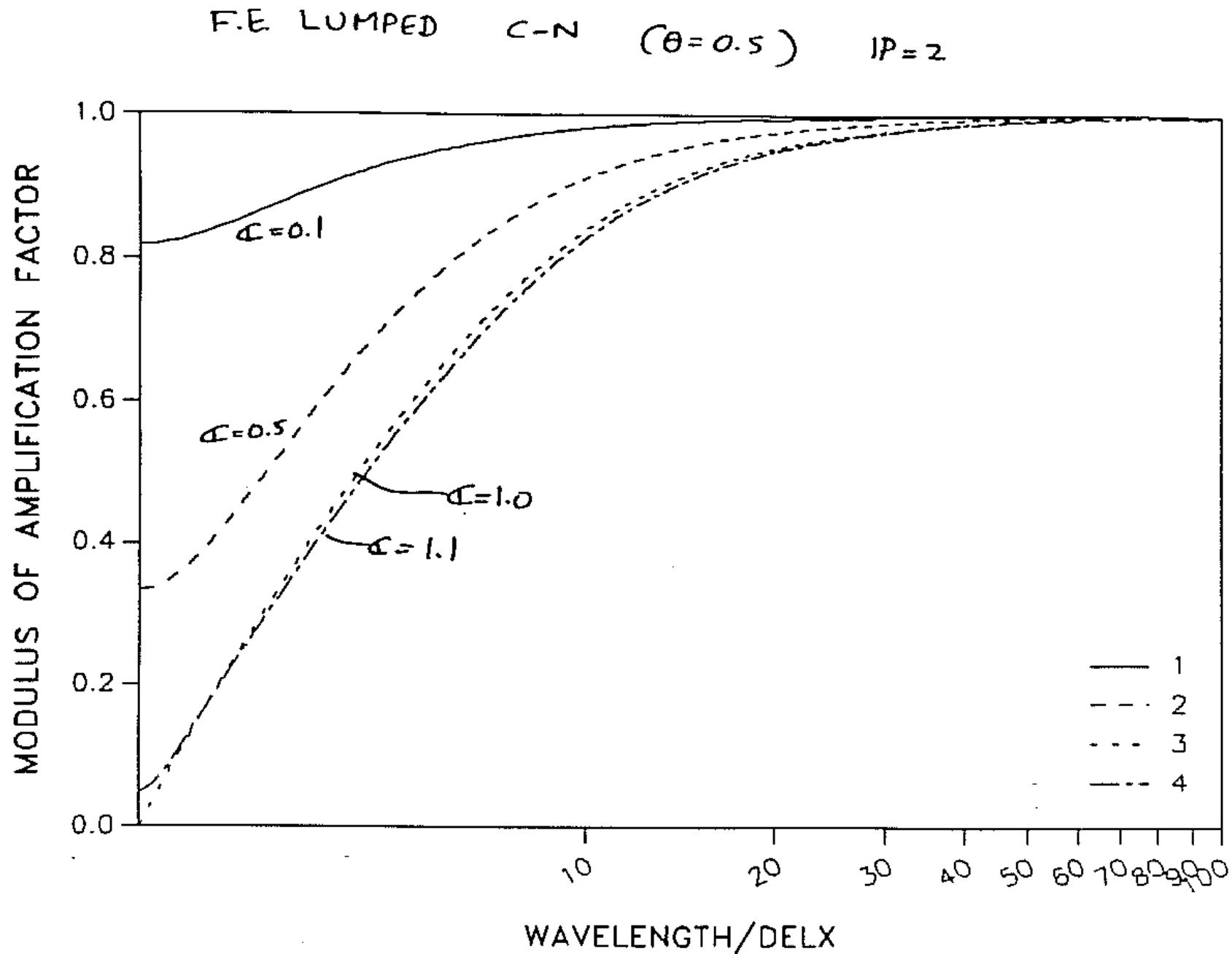


Fig. L29.7a2

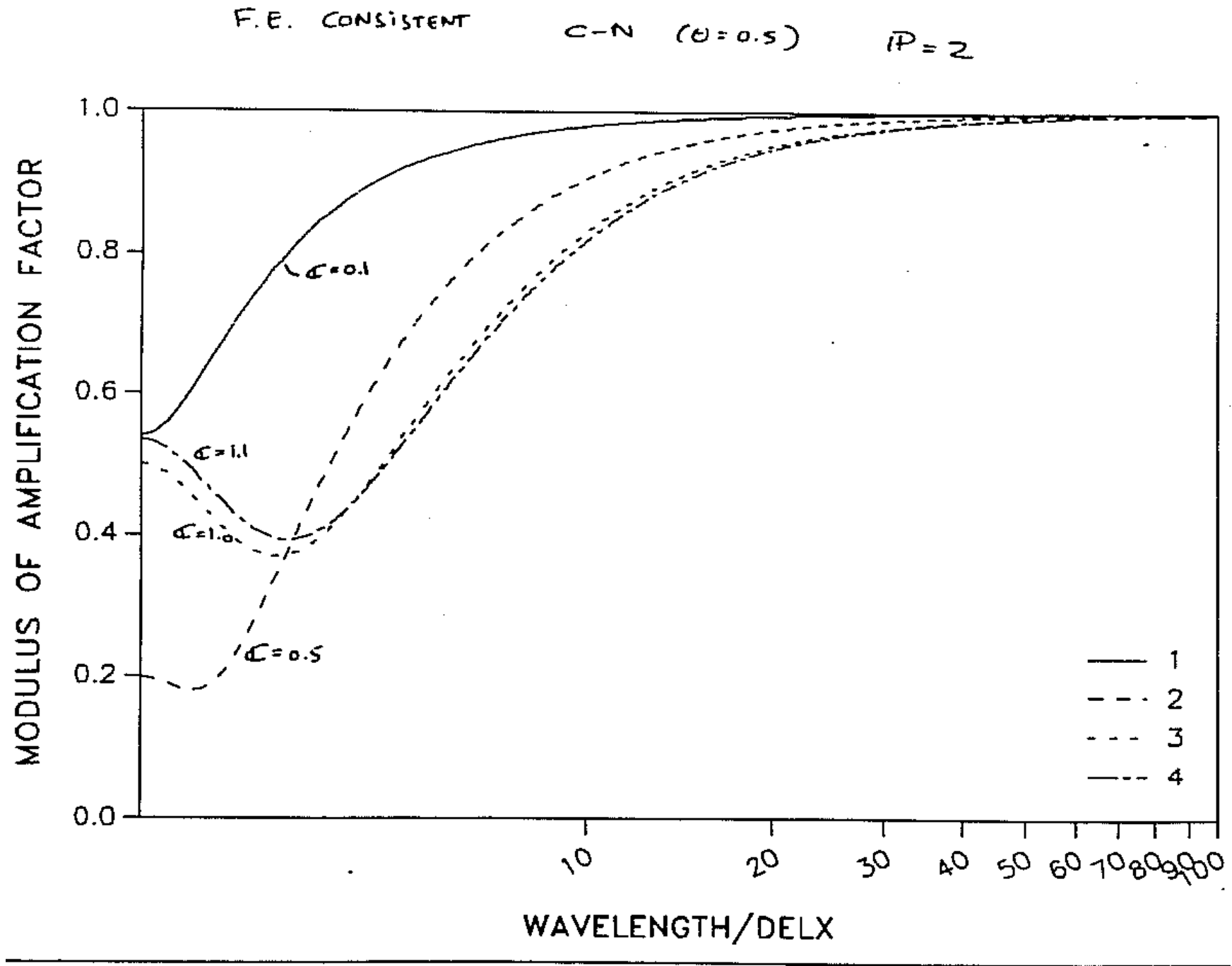


Fig. L29.7b1

F.E. LUMPED

C-N ($\theta = 0.5$)

$P = 2$

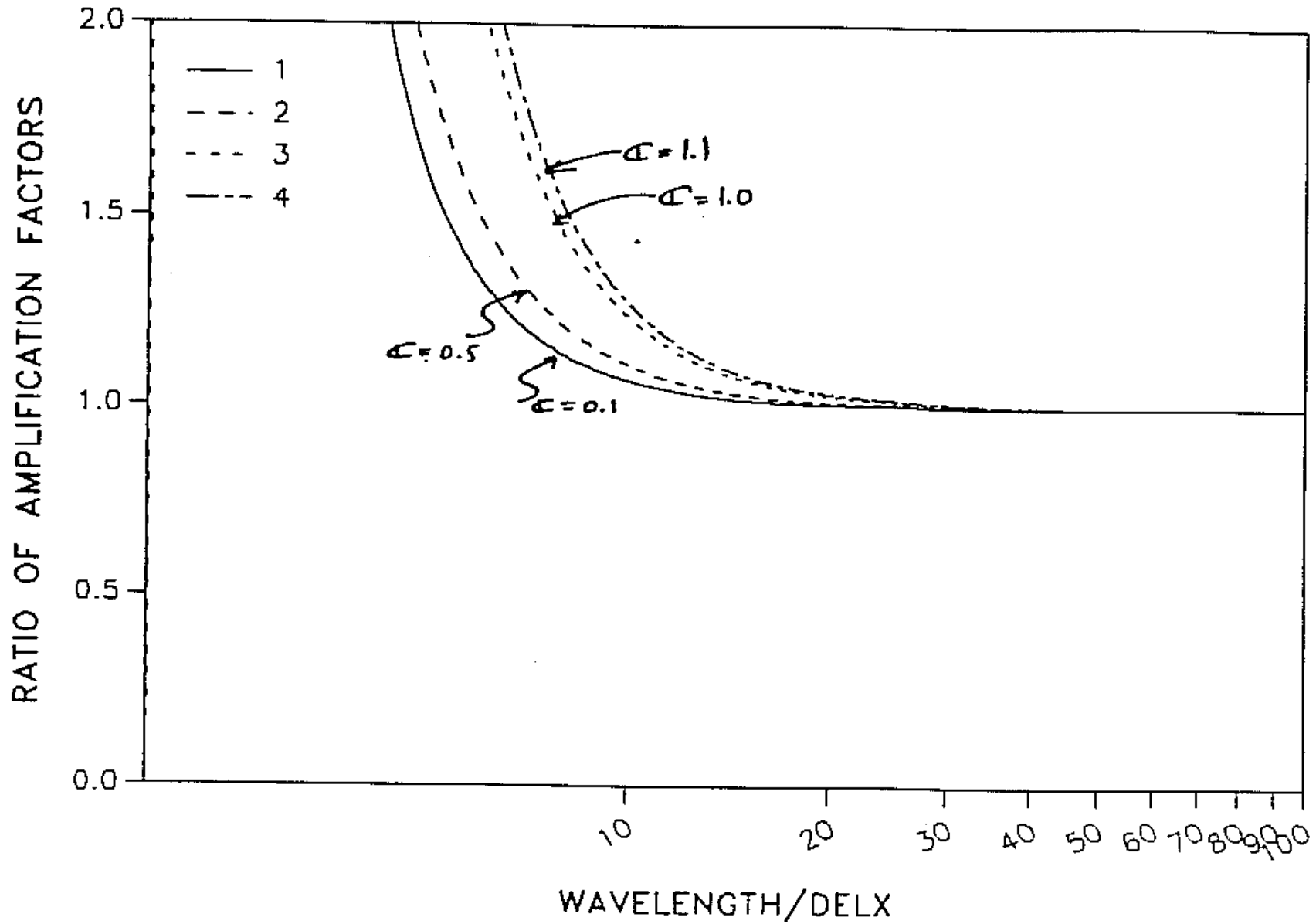


Fig. L29.7b2

F.E. CONSISTENT , C-N ($\theta = 0.5$) $IP = 2$

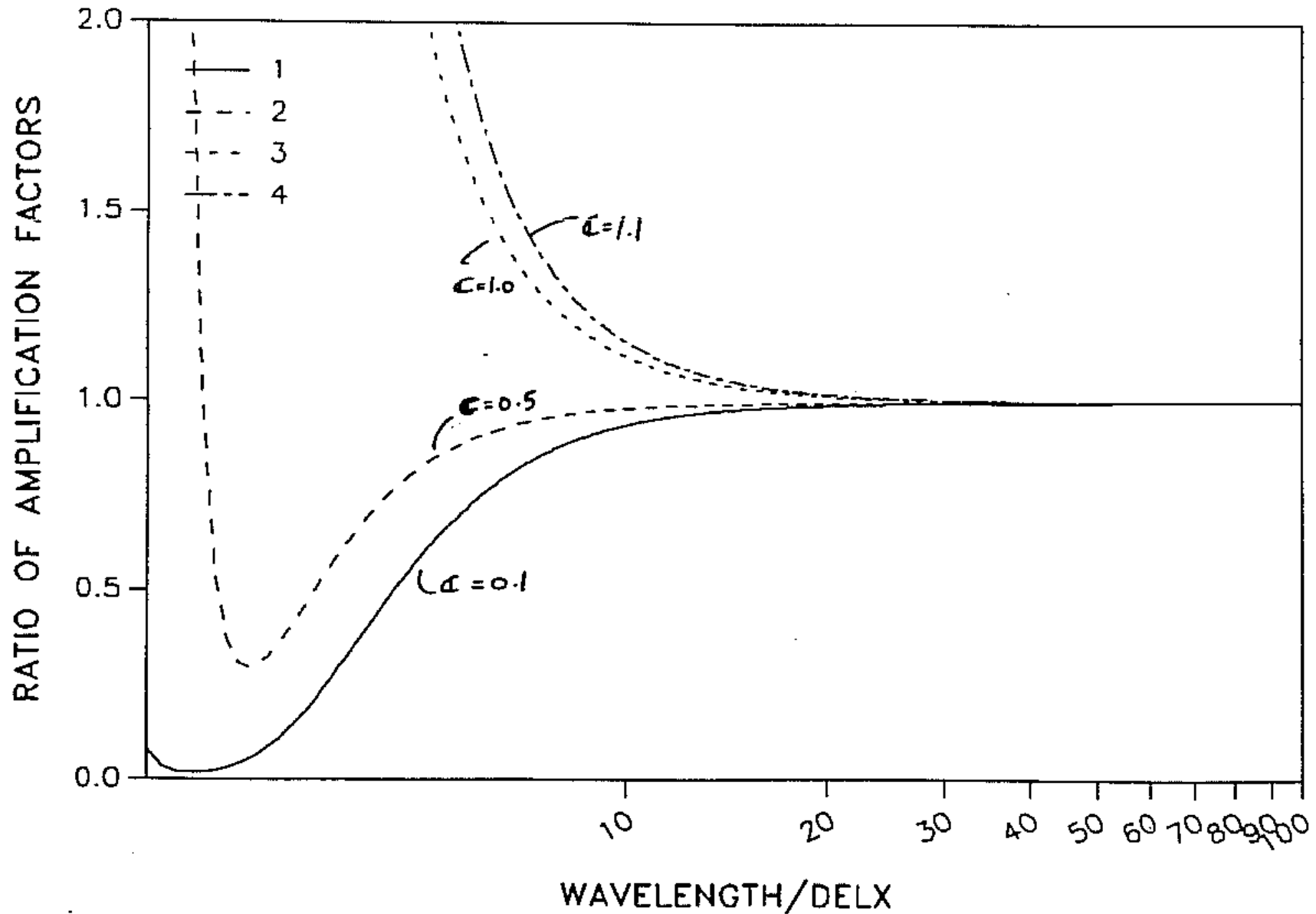


Fig. L29.7c1

F.E. LUMPED C-N ($\theta = 0.5$) $P = 2$

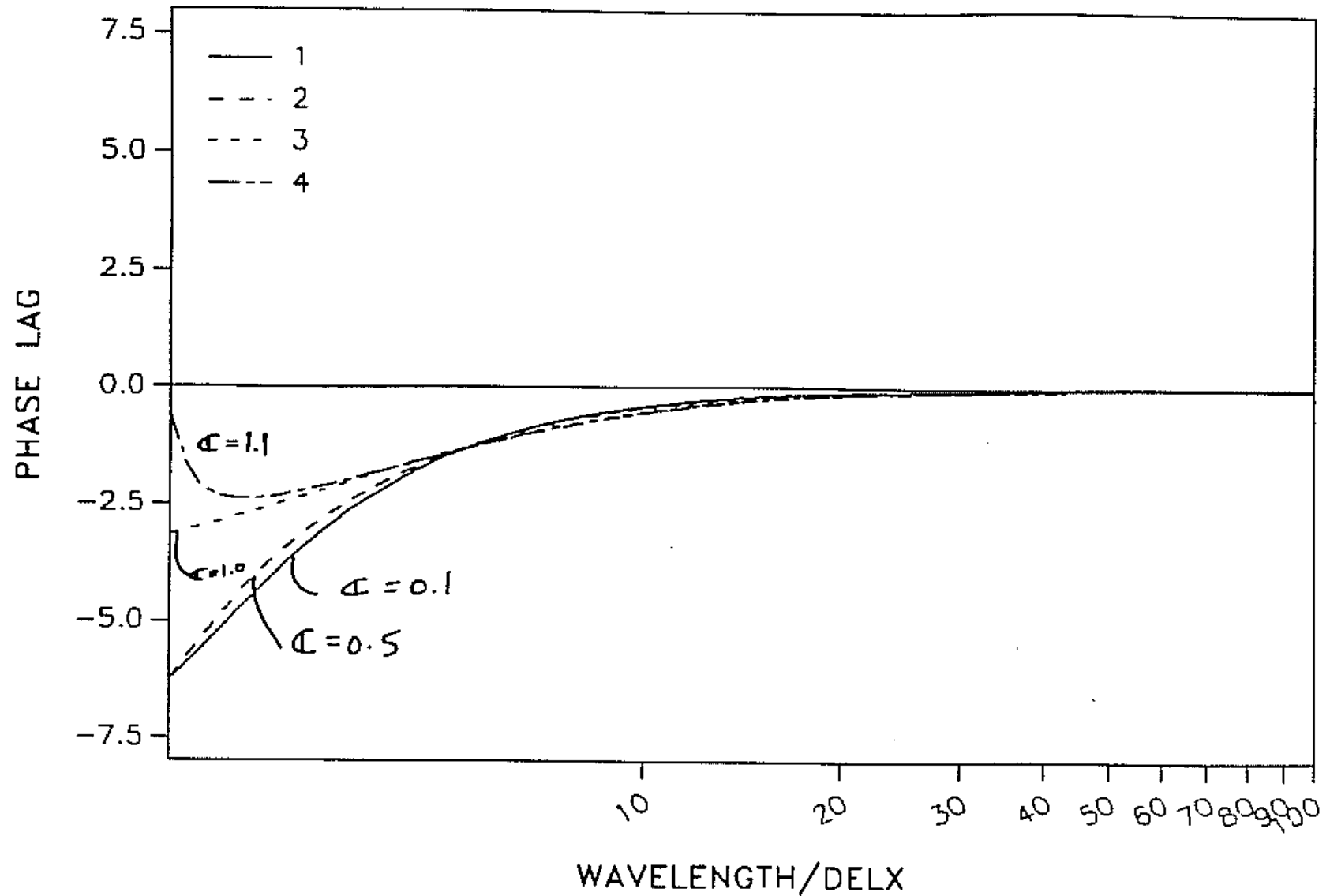


Fig. L29.7c2

F.E. CONSISTENT , C-N ($\theta = 0.5$) $P = 2$

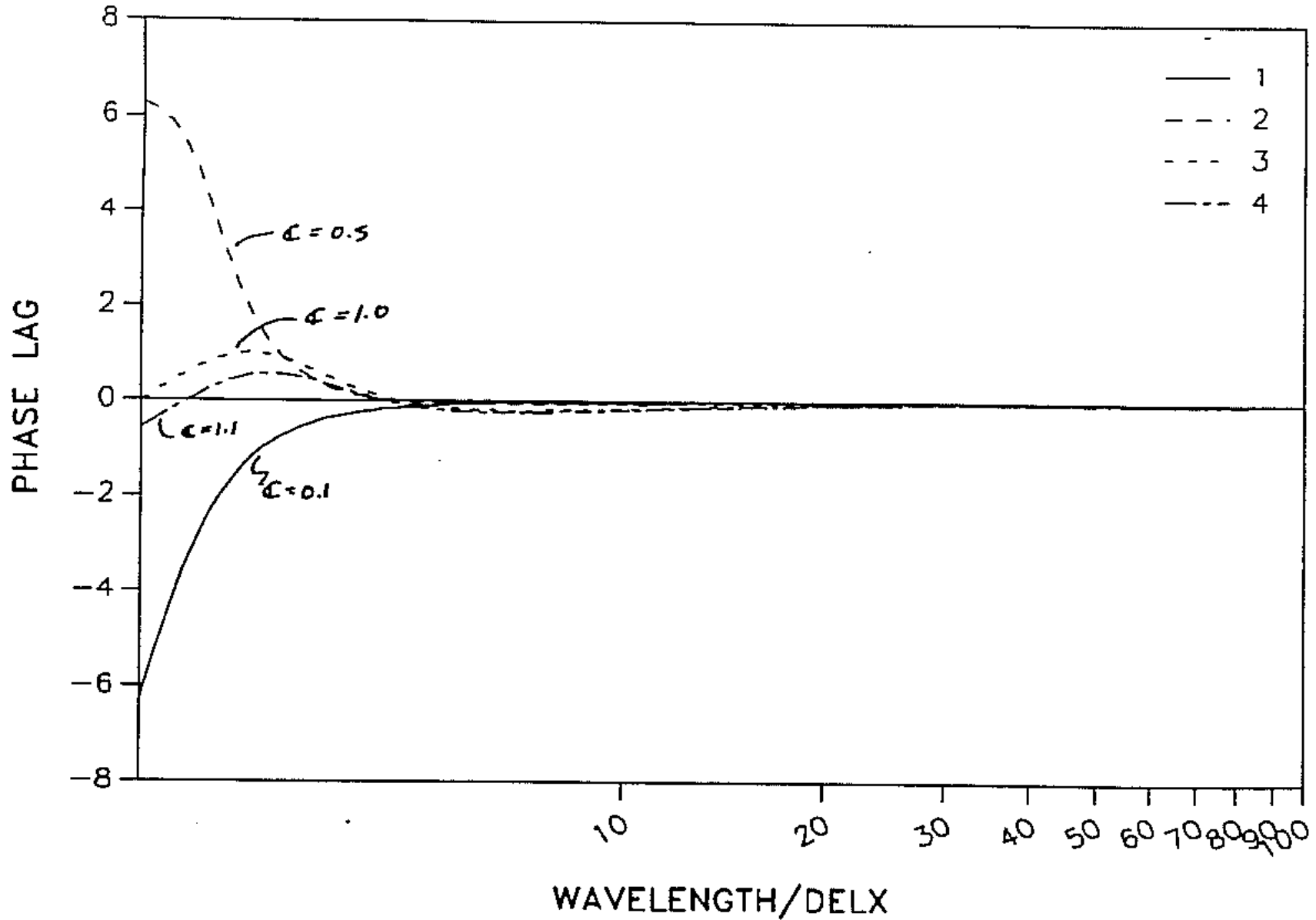


Fig. L29.8a1

PROBLEM SET #1A WITH LUMPING

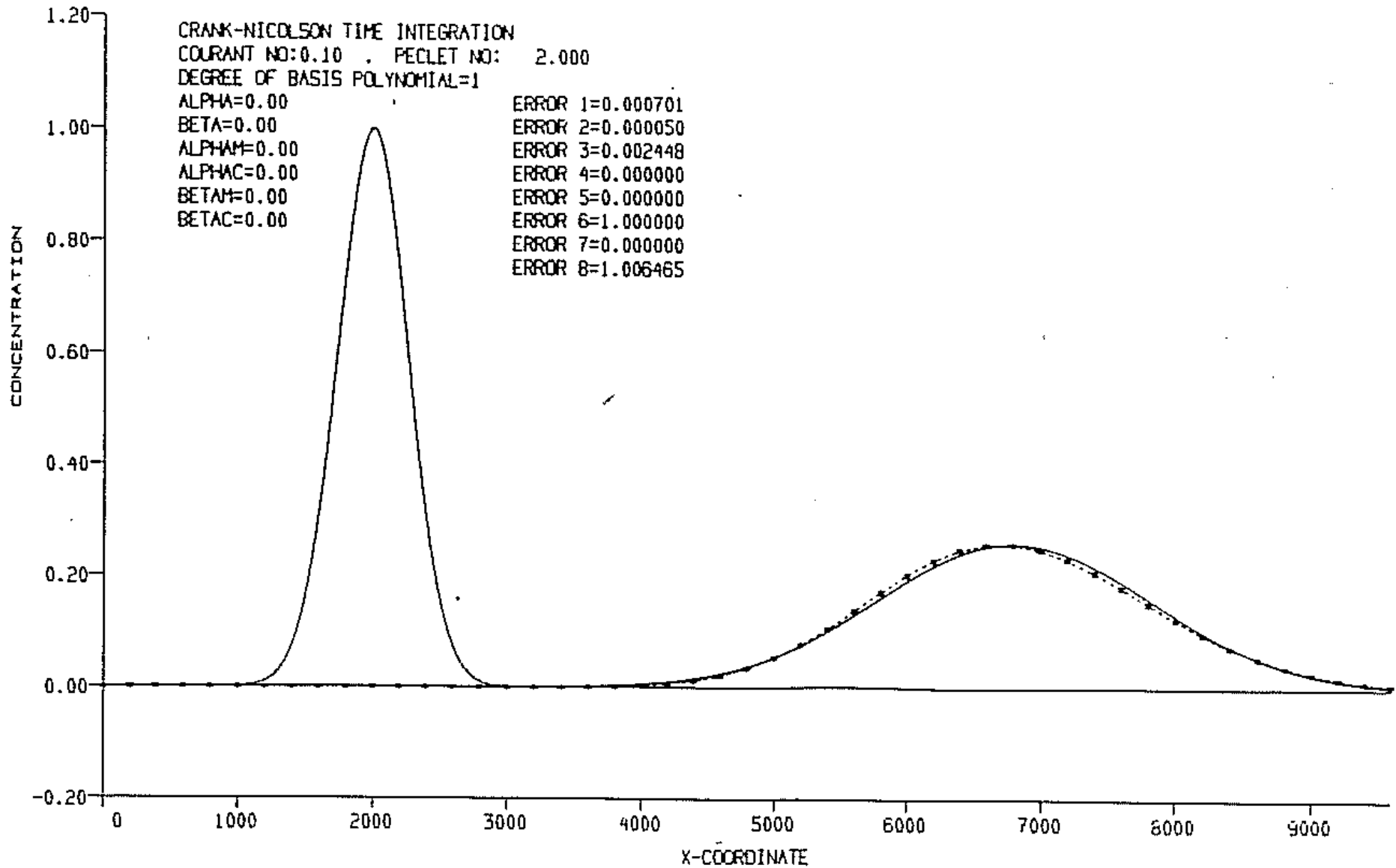


Fig. L29.8a2

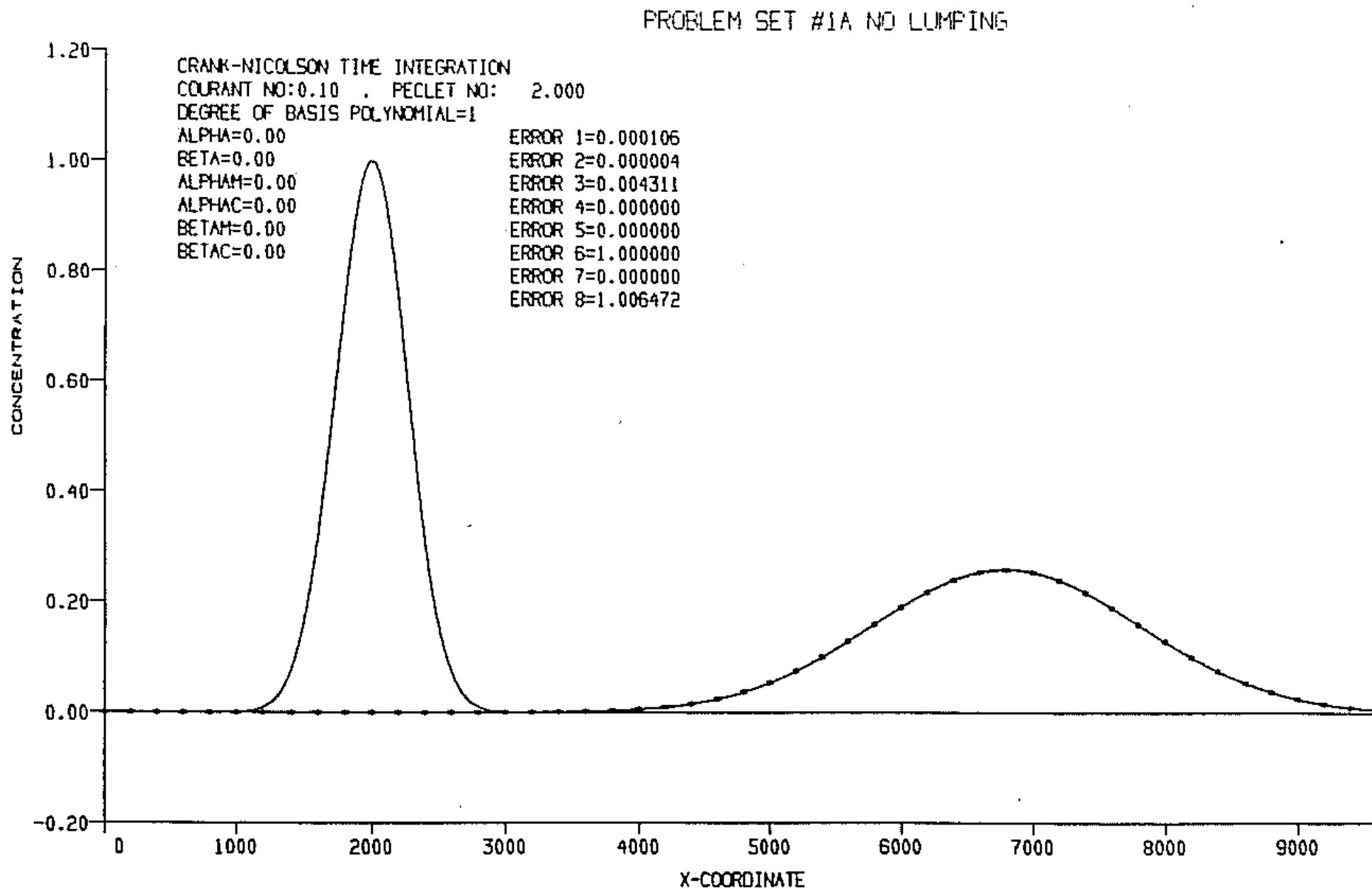


Fig. L29.8b1

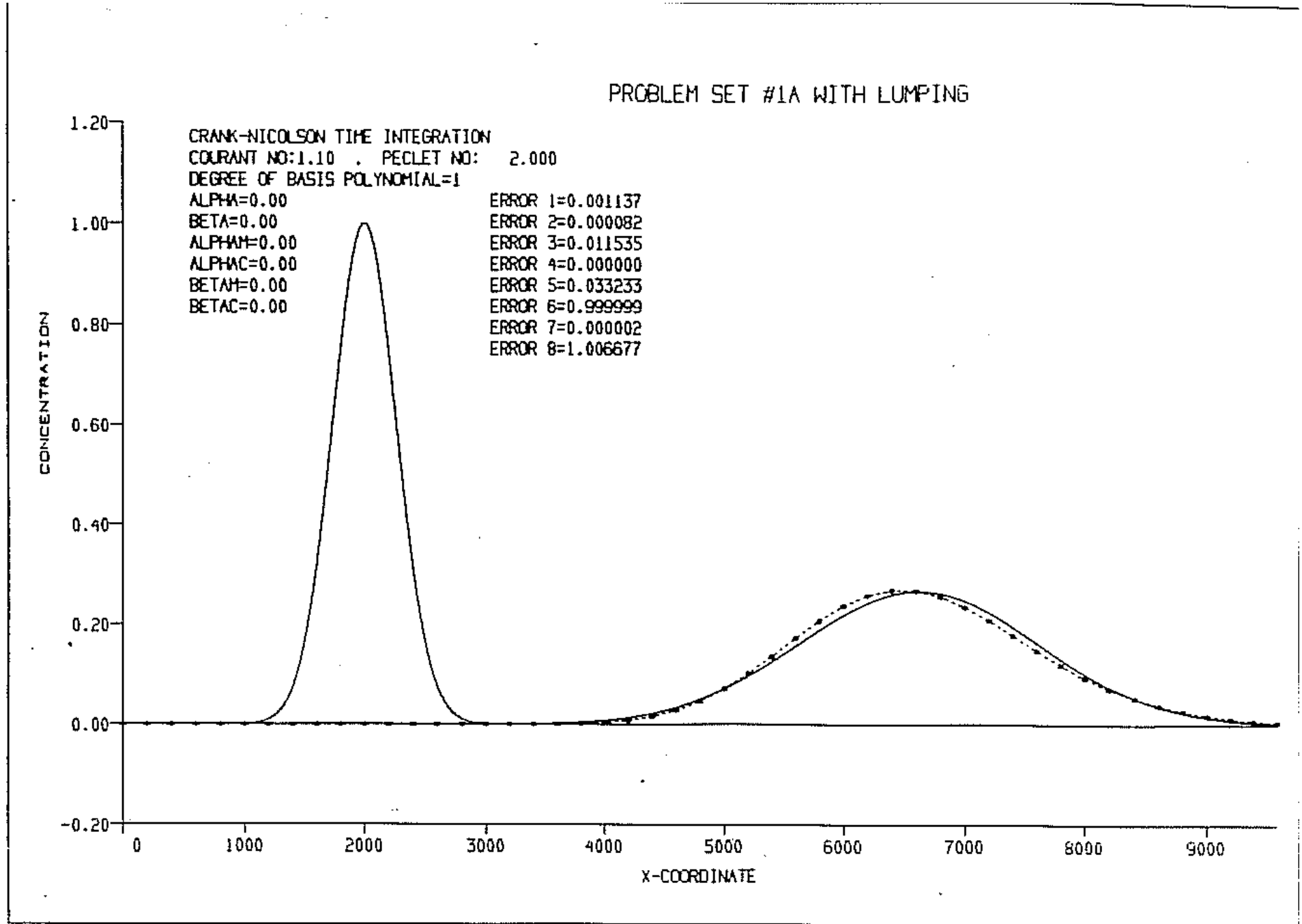


Fig. L29.8b2

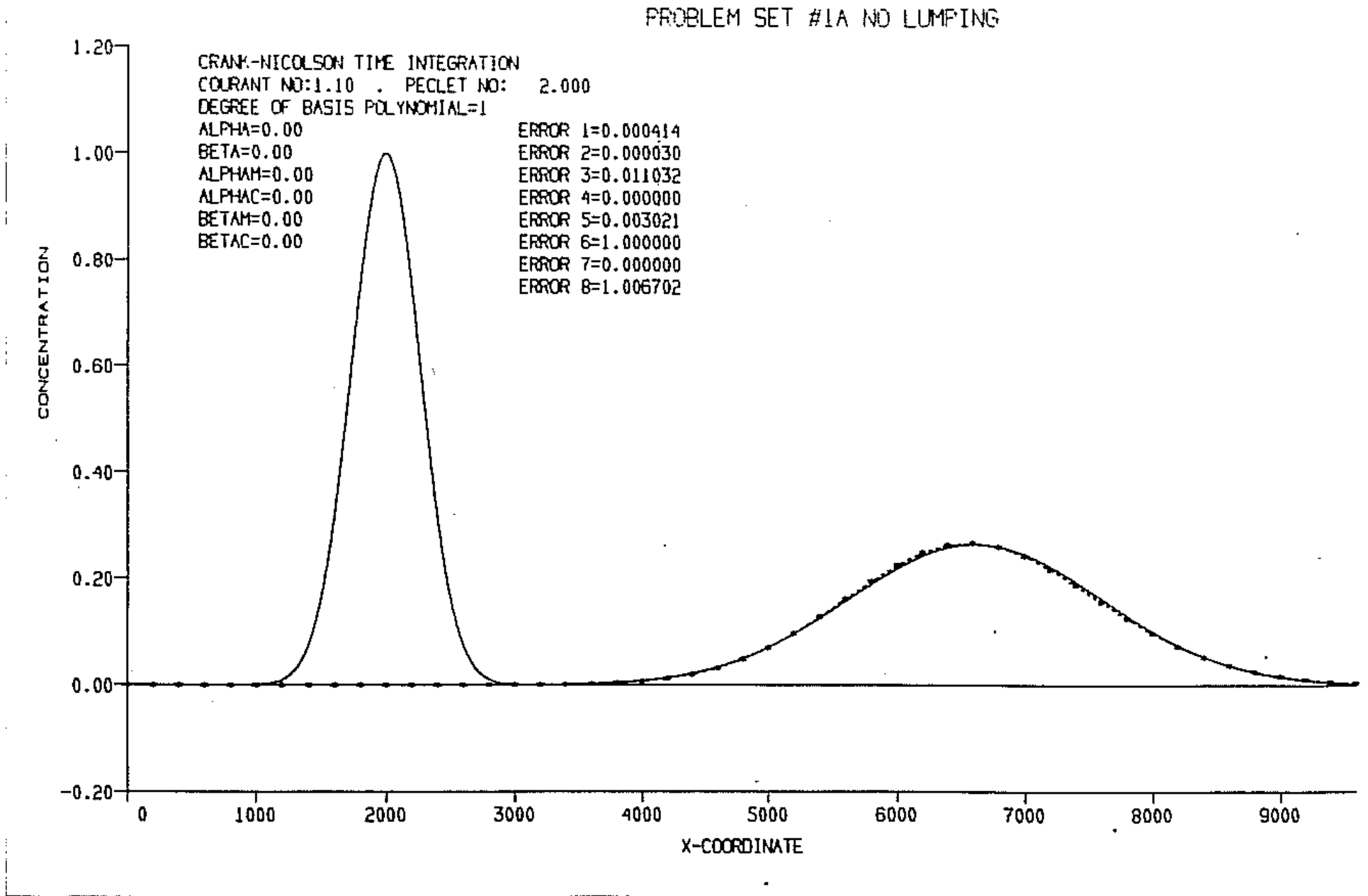


Fig. L29.9a1

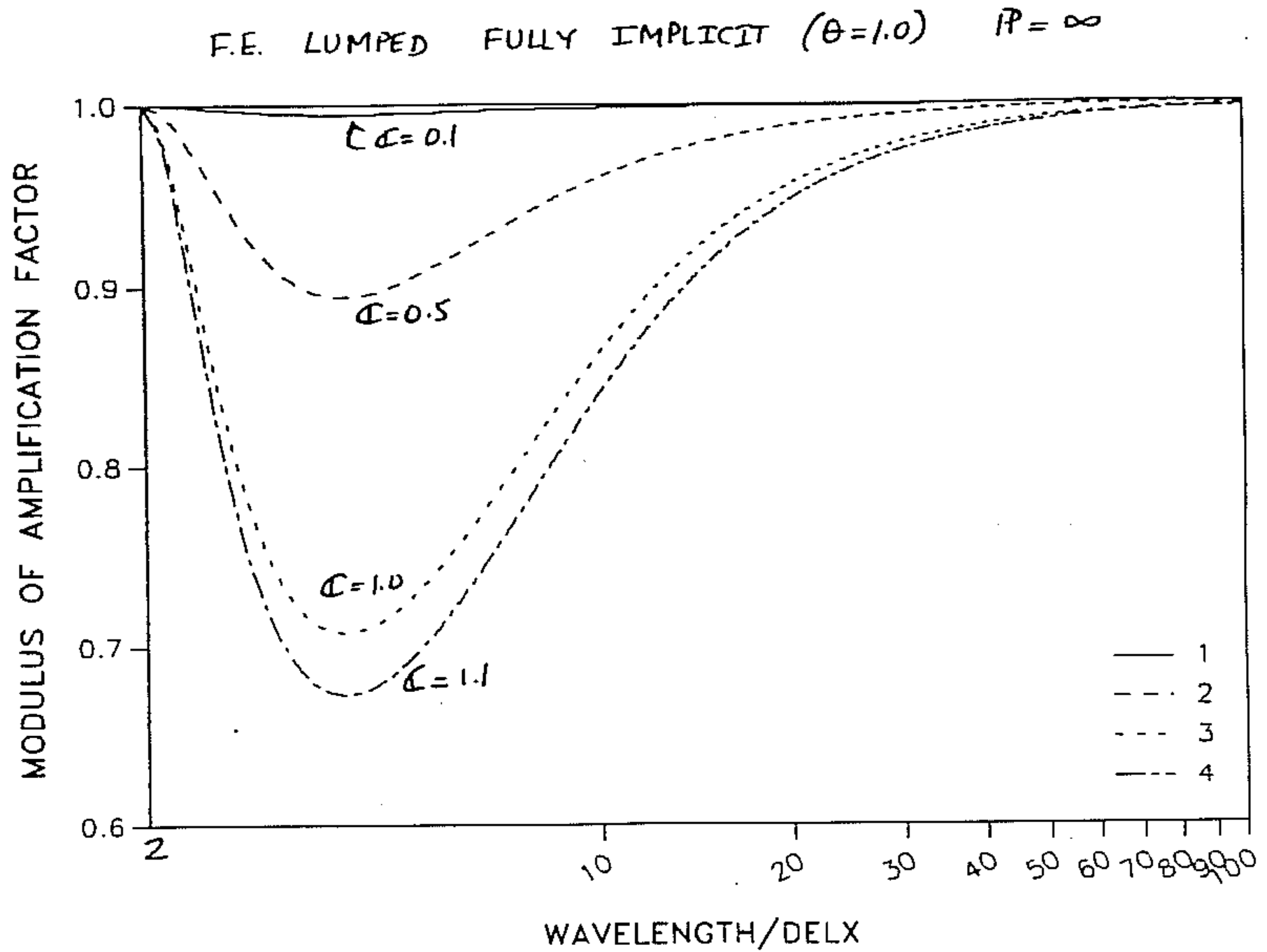


Fig. L29.9a2

F.E. CONSISTENT FULLY IMPLICIT ($\theta=1.0$) $\rho=0$

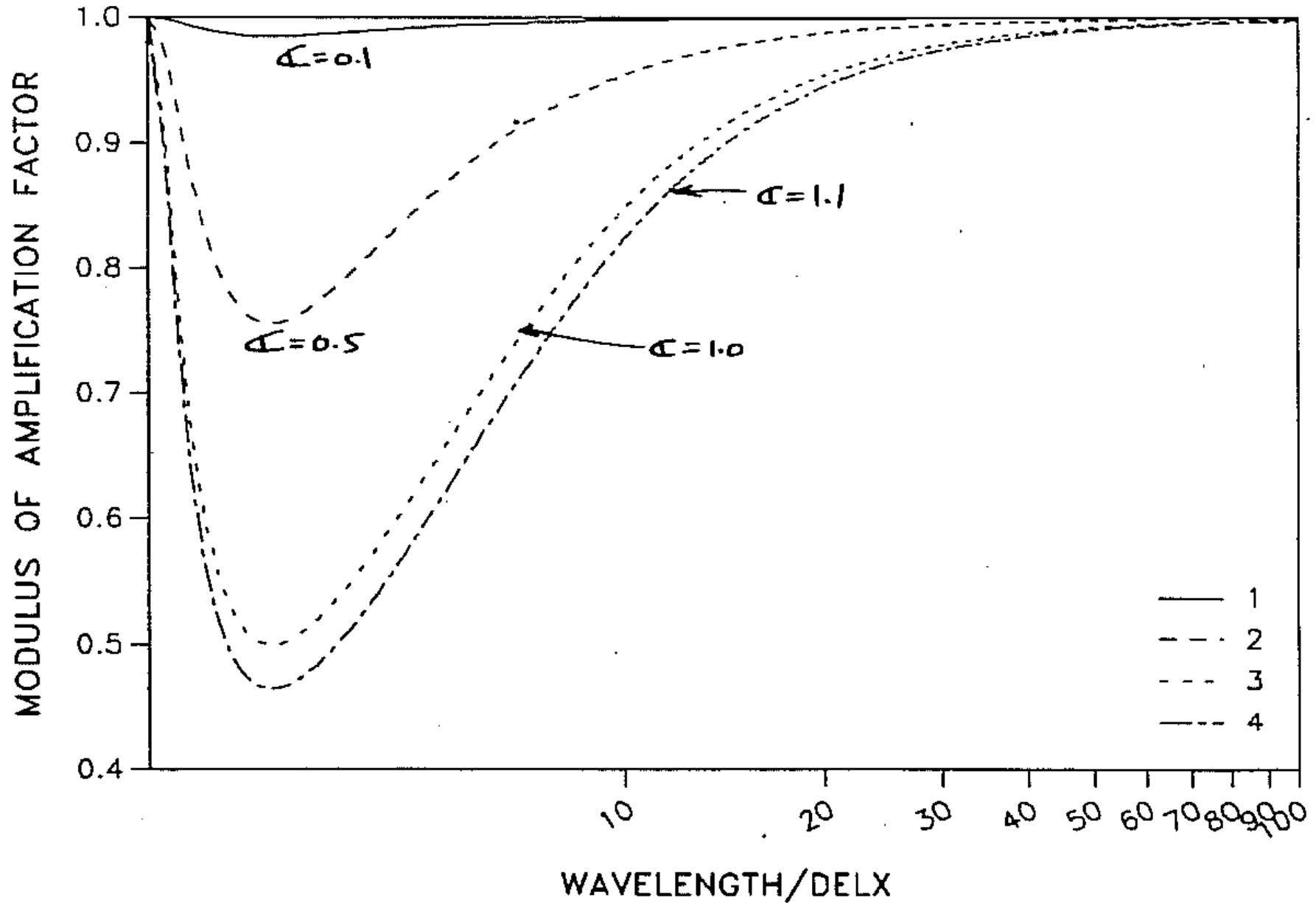


Fig. L29.9b1

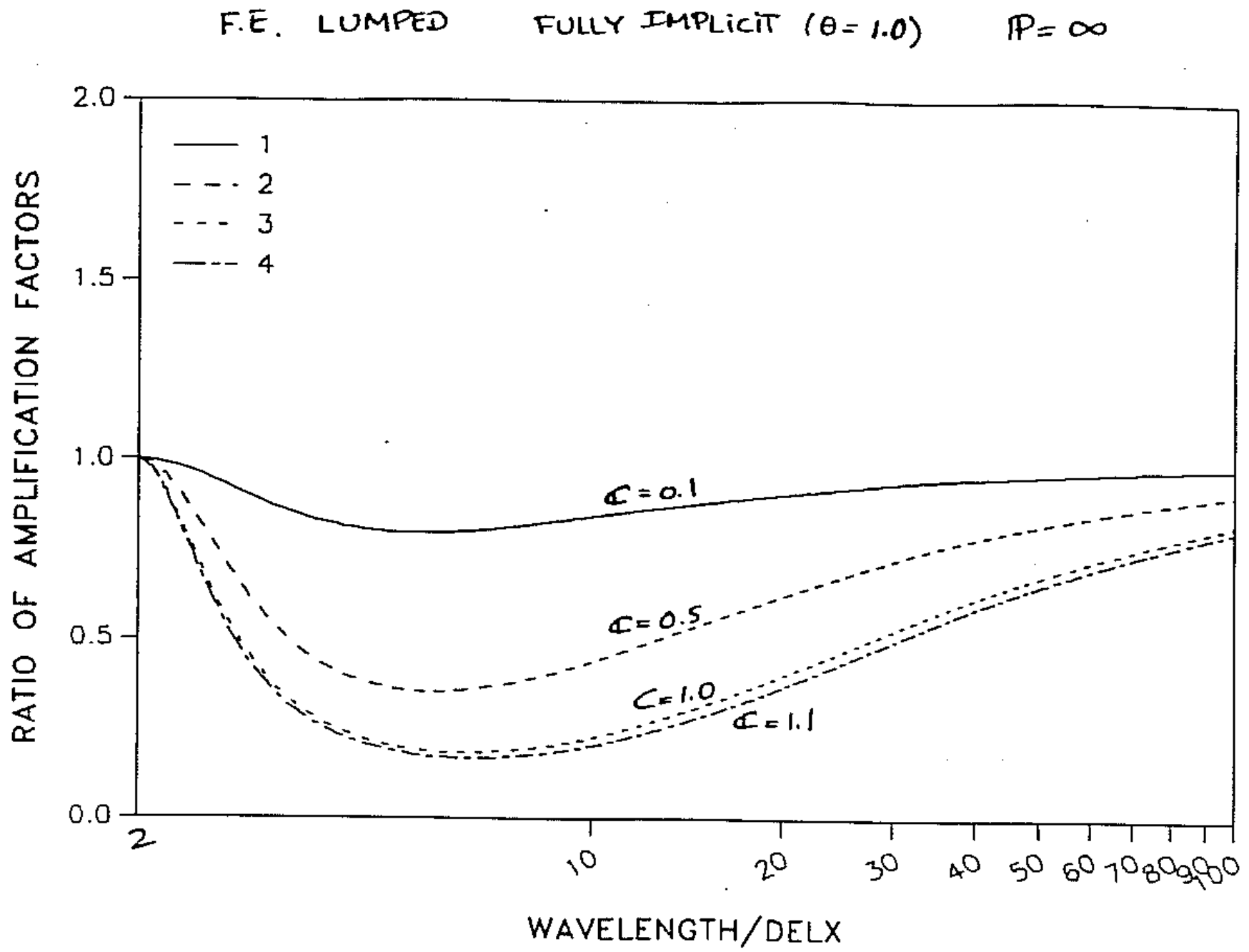


Fig. L29.9b2

F.E. CONSISTENT FULLY IMPLICIT ($\theta = 1.0$) $IP = \infty$

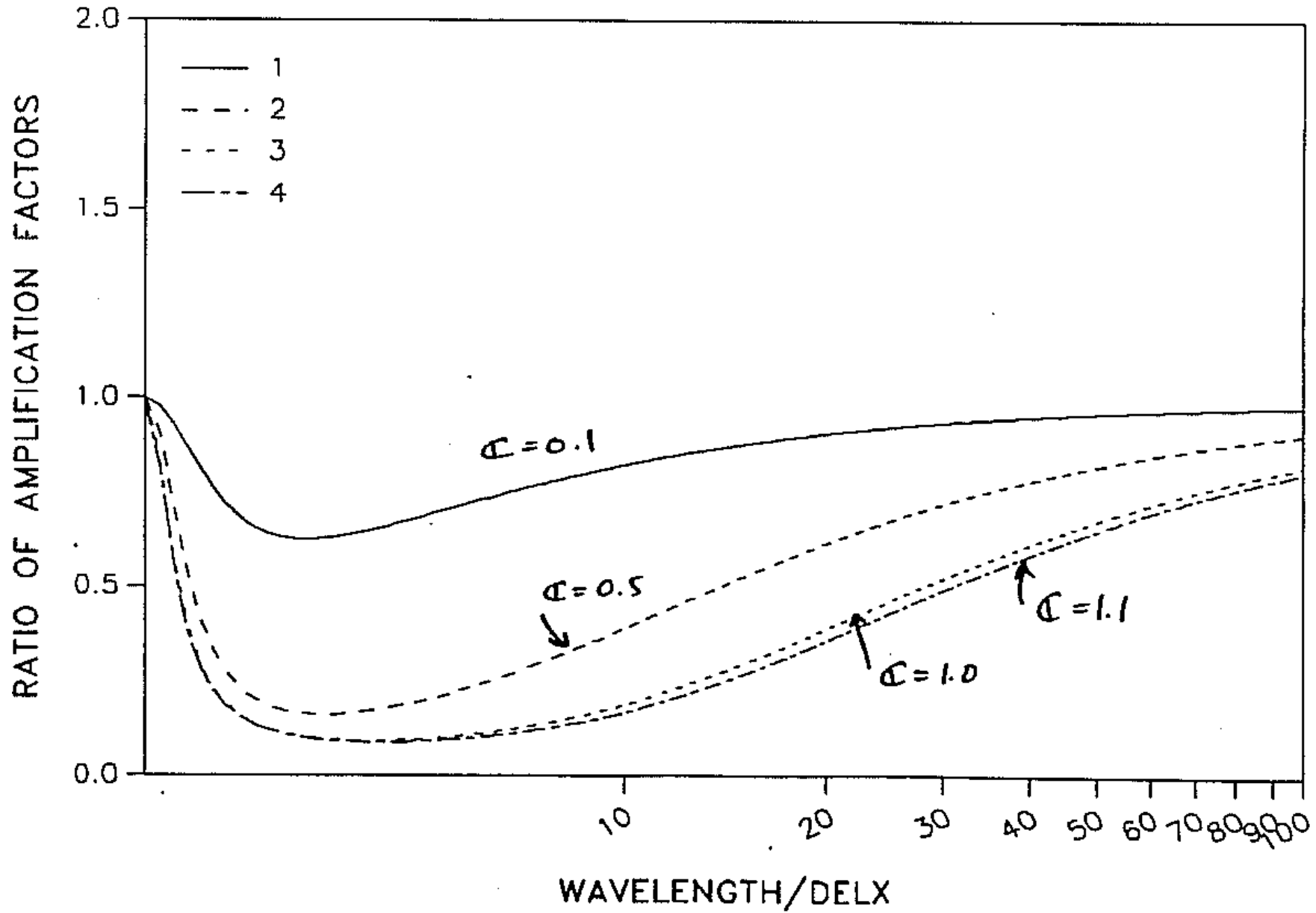


Fig. L29.9c1

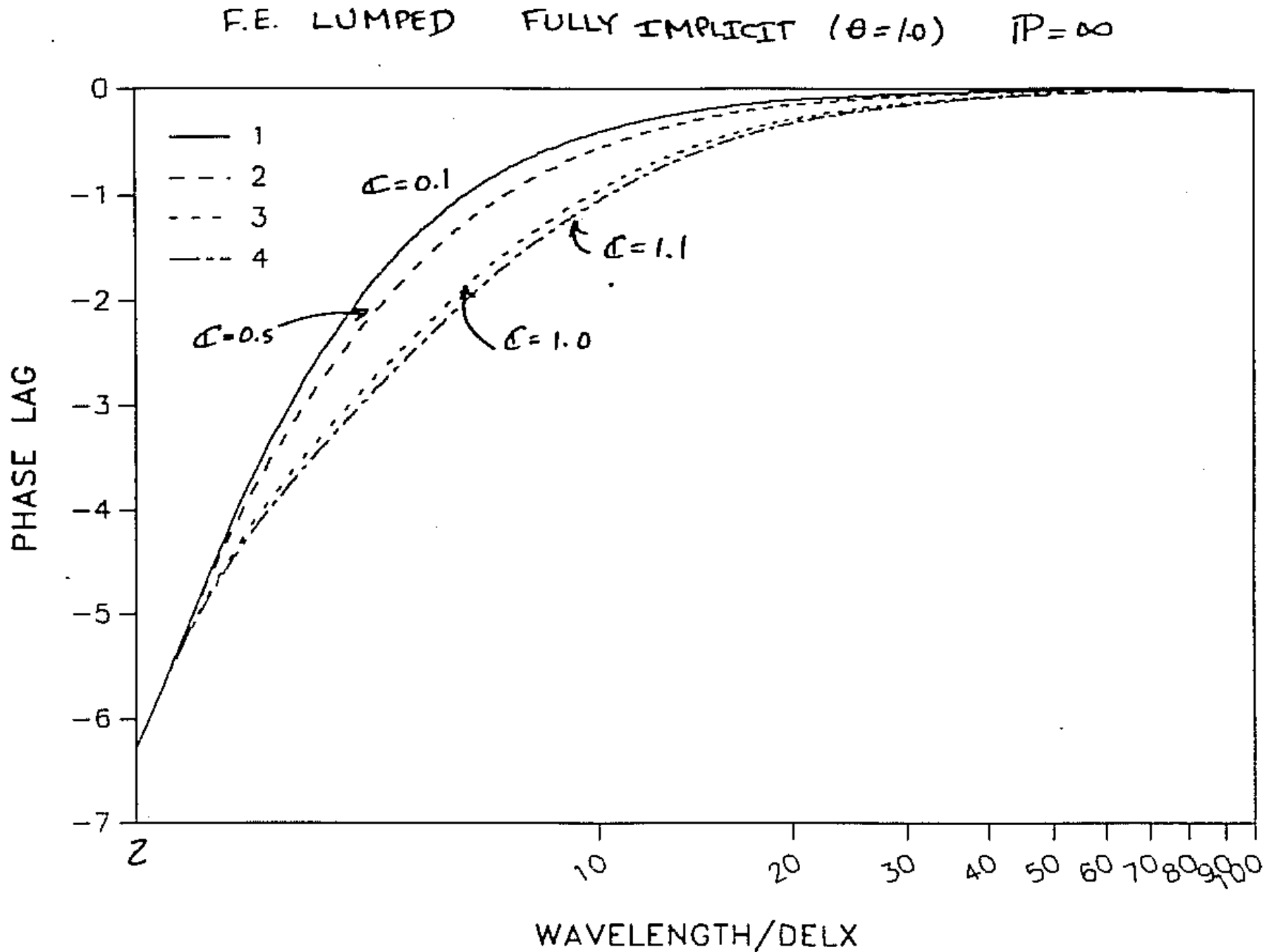


Fig. L29.9c2

F.E. CONSISTENT FULLY IMPLICIT ($\theta = 1.0$) $IP = \infty$

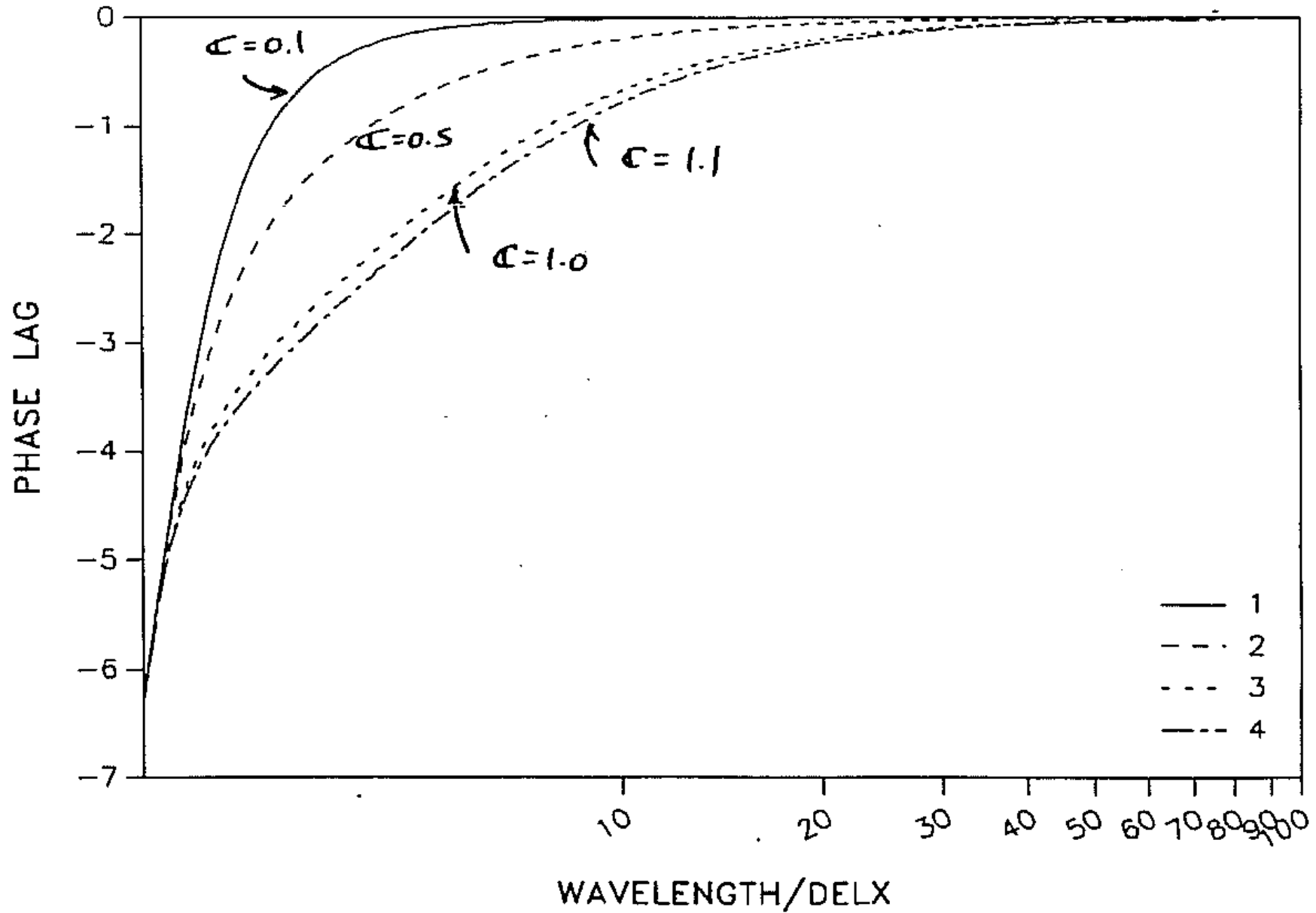


Fig. L29.10a1

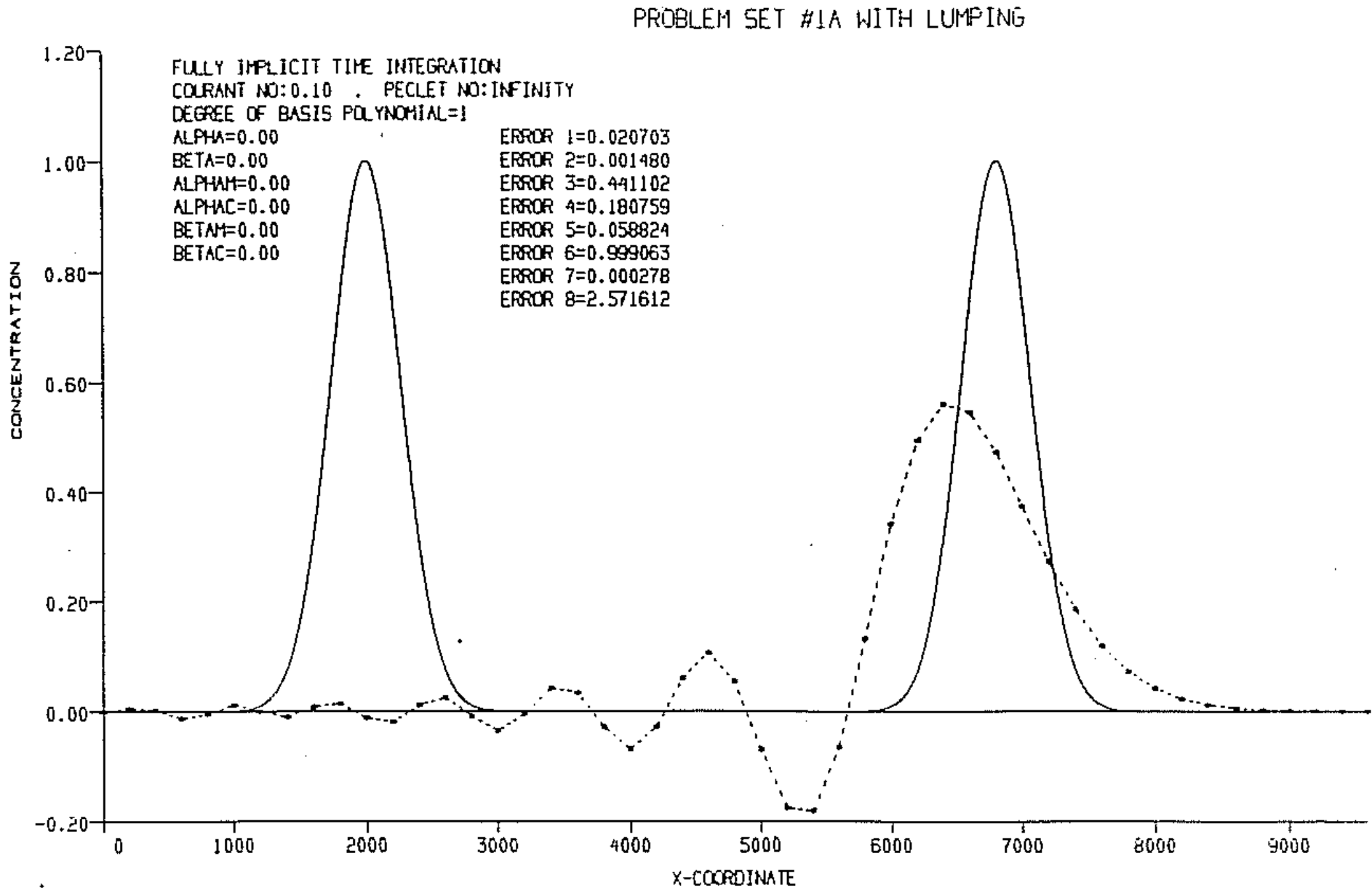


Fig. L29.10a2

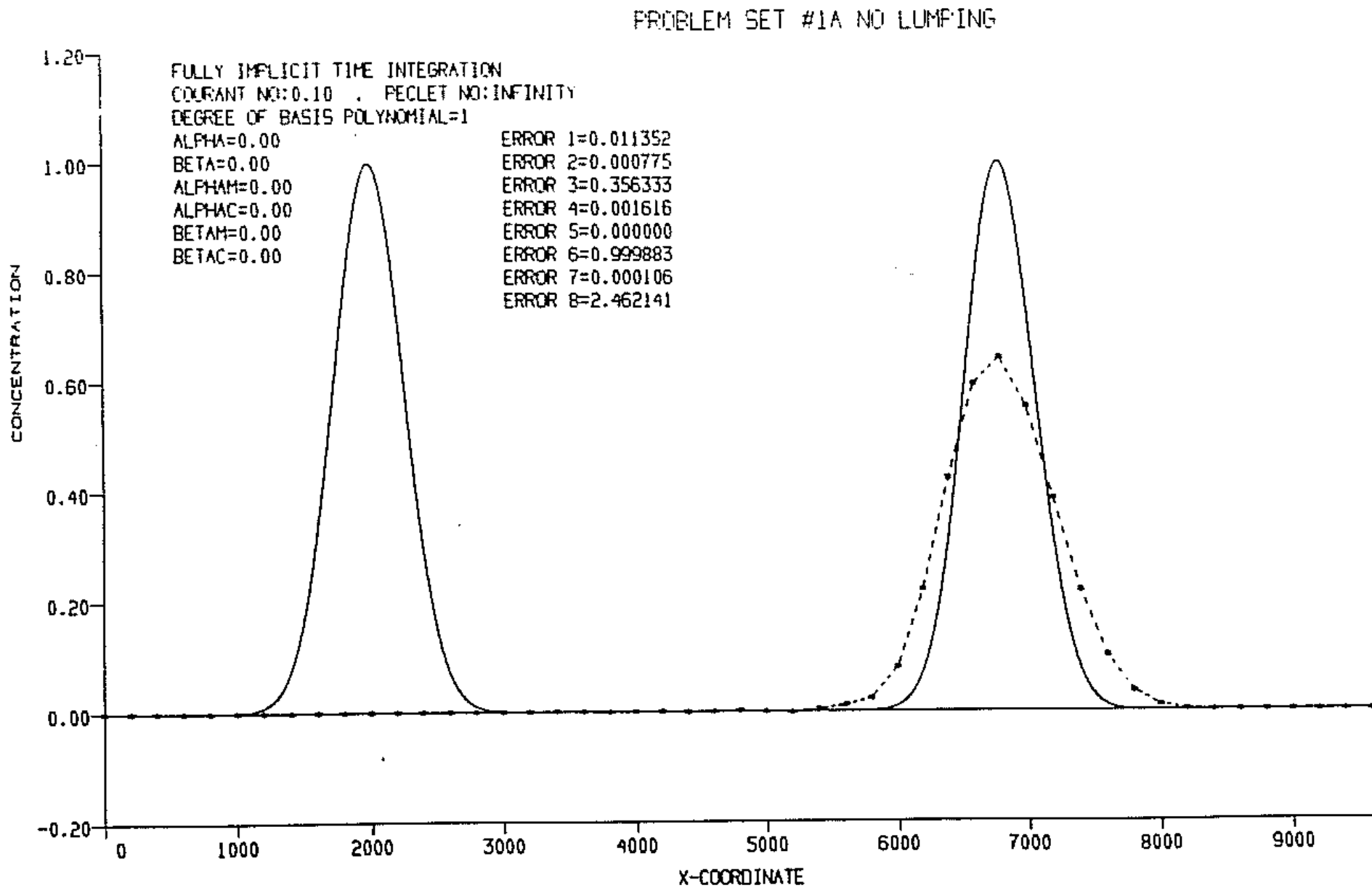


Fig. L29.10b1

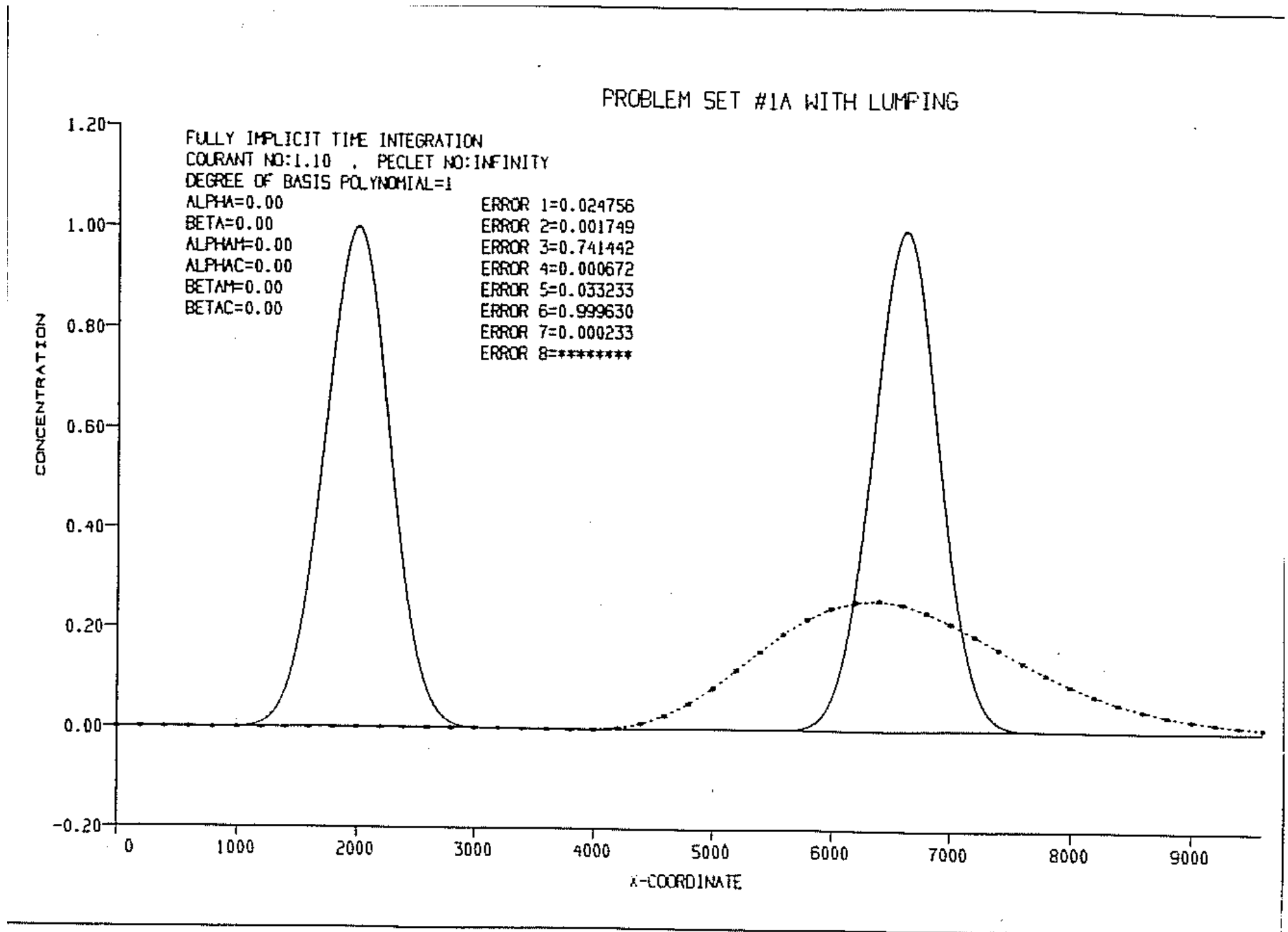


Fig. L29.10b2

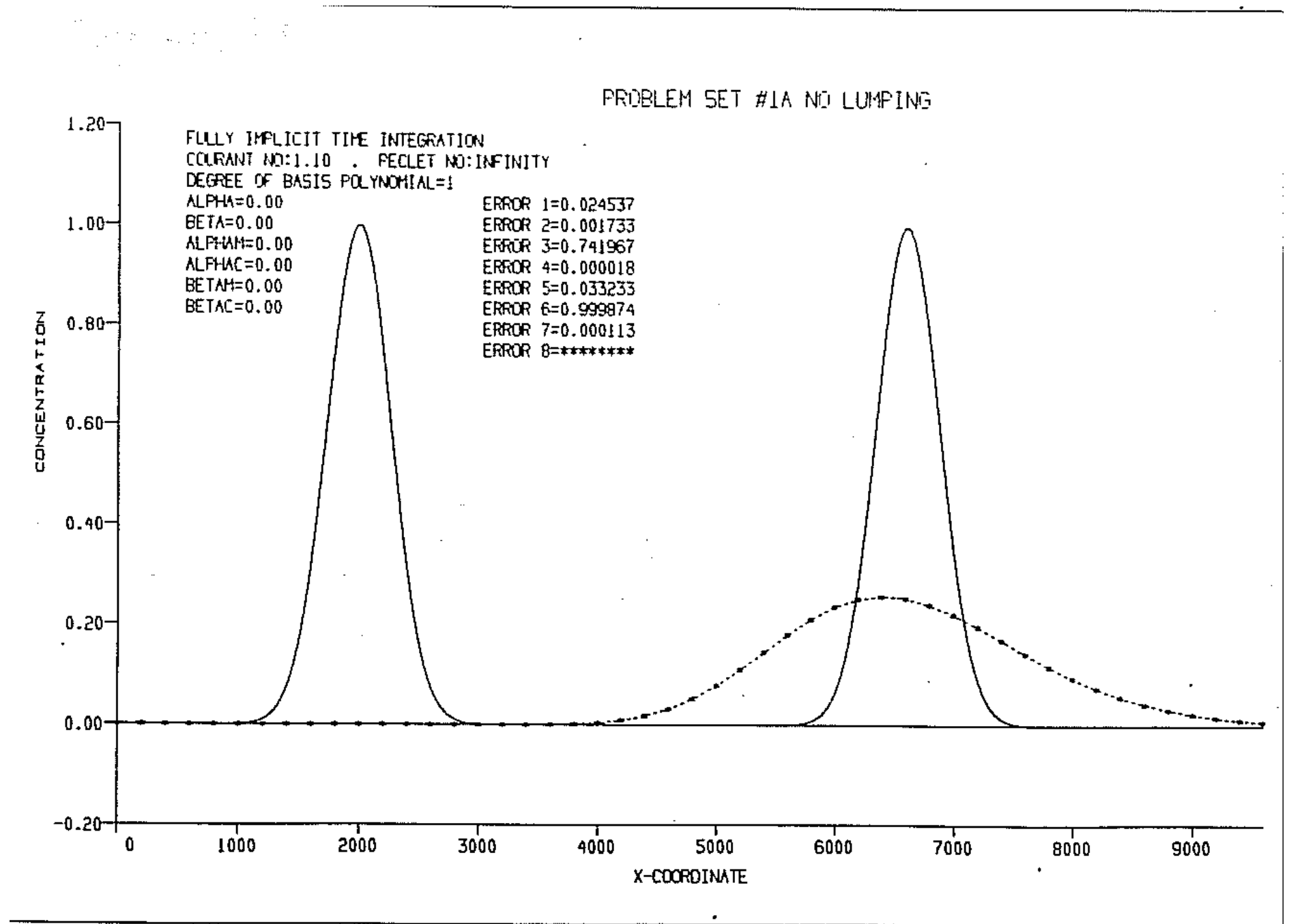


Fig. L29.11a1

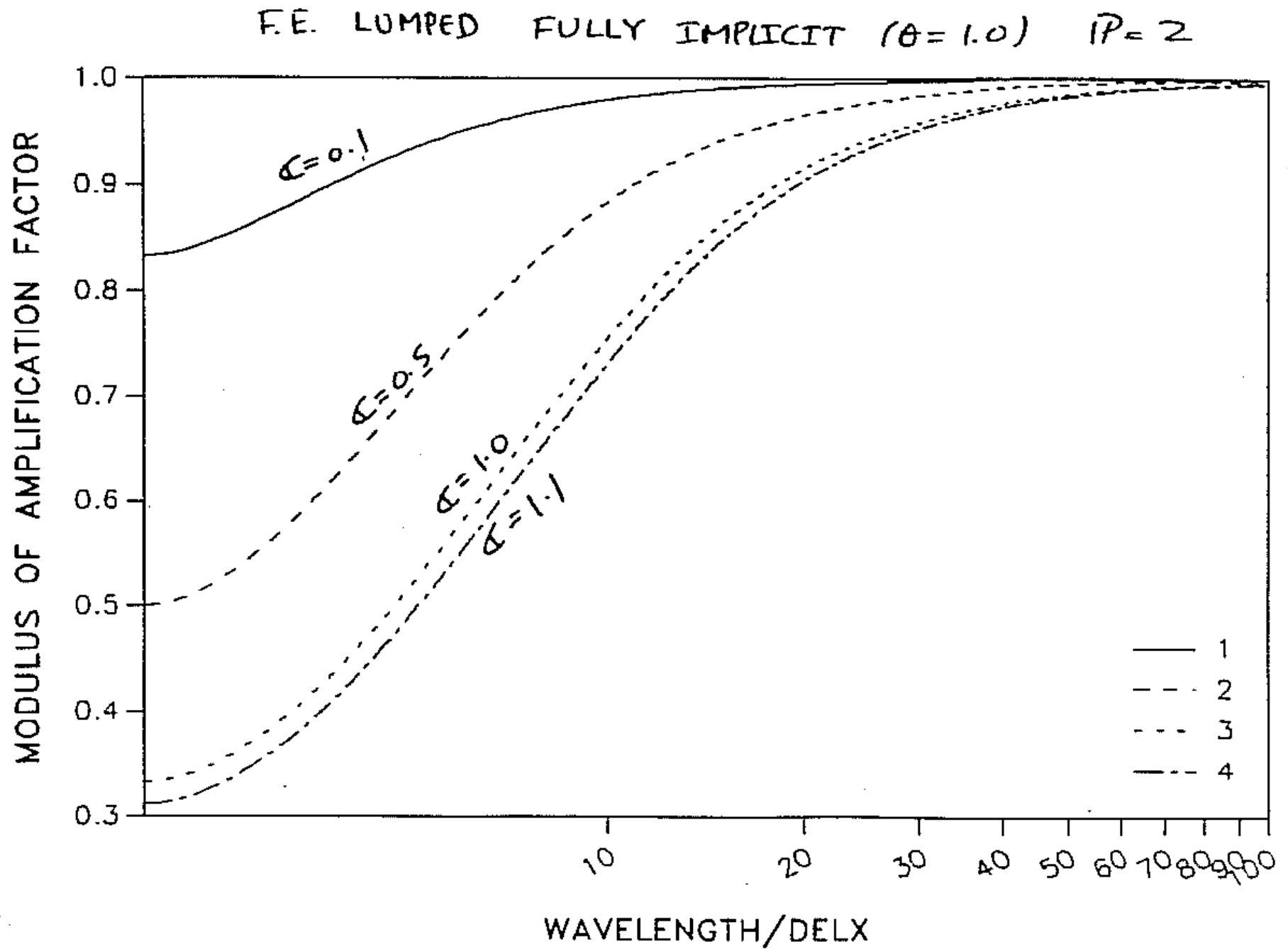


Fig. L29.11a2

F.E. CONSISTENT FULLY IMPLICIT ($\theta=1.0$) $P=2$

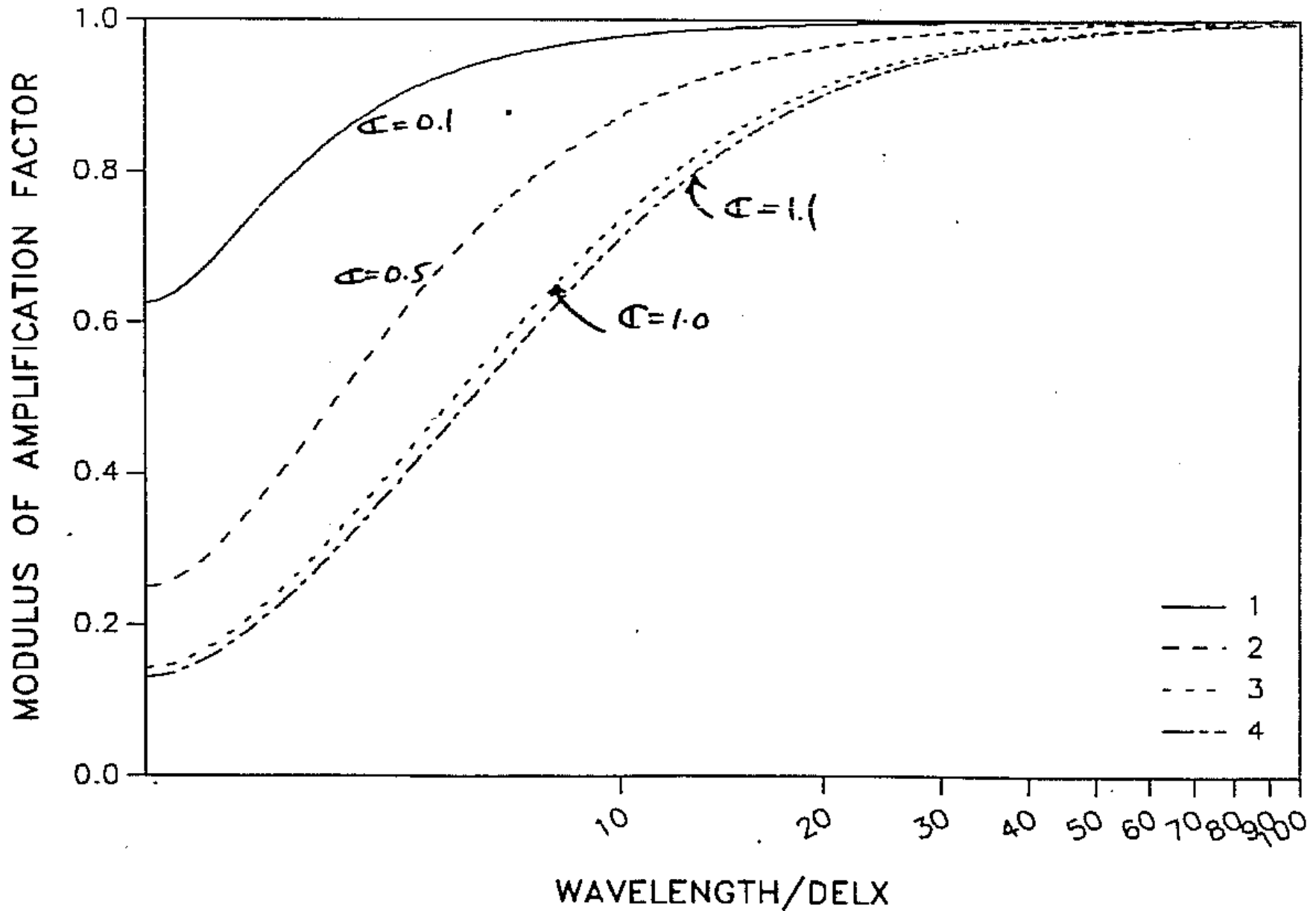


Fig. L29.11b1

F.E. LUMPED FULLY IMPLICIT ($\theta=1.0$) $\cdot IP=2$

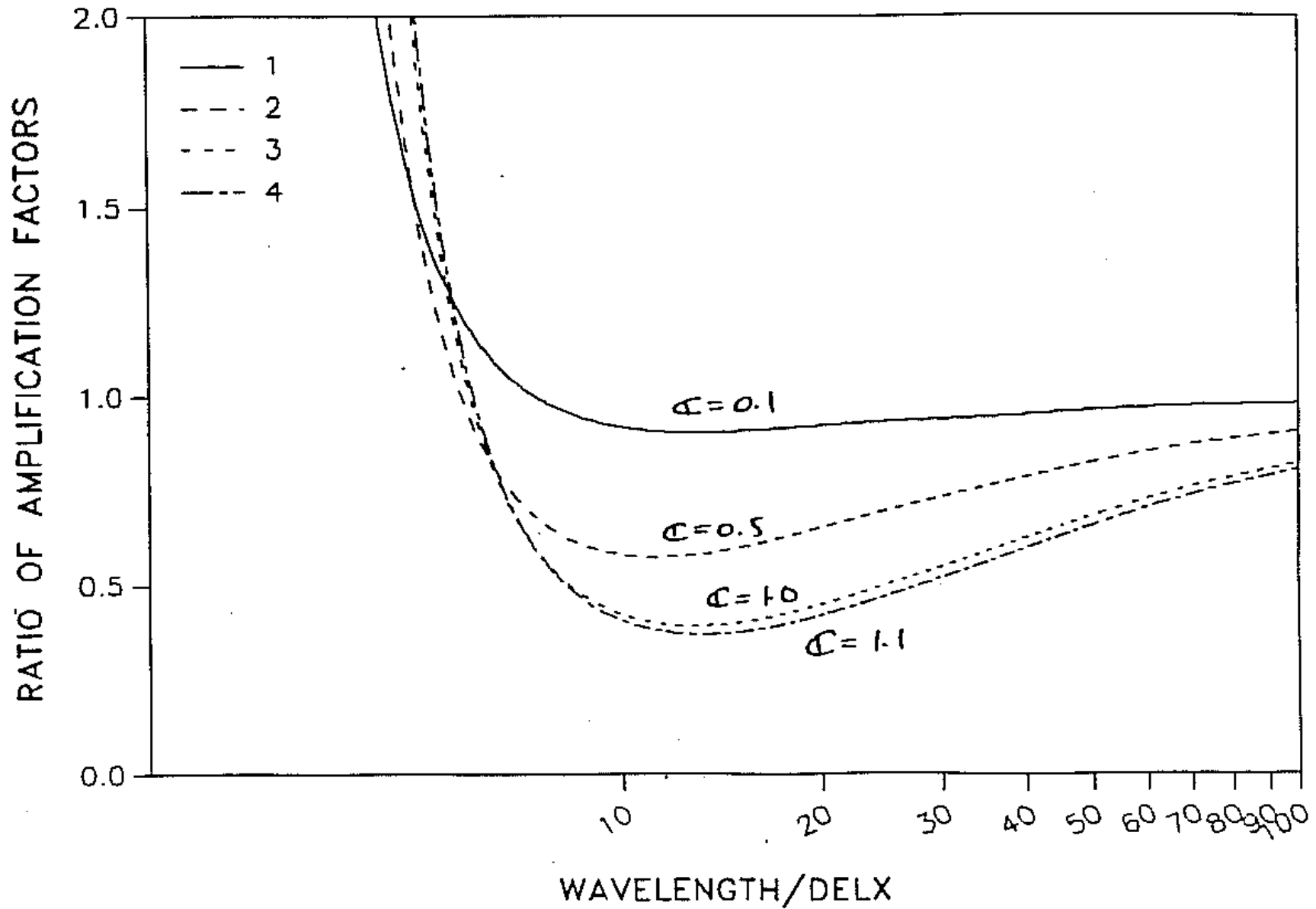


Fig. L29.11b2

F.E. CONSISTENT FULLY IMPLICIT ($\theta = 1.0$)

$\nu P = 2$

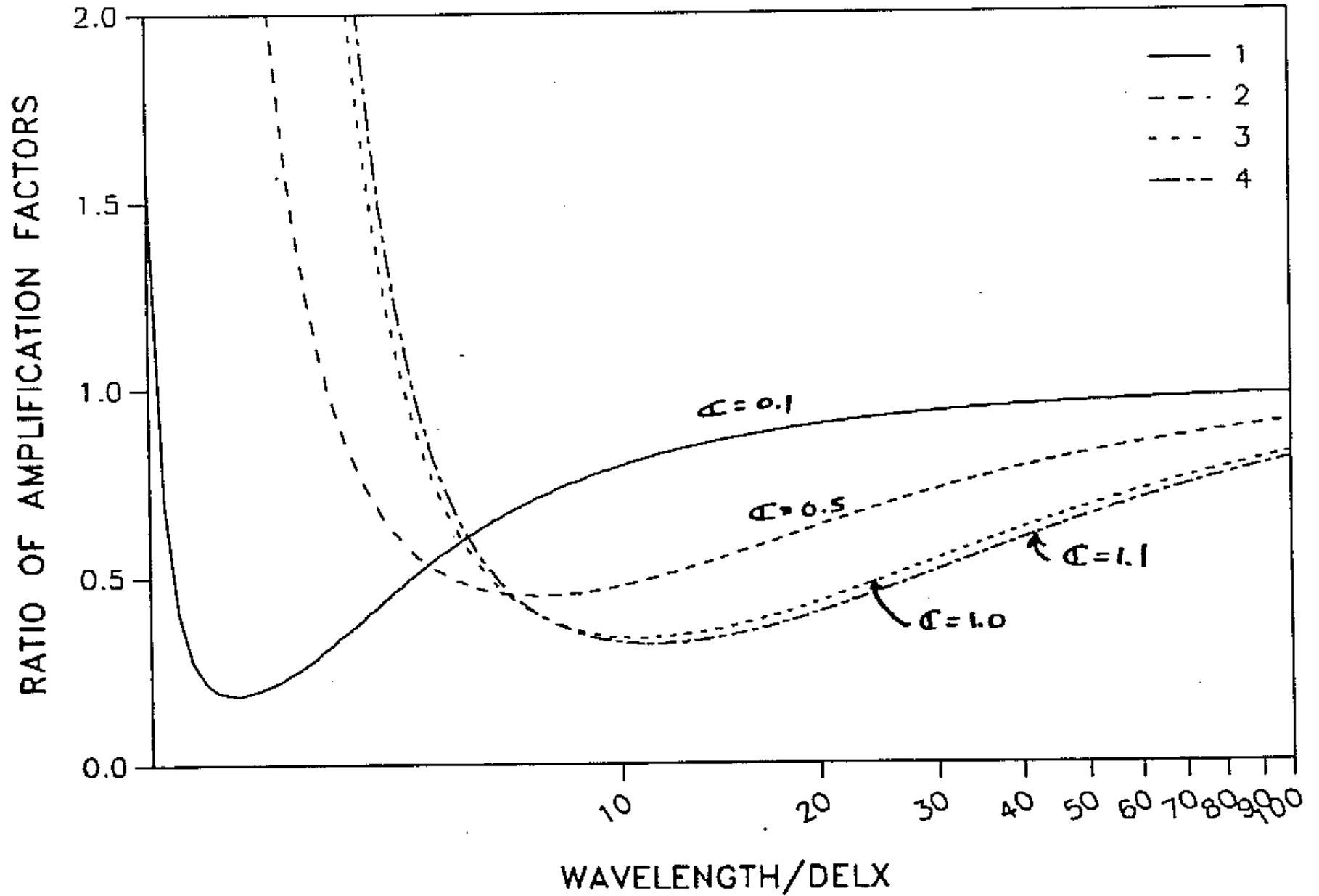


Fig. L29.11c1

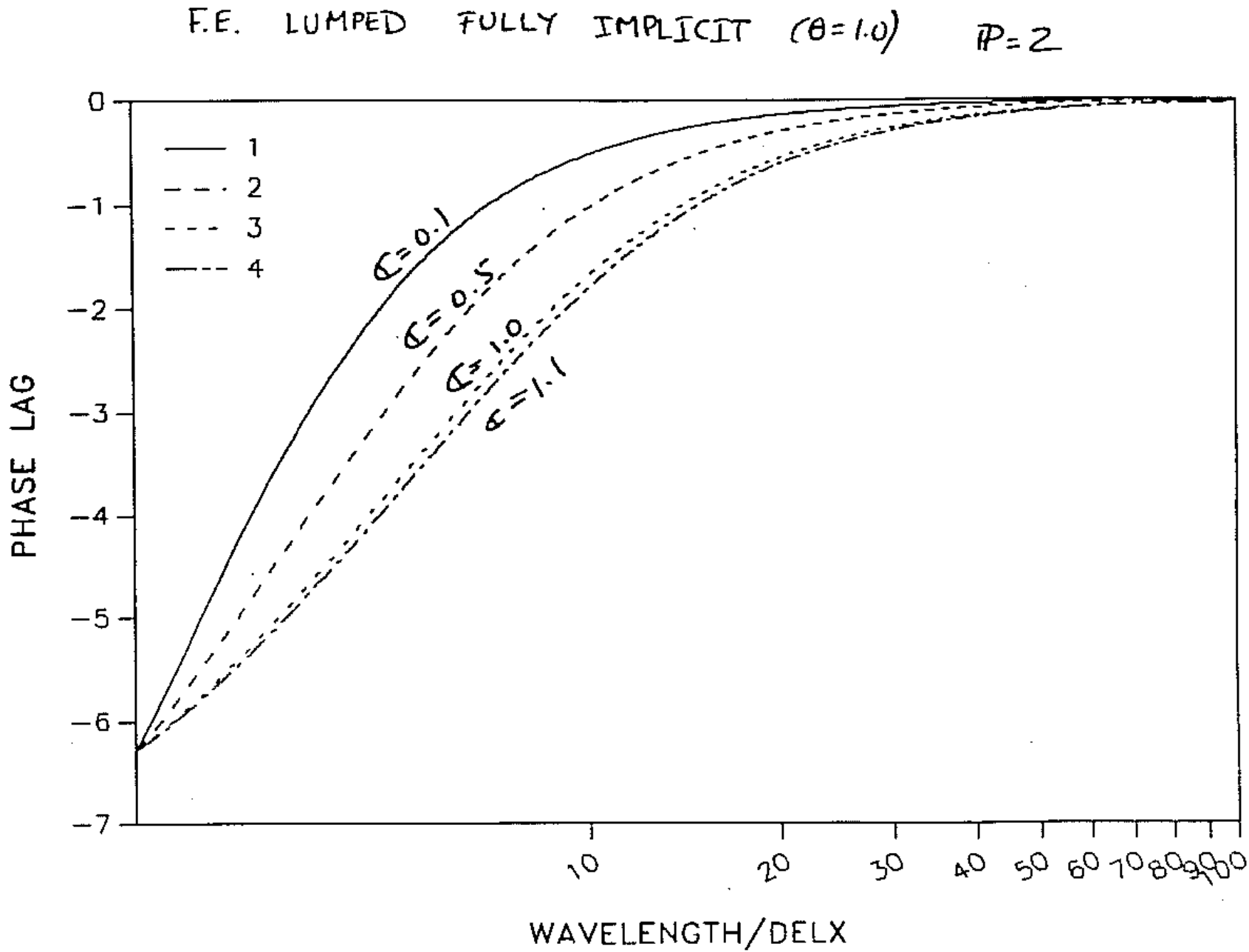


Fig. L29.11c2

F.E. CONSISTENT FULLY IMPLICIT ($\theta = 1.0$)

$P = 2$

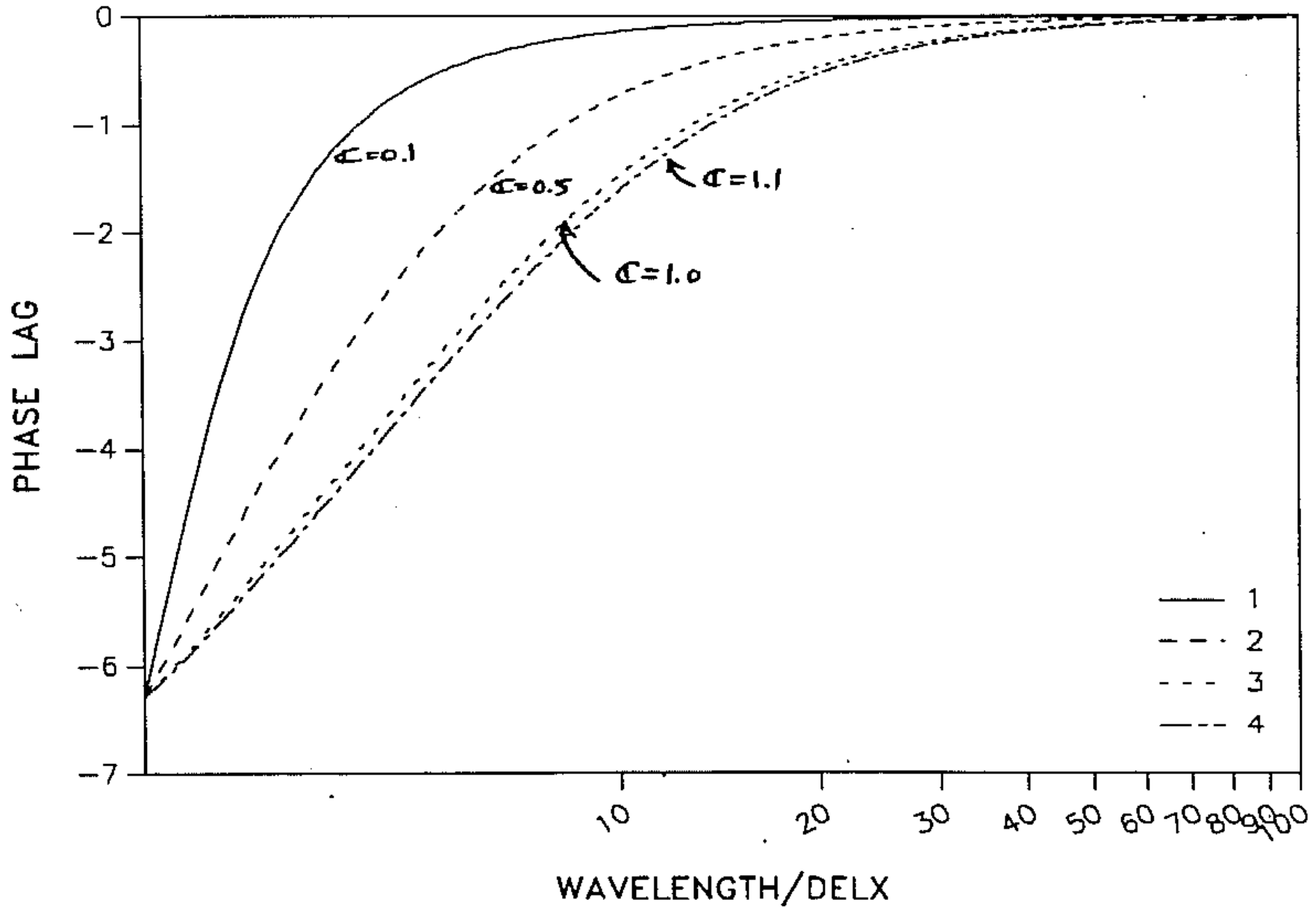


Fig. L29.12a1

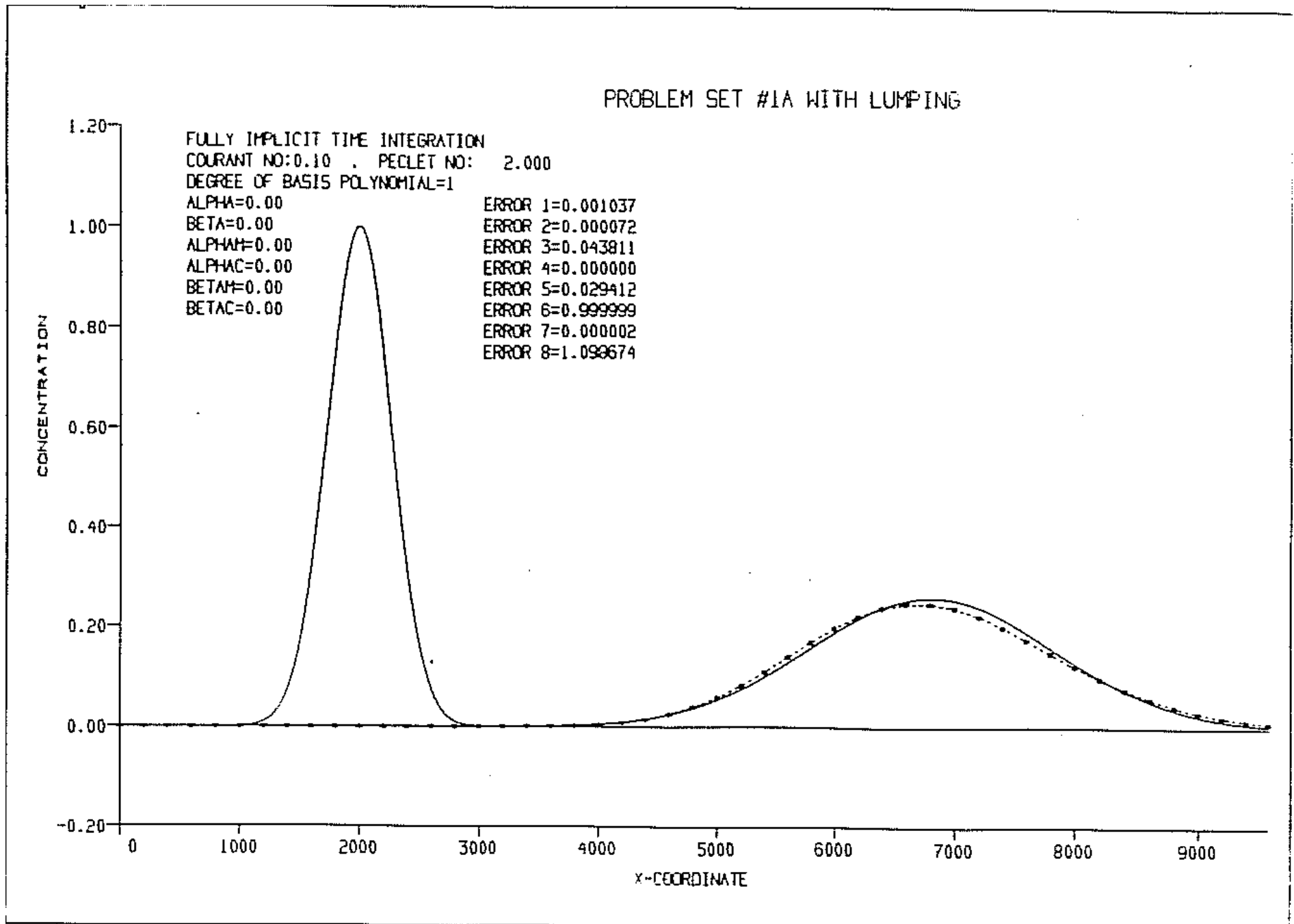


Fig. L29.12a2

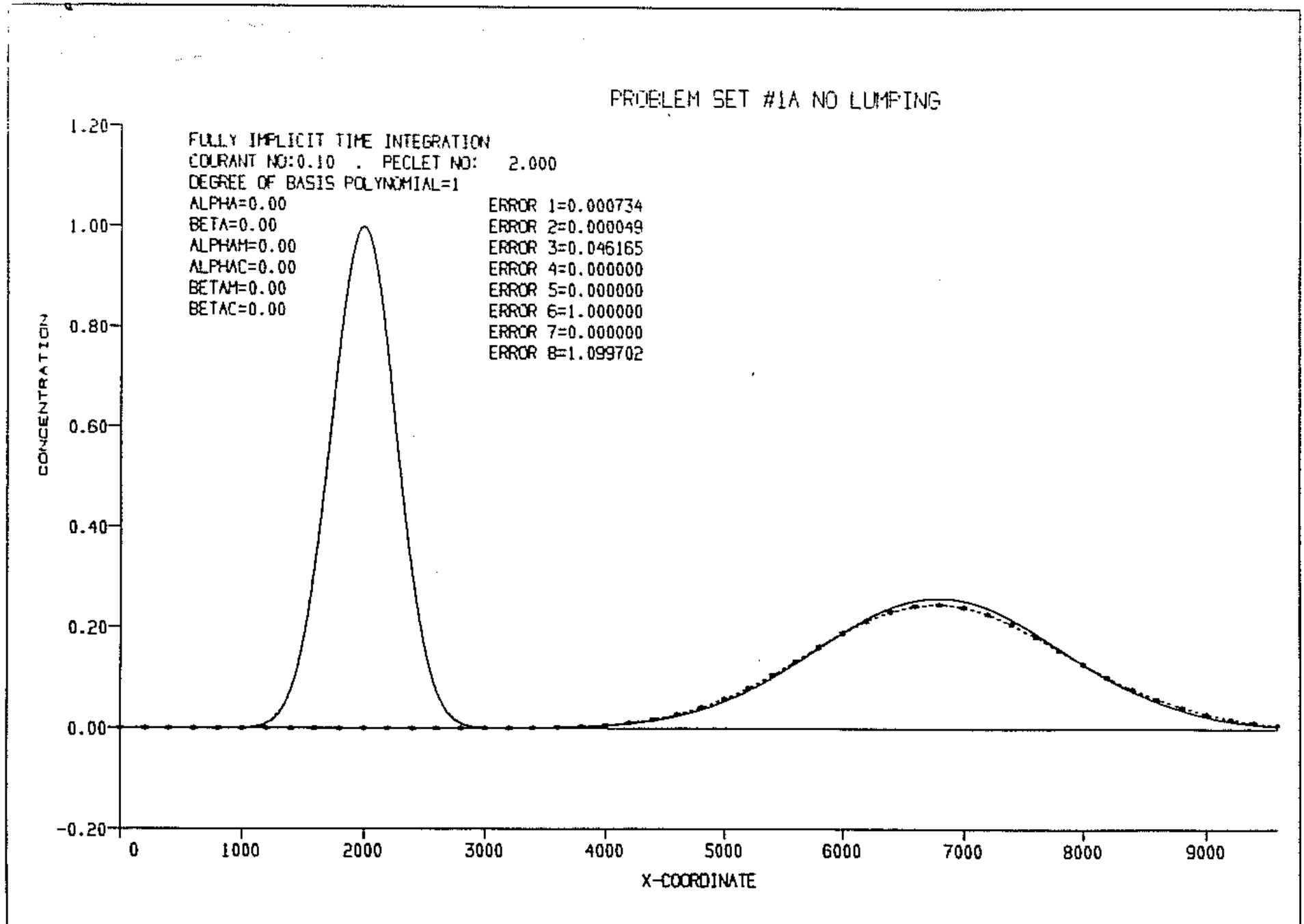


Fig. L29.12b1

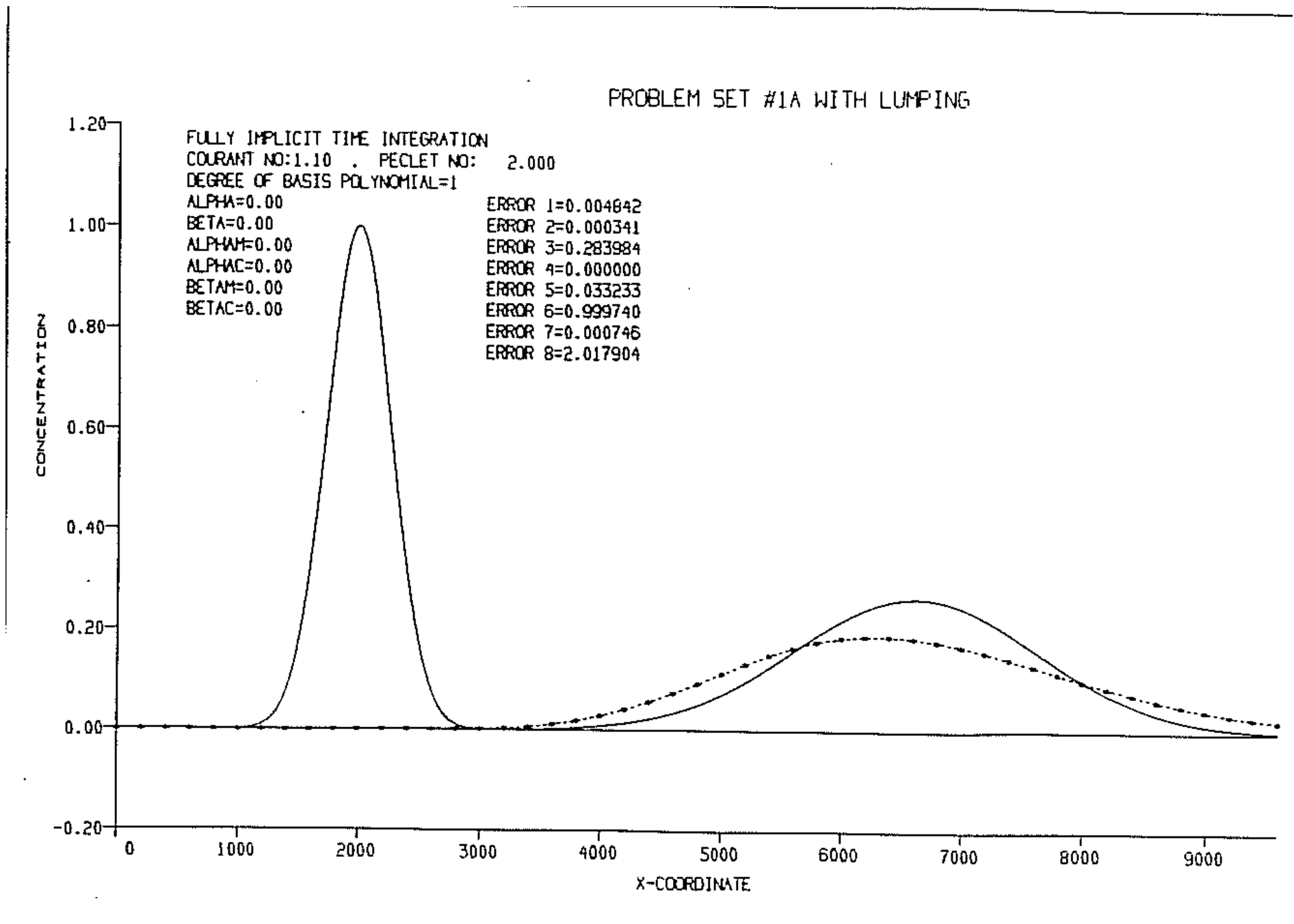


Fig. L29.12b2

